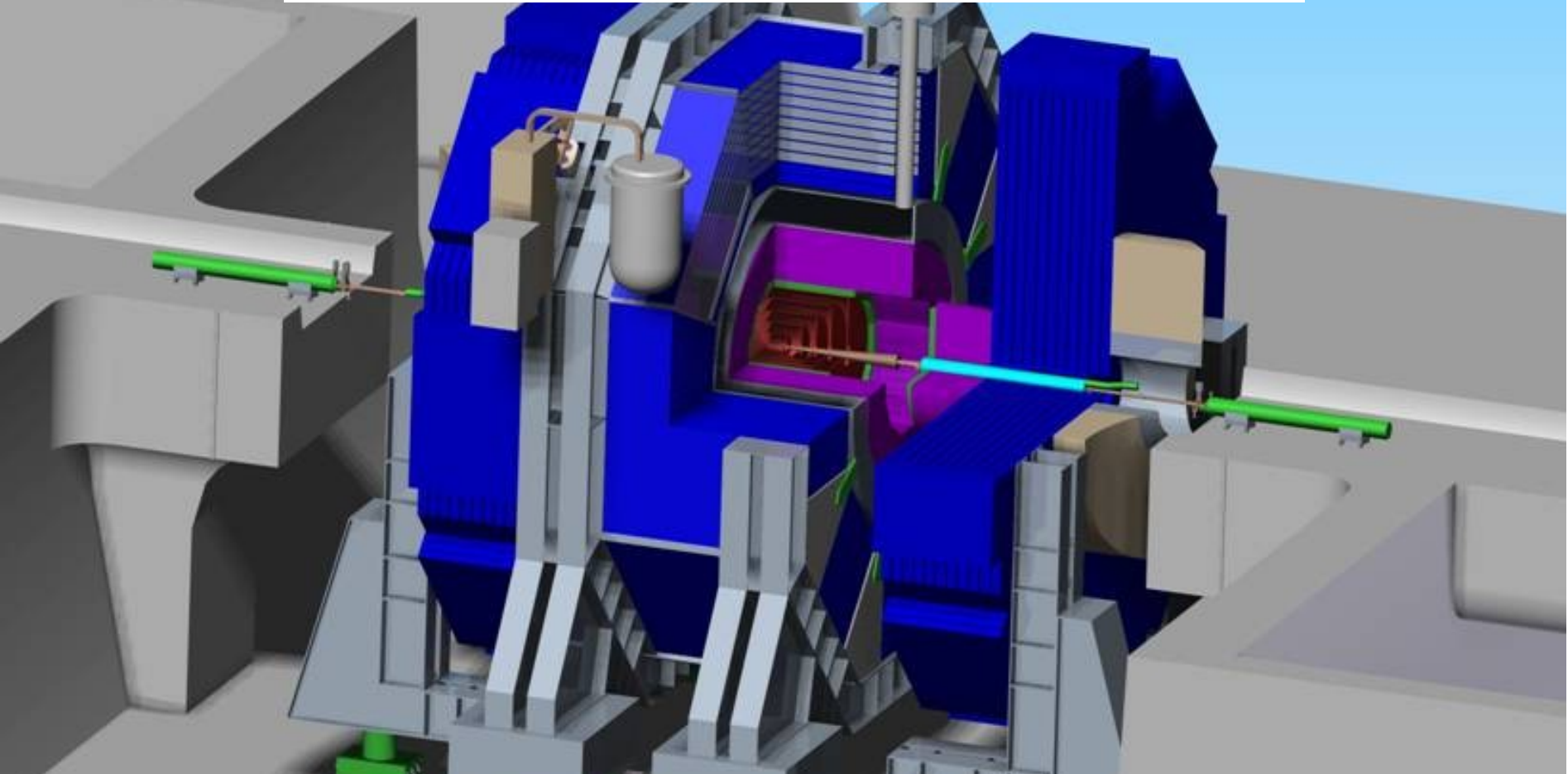


# ALCPG11 Eugene- Oregon, March 2011



SiD MDI

Marco Oriunno, SLAC



# Outline of the talk

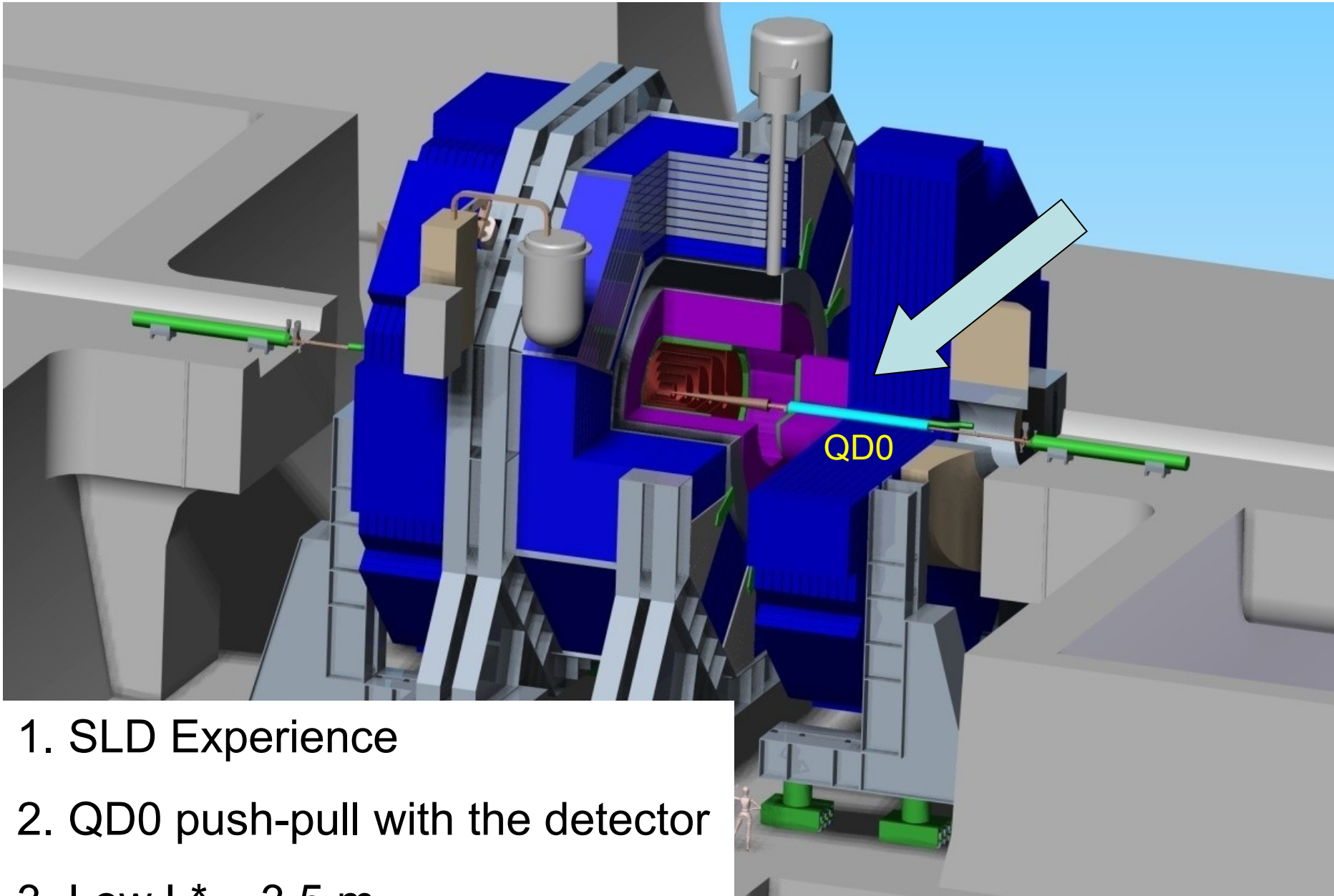
1. Push Pull

2. Vibrations

Other SiD - MDI talks in this conference :

Sunday,	Vibration Measurements and Transfer Functions at SLAC,	K. Bertsche
Monday,	Feedback Analysis with current xfer functions & beam parameters	G.White
Monday,	SR Update,	M. Sullivan
Monday,	HOM heating at the IP and in QD0,	A. Novohatski
Monday,	FSI Alignment,	K. Riles

# QD0 supported from the doors

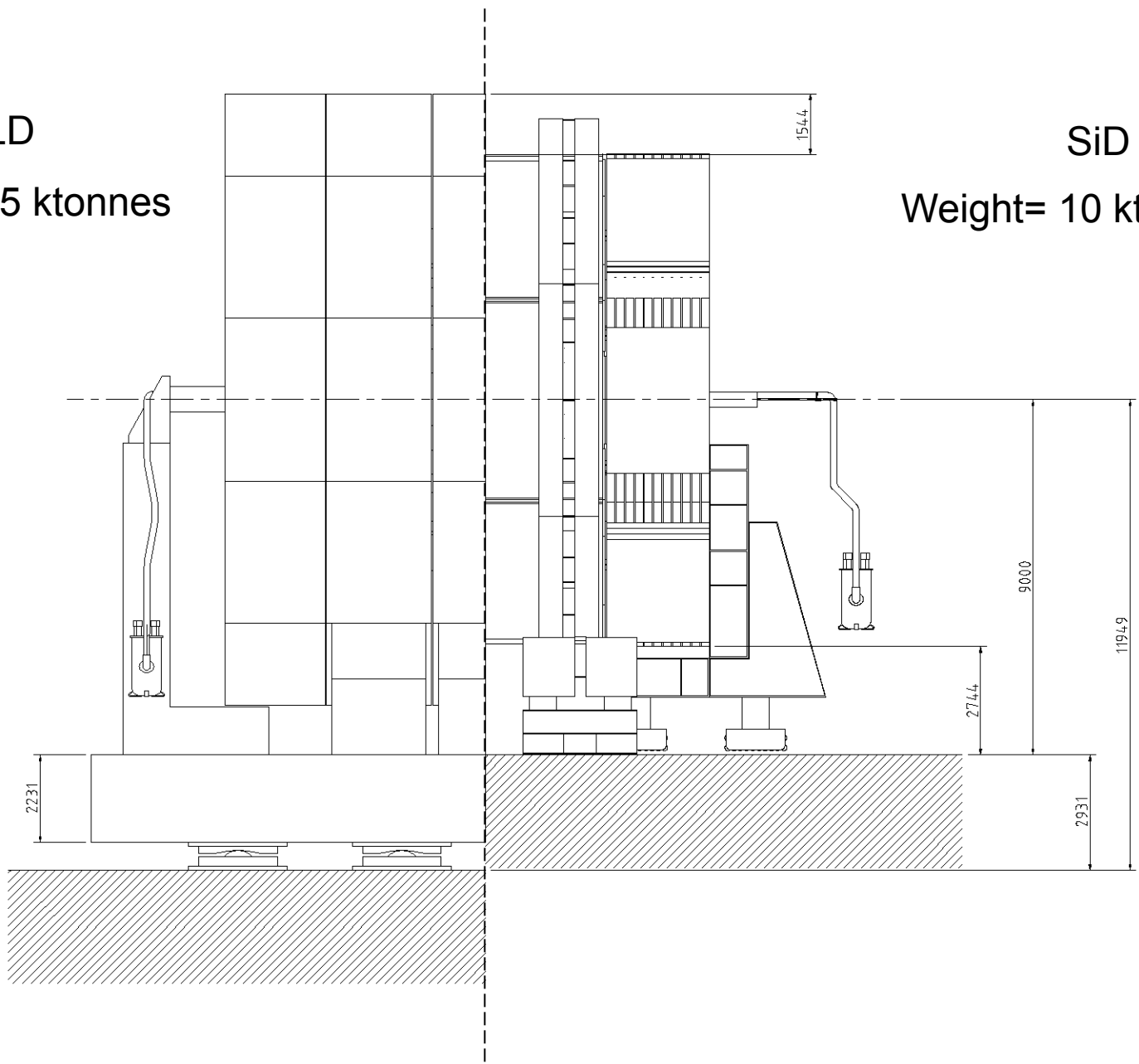


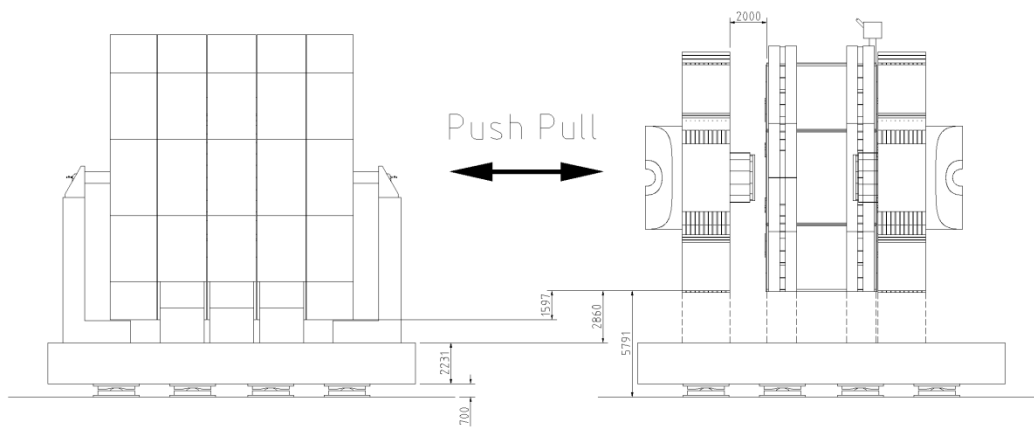
1. SLD Experience
2. QD0 push-pull with the detector
3. Low  $L^* \sim 3.5$  m

# ILD and SiD differences

ILD  
Weight= 15 ktonnes

SiD  
Weight= 10 ktonnes

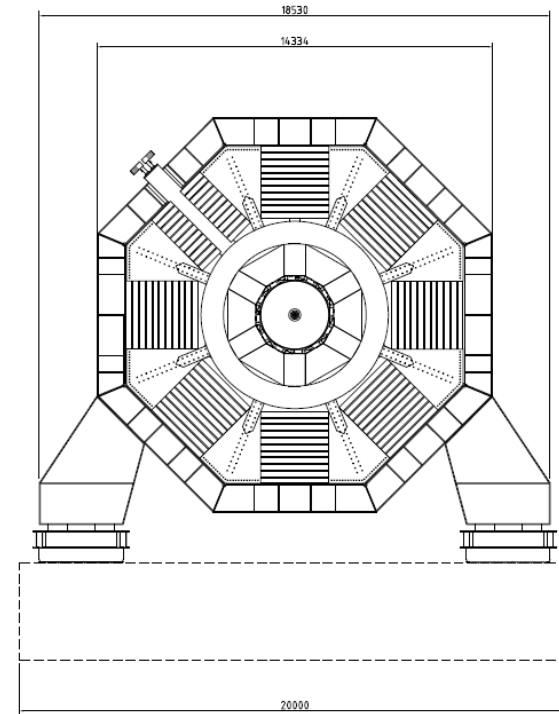
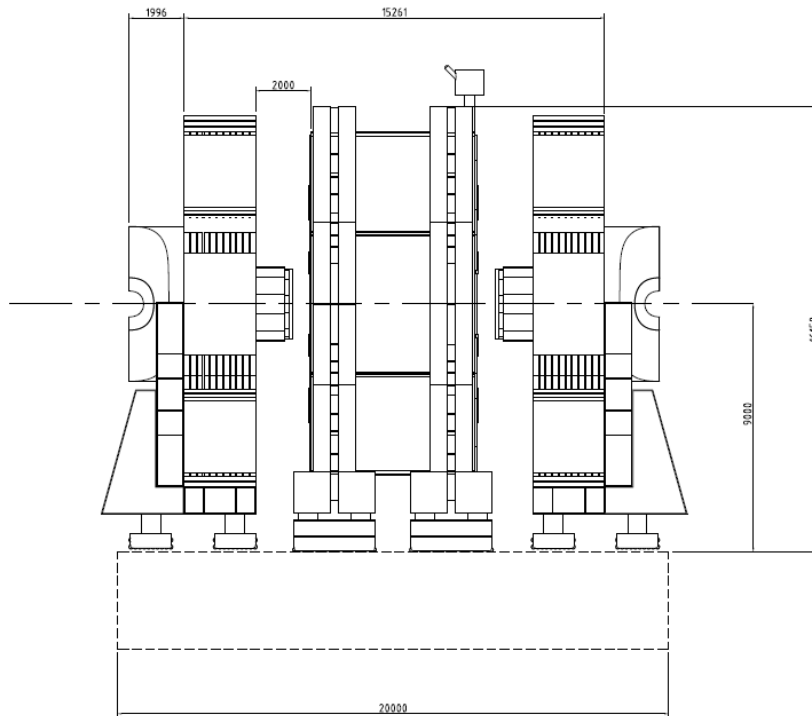




Option 3, ILD and SiD on platforms

Under Study

# SiD Platform Functional Requirements



SiD nominal mass: Barrel 5000 T; (each) Door 2500 T

Dimensions:

Z = 20.0 m

X = 20.0 m

Delta Y = 9 m (Top of Platform to beamline)

Positioning Tolerance on beamline

Consider points Z=+-max, X=0. Position to + 1mm wrt references in X,Y,Z

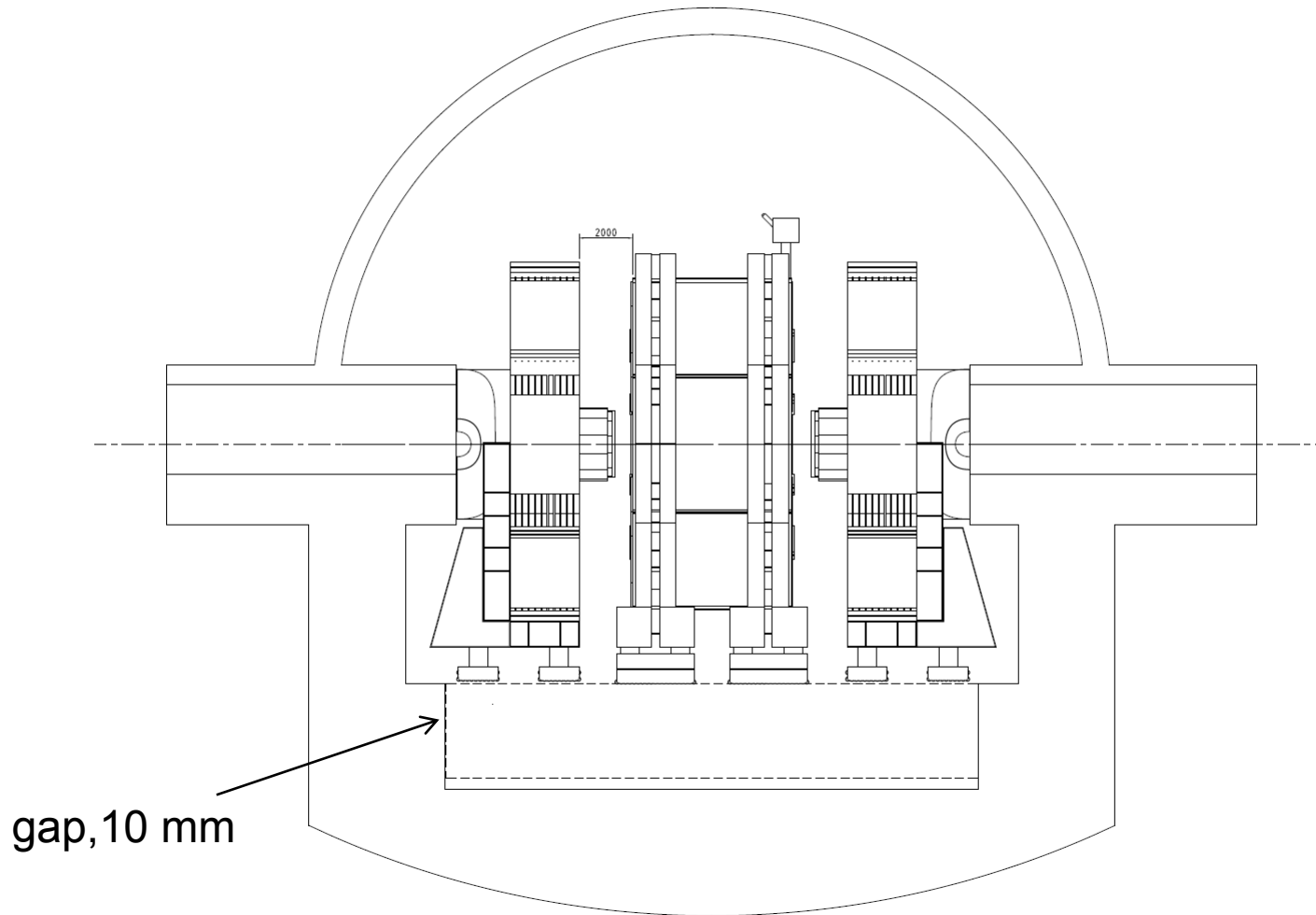
Consider points Z=+-max, X=+-max: Position to +- 1 wrt references in Y.

Static Deformations: <+-2 mm

Vibration budget < 50nm between 1 and 100 Hz, at the QD0's (relative)

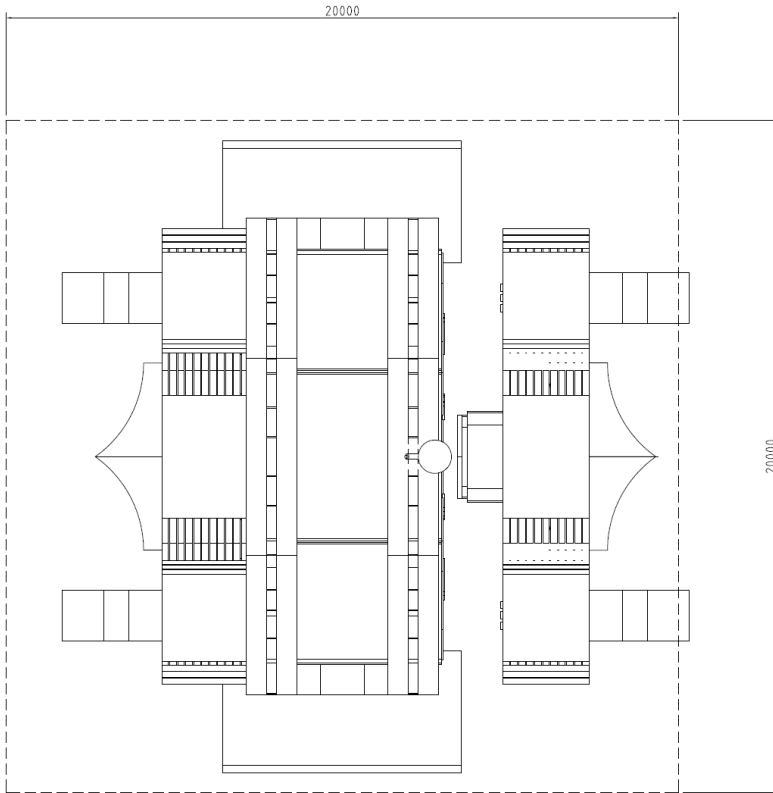
Seismic stability: Appropriate for selected site.  
(Beamline must be designed with sufficient compliance that VXD will survive)

# SiD Platform Functional Requirements

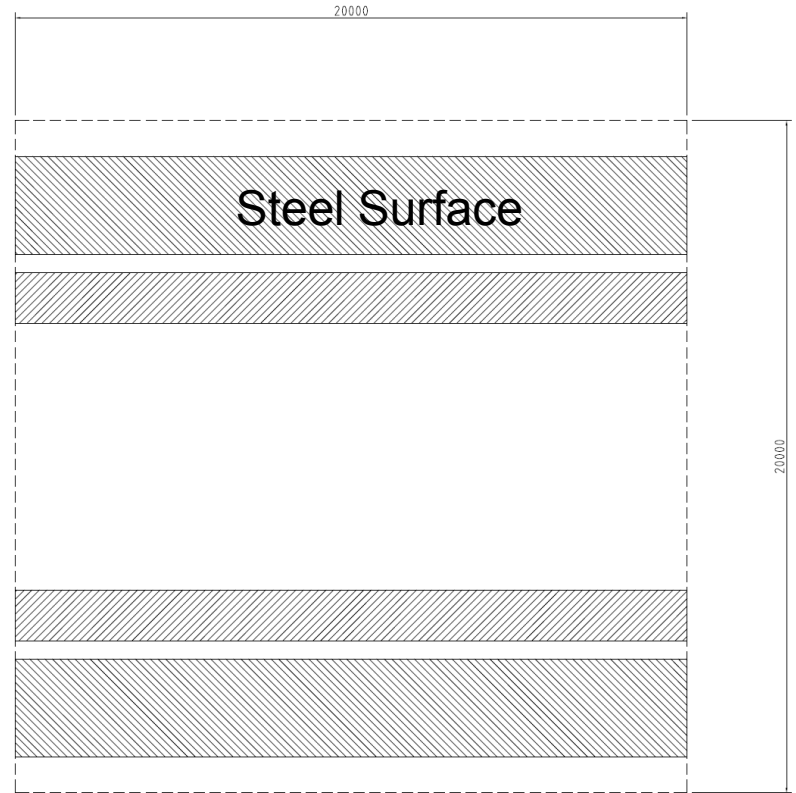


Wall clearance ~10 mm. Platform comes to side wall, there is no apron or apron matches platform elevation.

# SiD Platform Functional Requirements



Detector on platform Top View



Platform Top View

## Surface Features:

Steel Surface near legs

Steel rails for doors

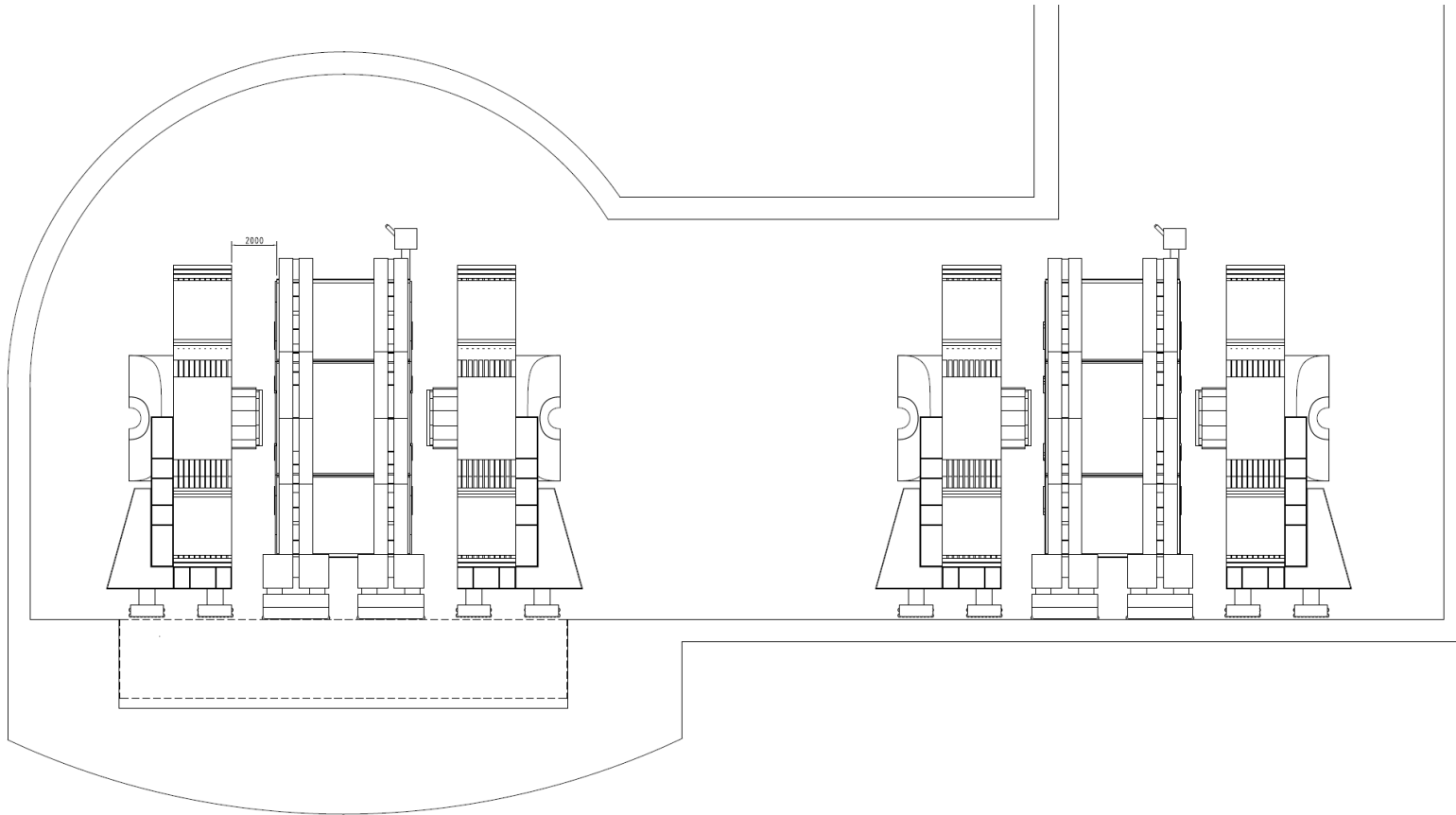
“Receptacles” for tie seismic tiedowns of SiD Barrel and

Doors

Removable Safety railings



# SiD Platform Functional Requirements



Accelerations:

$<1 \text{ mm/s}^2$

Transport velocity:

$V > 1 \text{ mm/s}$  after acceleration

Life: 100 motion cycles.

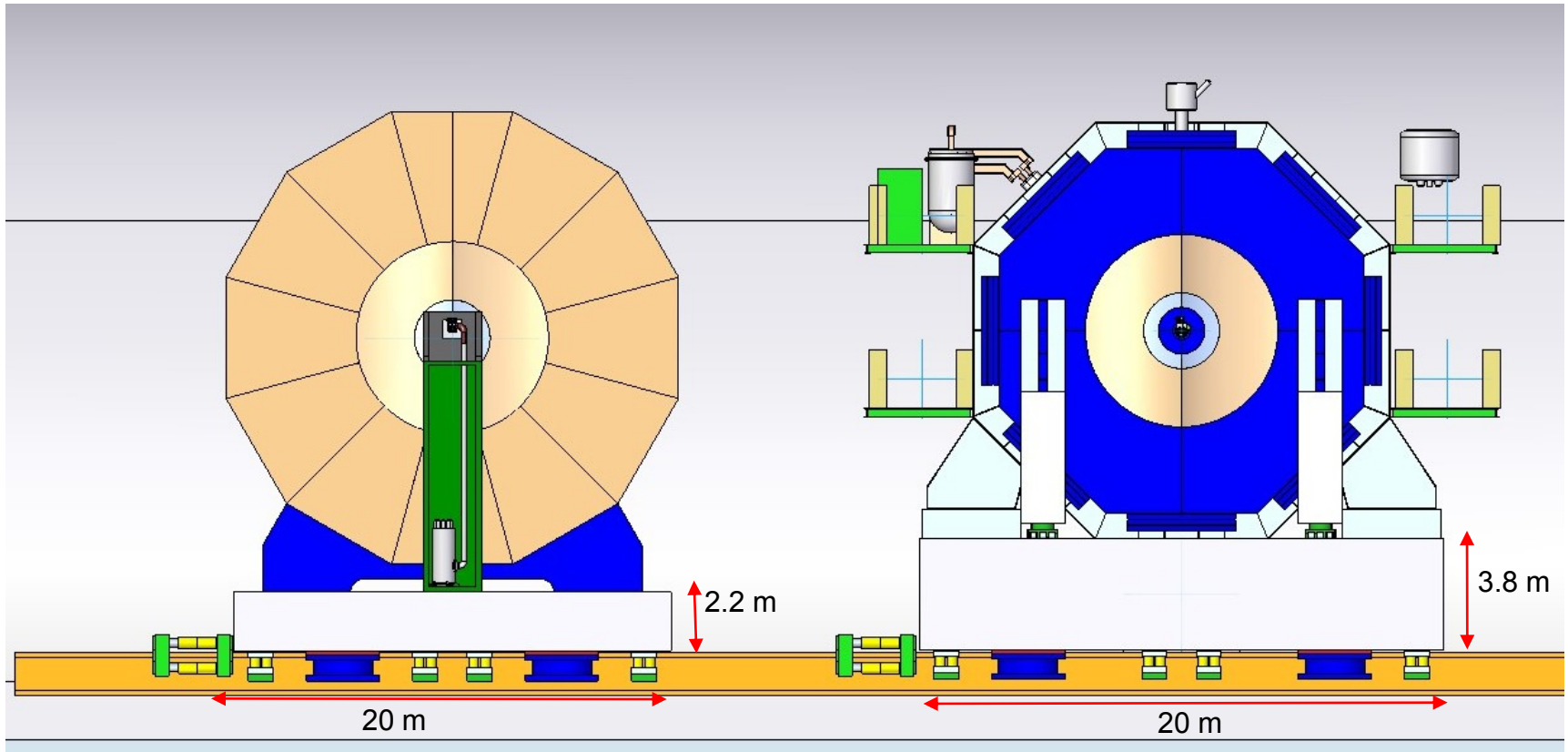
Reliability: Transport modularity must be such that repairs/replacement/maintenance can be accomplished in garage position and within 20 elapsed days.

Any equipment required for transport shall reside below the platform surface.

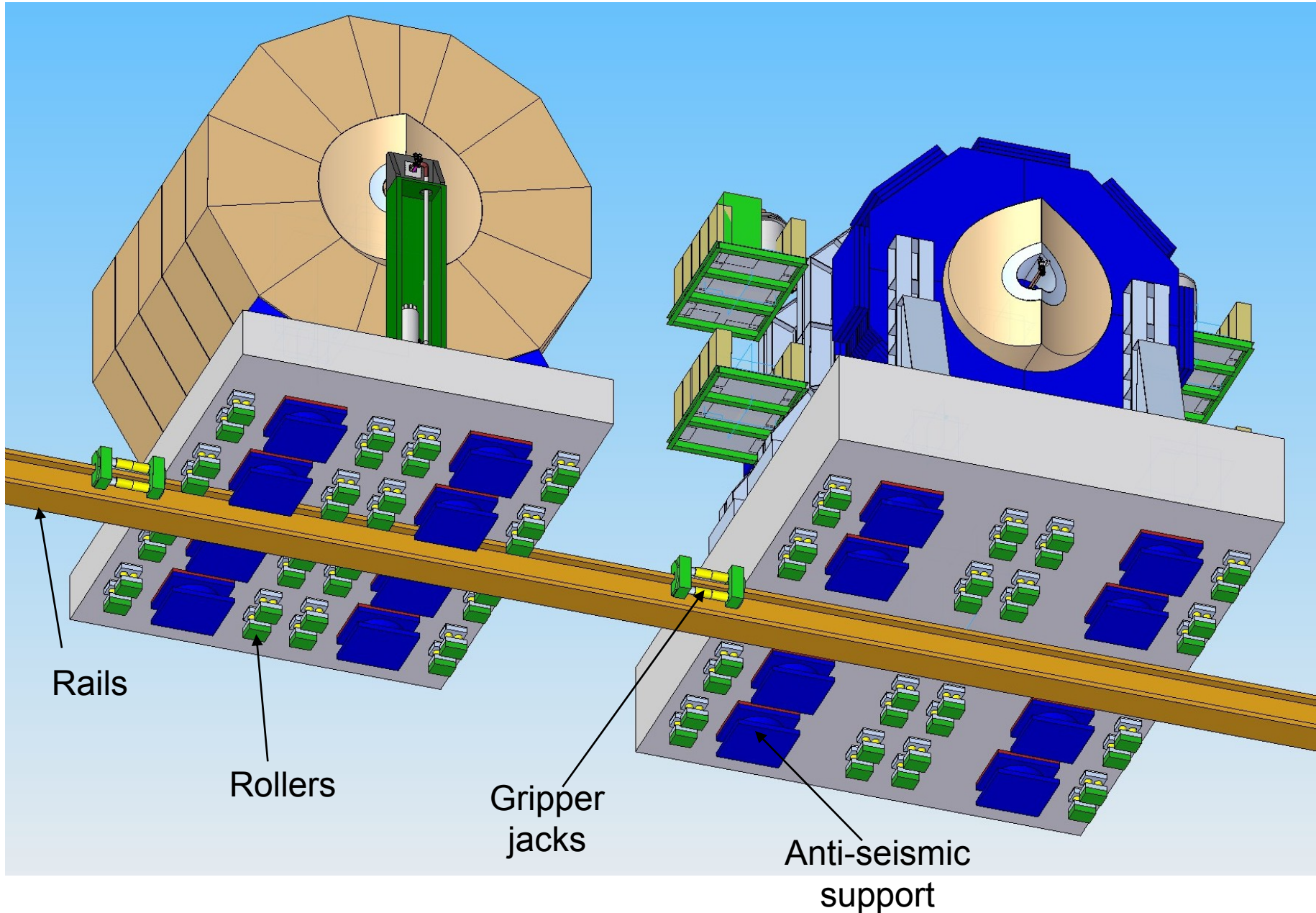
Transport equipment shall not eject particulates that reach platform surface (need spec on how much)

# Thick platform

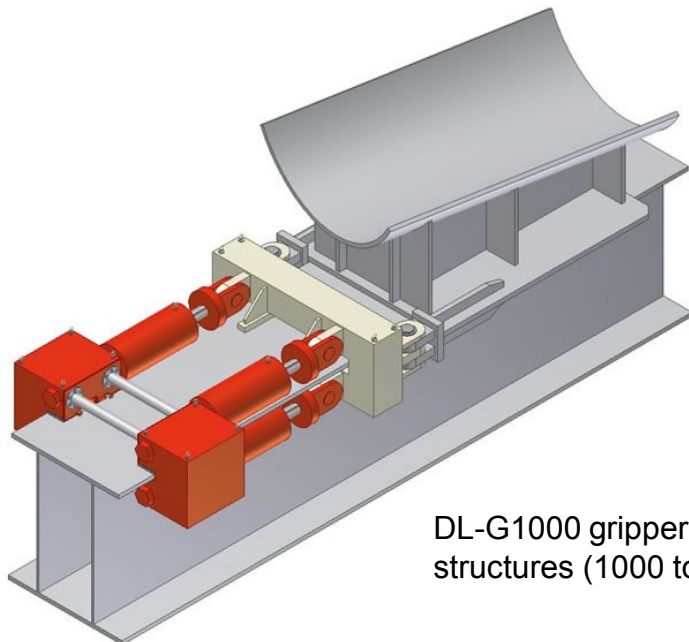
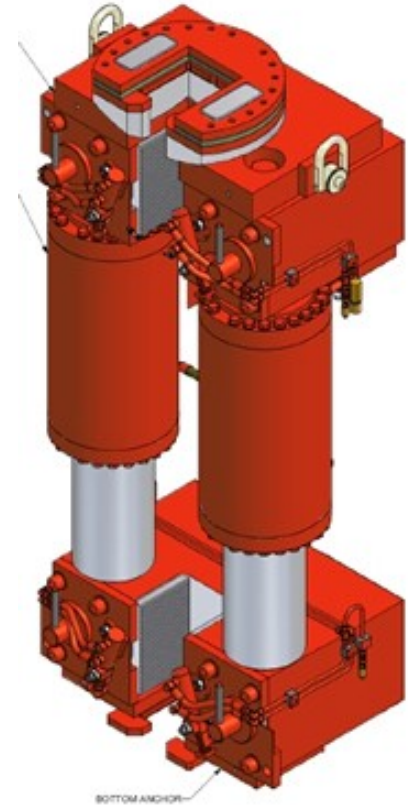
Extra Height to accommodate the difference of the two detectors



# Gripper Jacks on rail



# Motion system



DL-G1000 gripper jack for load out of offshore structures (1000 tonnes push / pull capacity)

Gripper Jacks, 1' 000 T

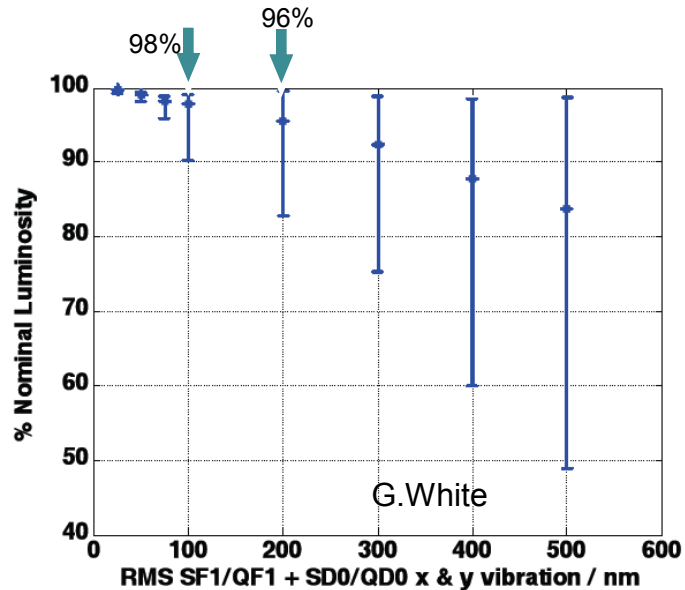
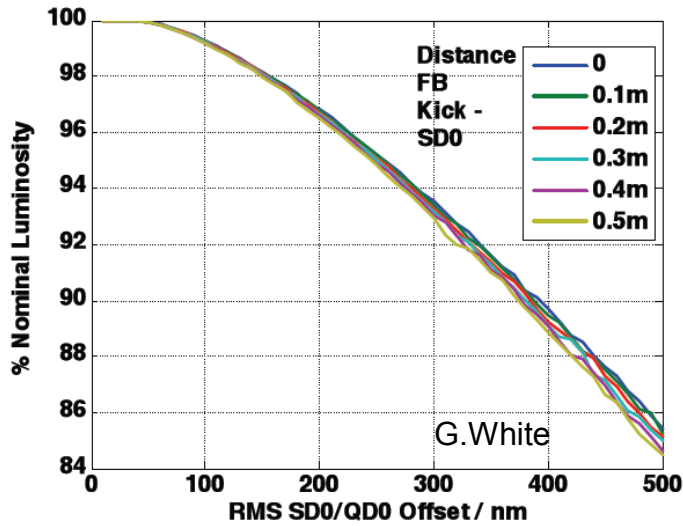
# Vibrations

## QD0 stability Requirements

Most acute luminosity loss mechanism due to relative jitter of final focusing magnet elements : Ground Motion and Mechanical vibration sources

ILC has Active Fast Feedback based on beam trajectory after collision

Max. Integrated displacement:  $100 \div 200 \text{ nm} > 5 \text{ Hz}$



Lumi loss due to beam offset in SD0 (beamsize growth) and IP misalignment of beams



# Vibrations : Absolute, Relative and Coherent and motion

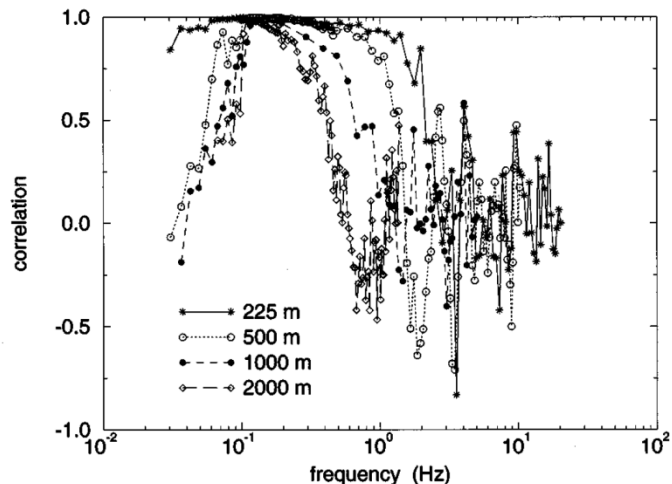
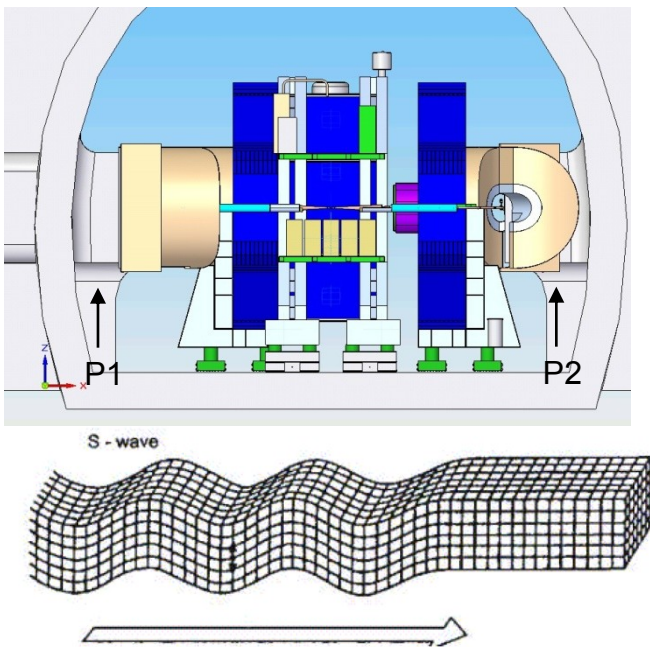


FIG. 3. Correlation spectra of ground motion measured at CERN in the LEP tunnel [7]. The distances between sensors were 225, 500, 1000, and 2000 m.

$$\text{Coherence} : N_{12}(f) = \frac{P_{12}}{\sqrt{P_1 P_2}} = J_0(\omega L/v)$$

If P1=P2, then :

$J_0$  = 0<sup>th</sup> Bessel function

L = distance between points

v = speed of sound in rock, ~3 km/s

$$\rho(\omega, L) = p(\omega) 2 \{ 1 - \text{Re}[N_{12}(\omega, L)] \}.$$

Relative displacement spectrum

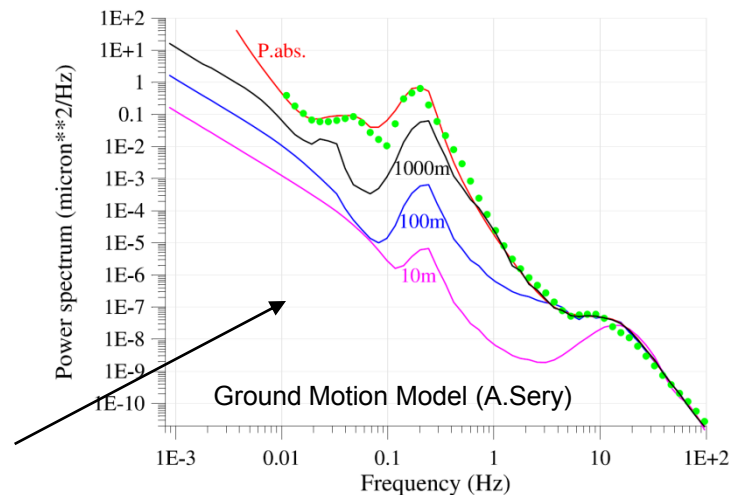
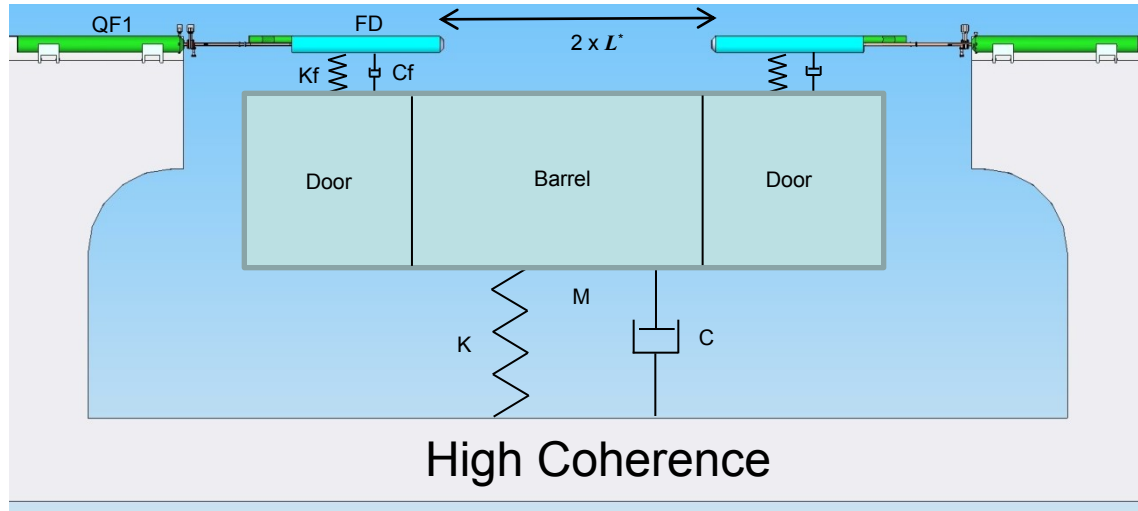
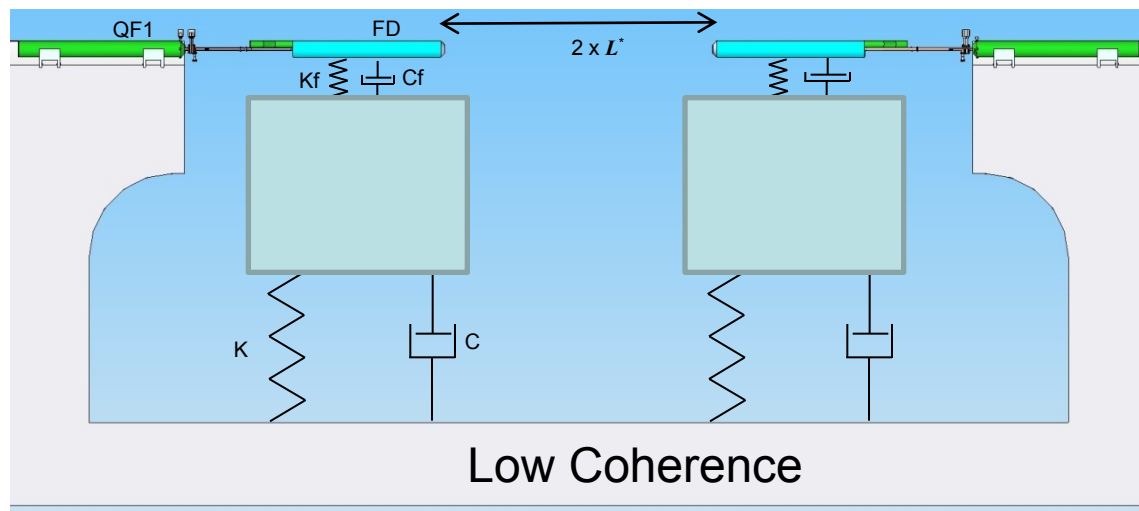
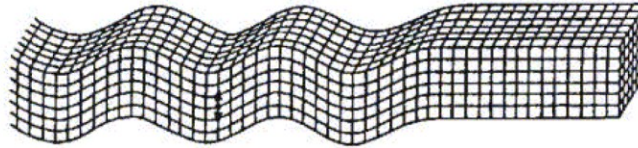


Figure 3: Measured (symbols) and modeling spectra  $p(\omega)$  of absolute motion and  $p(\omega, L)/2$  of relative motion for the 2 a.m. SLAC site ground motion model.

# QD0 Supports

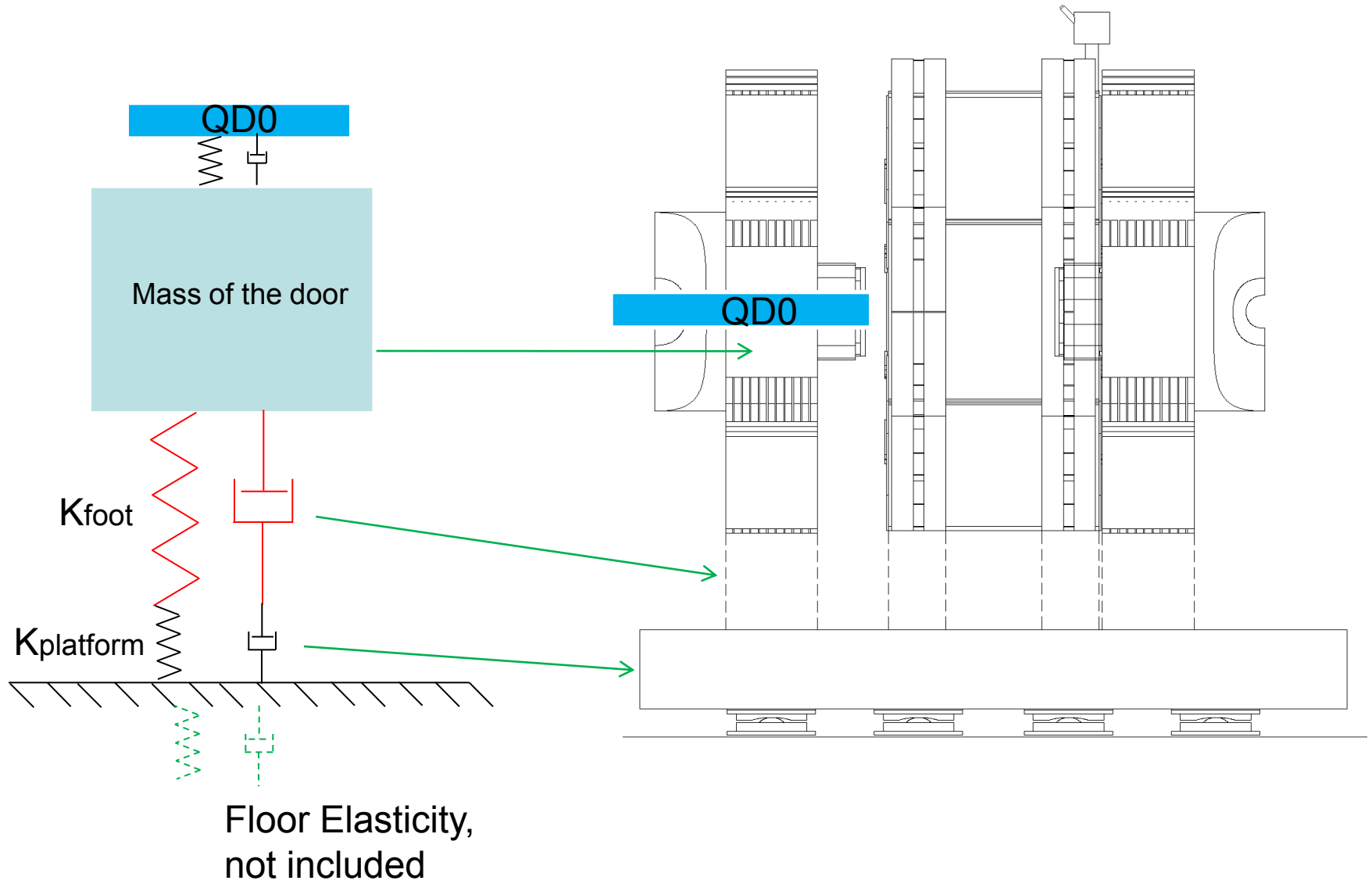


S - wave

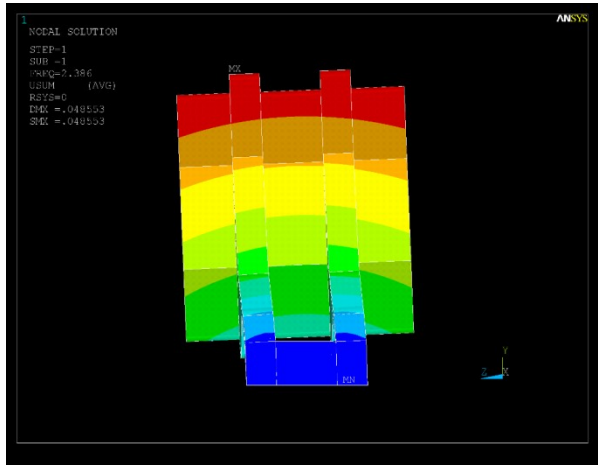




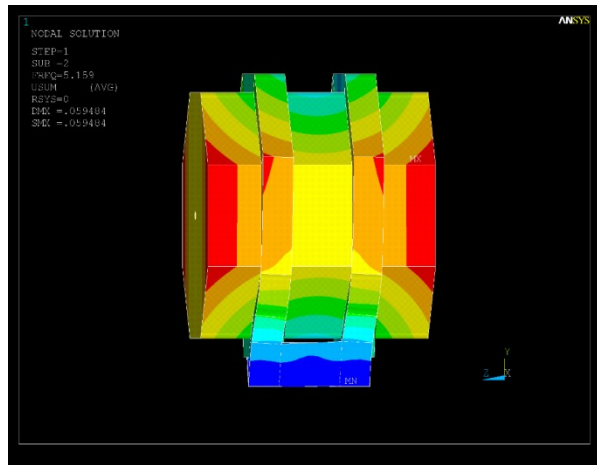
# SiD Vibration Model : 1 degree of freedom M,K,C oscillator



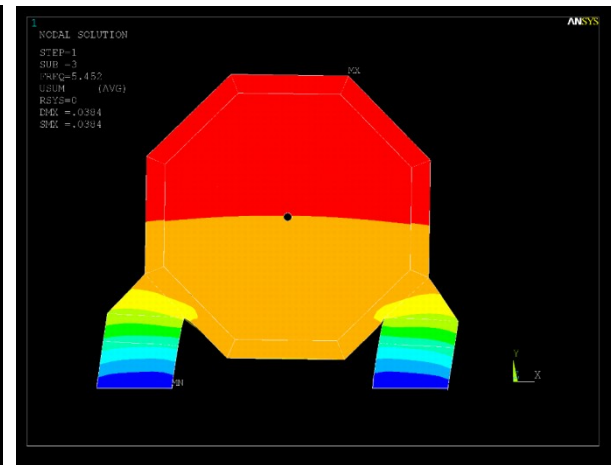
# SiD Free Vibration Mode



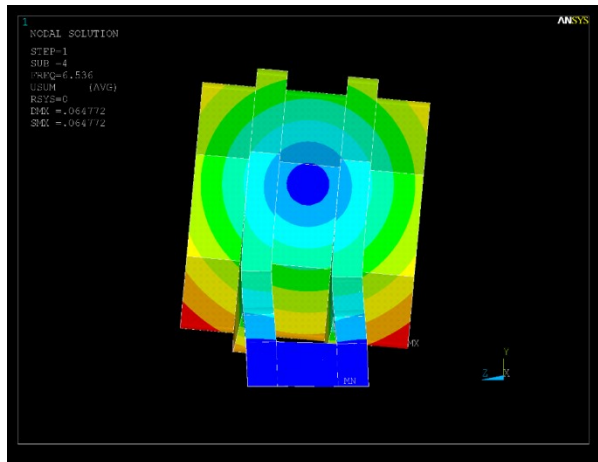
1<sup>st</sup> Mode, 2.38 Hz



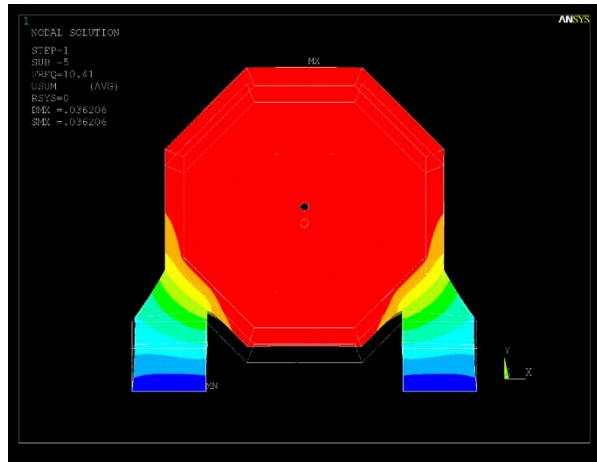
2<sup>nd</sup> Mode, 5.15 Hz



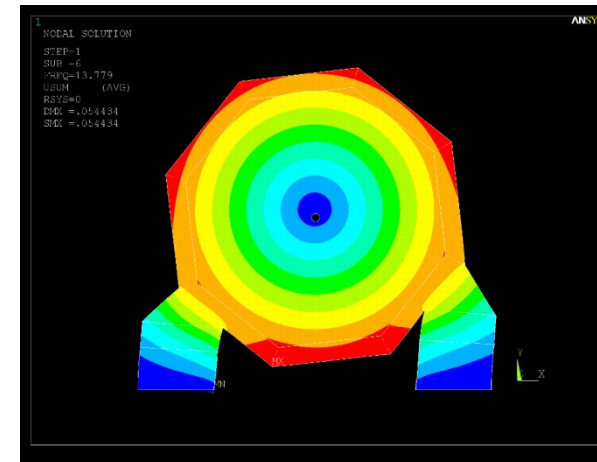
3<sup>rd</sup> Mode, 5.45 Hz



4<sup>th</sup> Mode, 6.53 Hz



5<sup>th</sup> Mode, 10.42 Hz

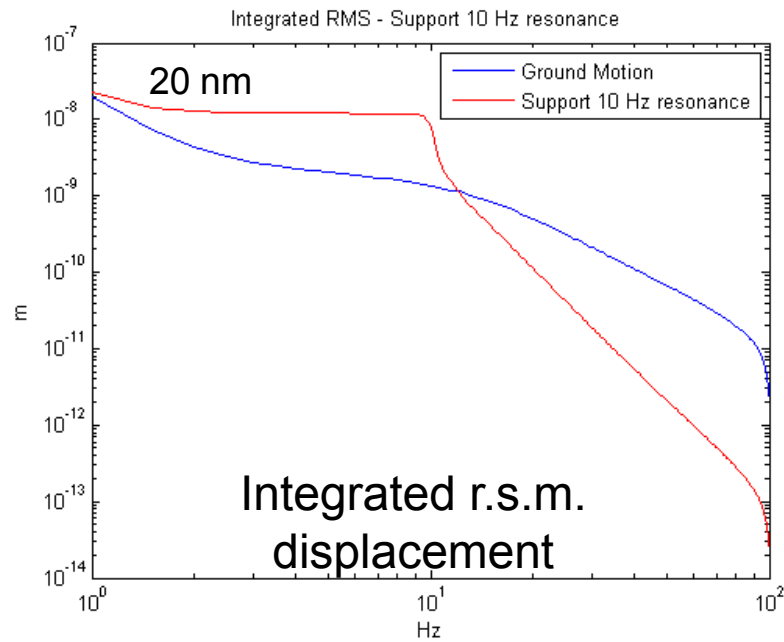
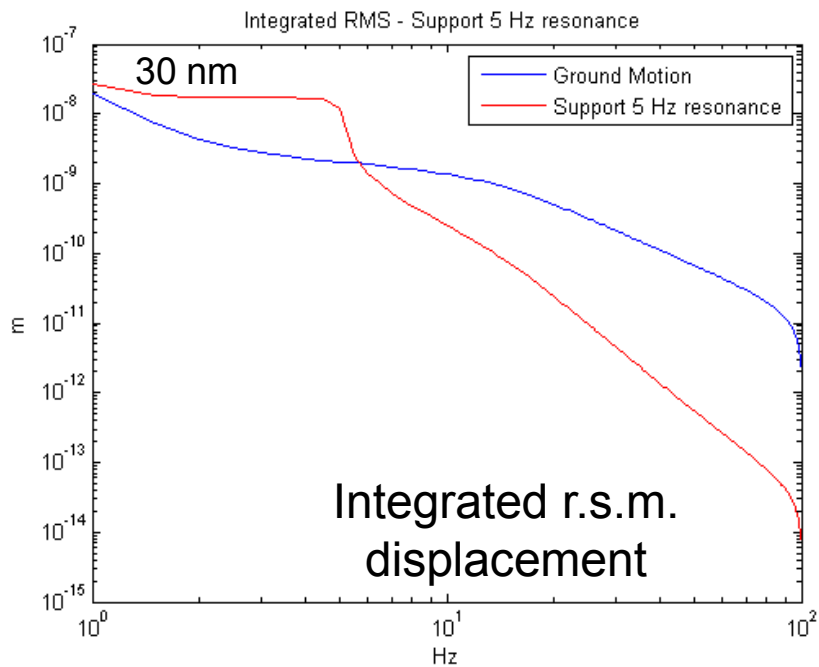
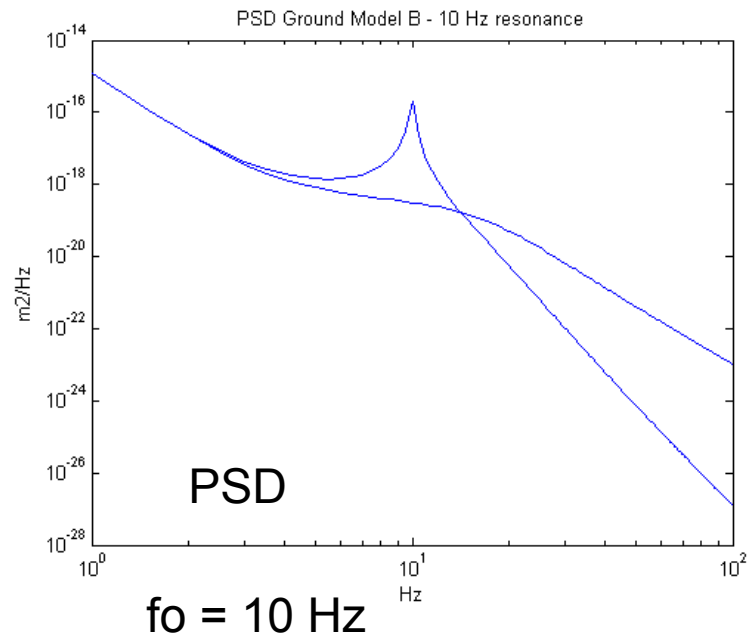
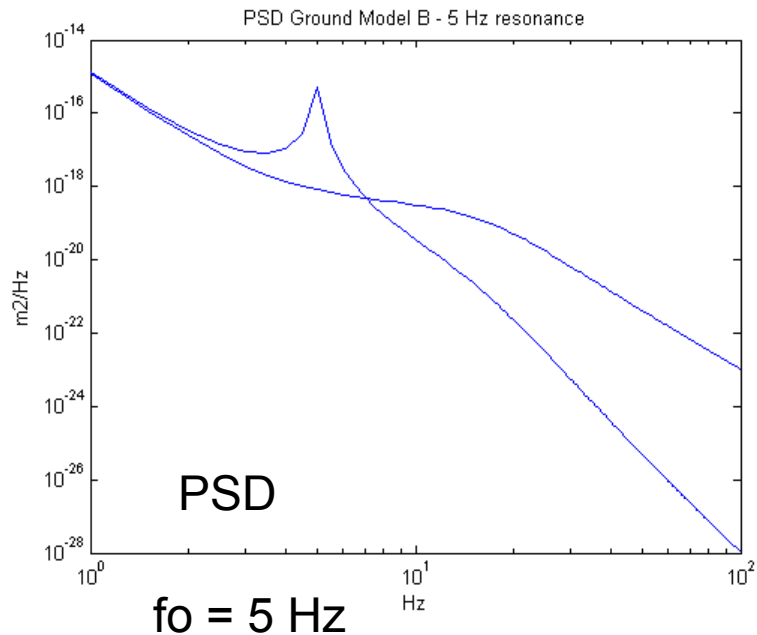


6<sup>th</sup> Mode, 13.7 Hz

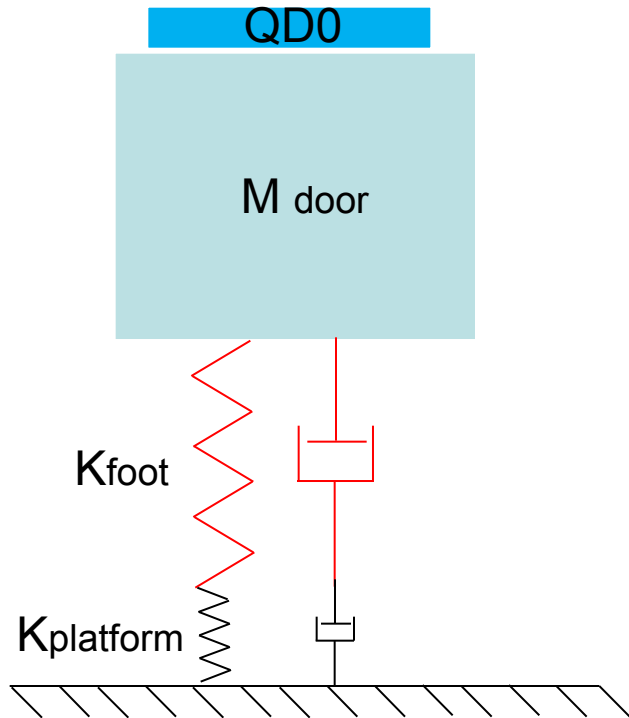


Vertical motion

# Random vibration Studies : SiD O.K. on the floor, no platform



# SiD Vibration Model : 1 degree of freedom M,K,C oscillator



$$f_n = \sqrt{\frac{f_f^2 f_p^2}{f_f^2 + f_p^2}}$$

1<sup>st</sup> mode system

$f_f$  = 1<sup>st</sup> mode SiD foot

$f_p$  = 1<sup>st</sup> mode platform

$c = 2\%$

$f_{\text{foot}} = 10 \text{ Hz}$  from FEA,

$f_{\text{platform}} = 15 \text{ Hz}$ , int. support,  
door-on-platform

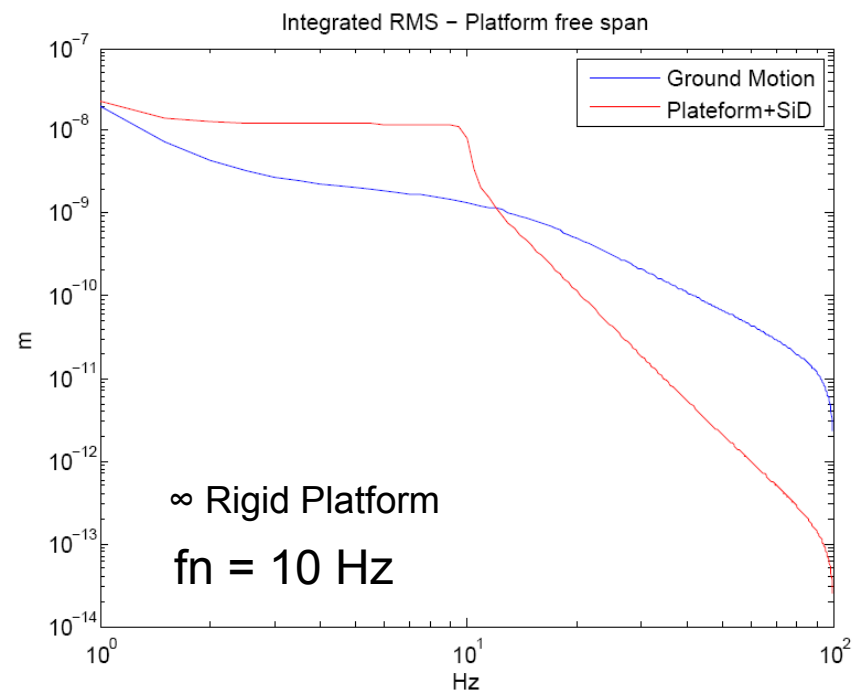
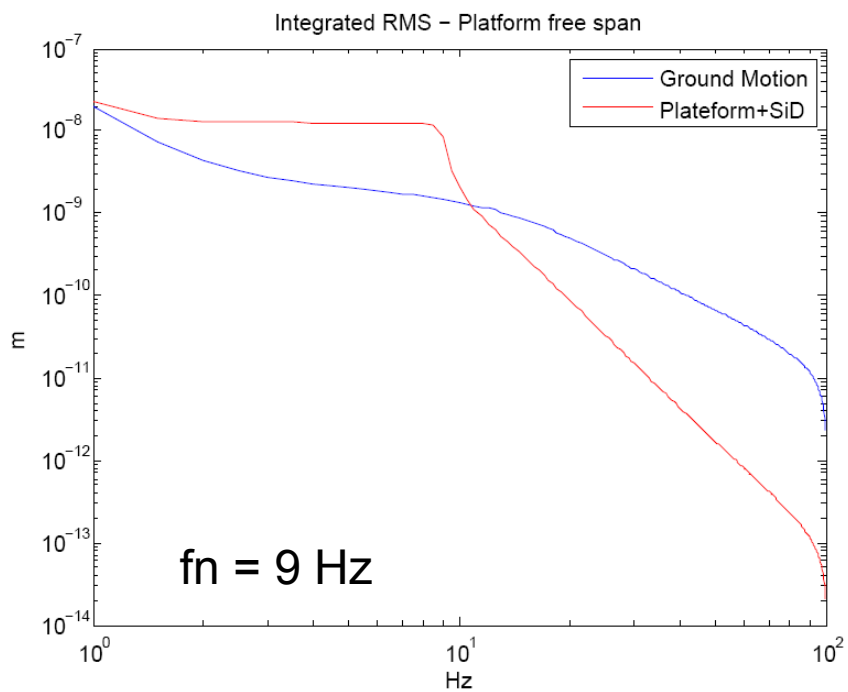
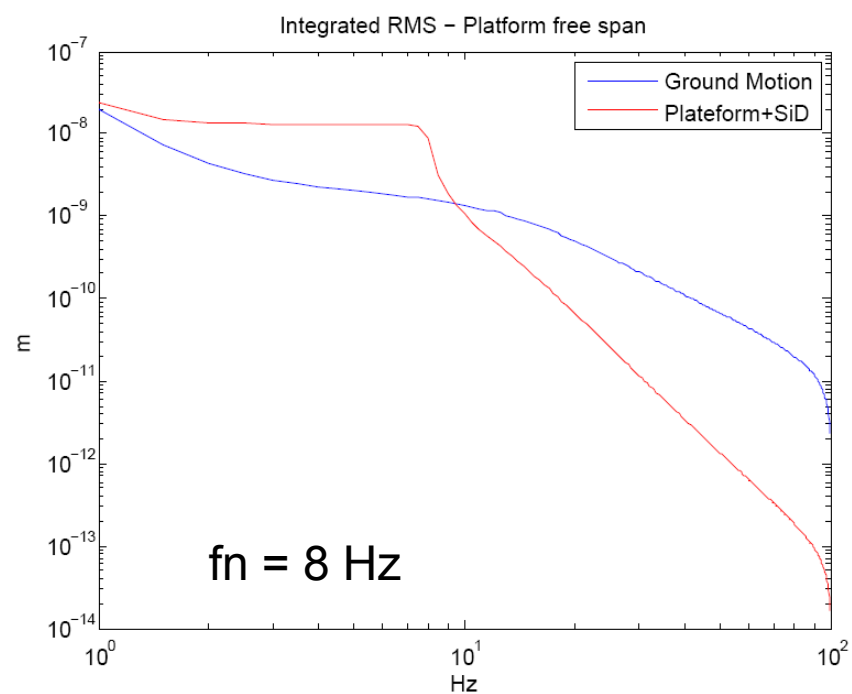
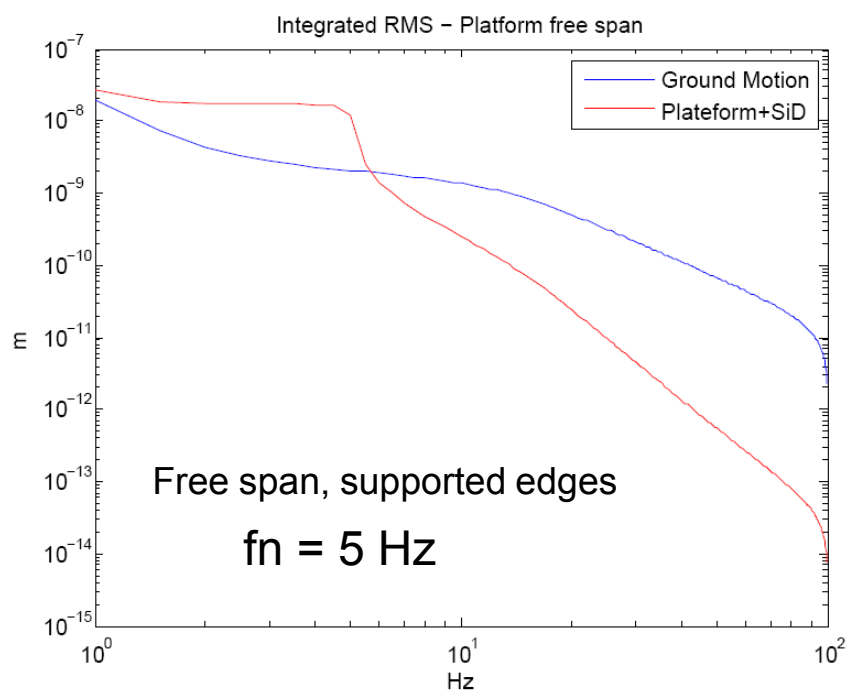
30 Hz, int.support,  
door-on-barrel

5 Hz

$f_n = 8 \text{ Hz}$

9 Hz

6 Hz, supported edges

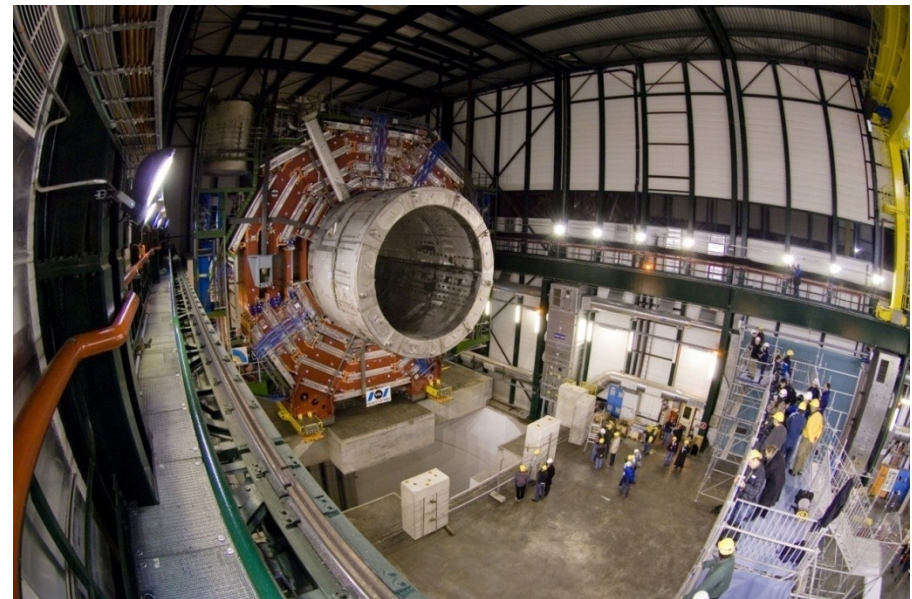


# Platform Simulation

Benchmark with exp.data

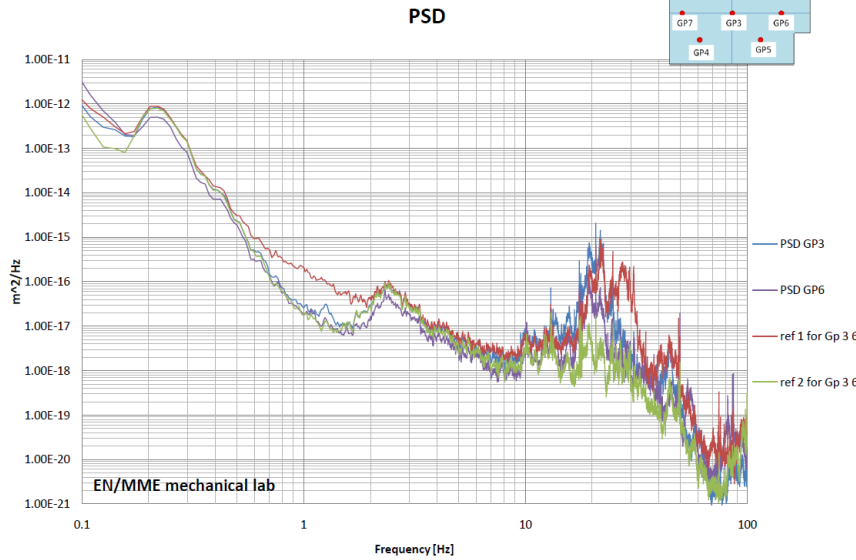


# The CMS Plug

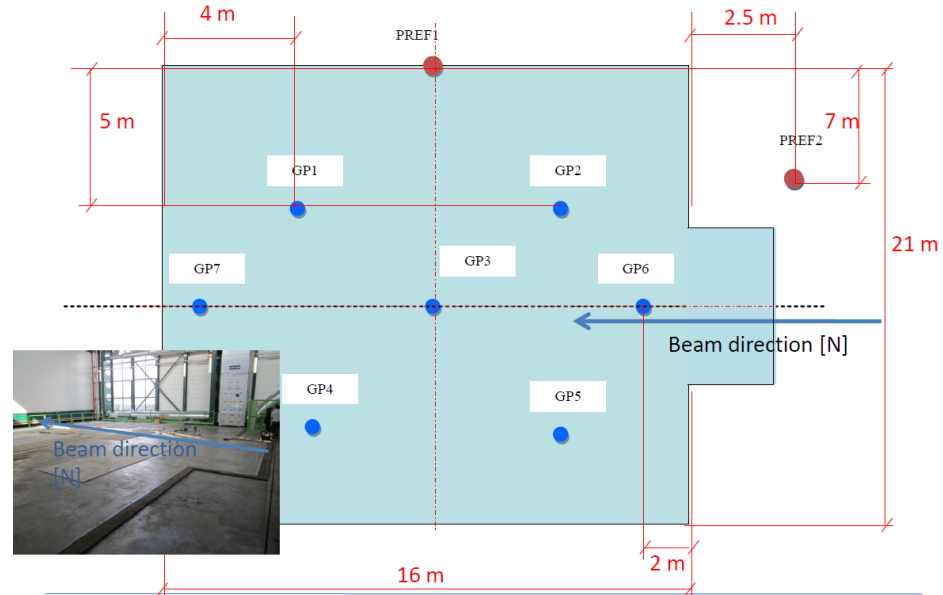


# Experimental Vibration measurements – CMS Plug

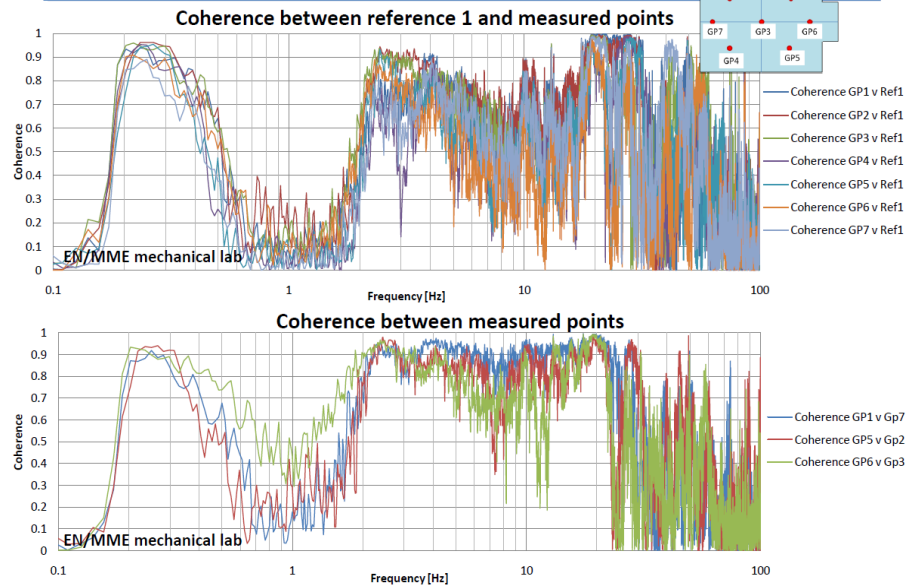
PSD for a typical measurement



Sensor position



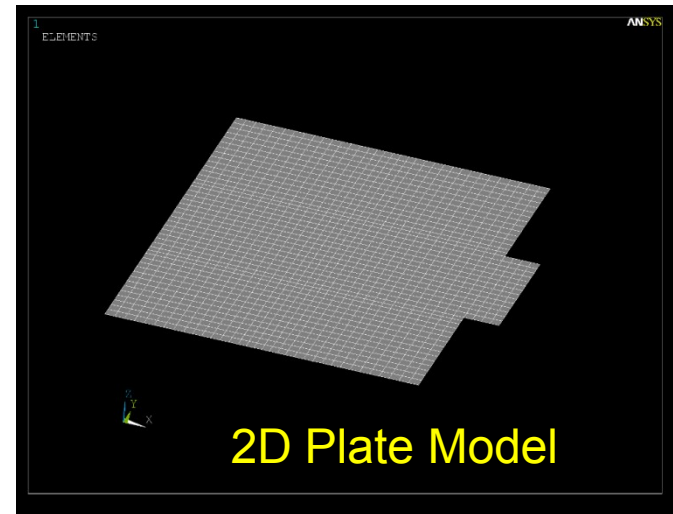
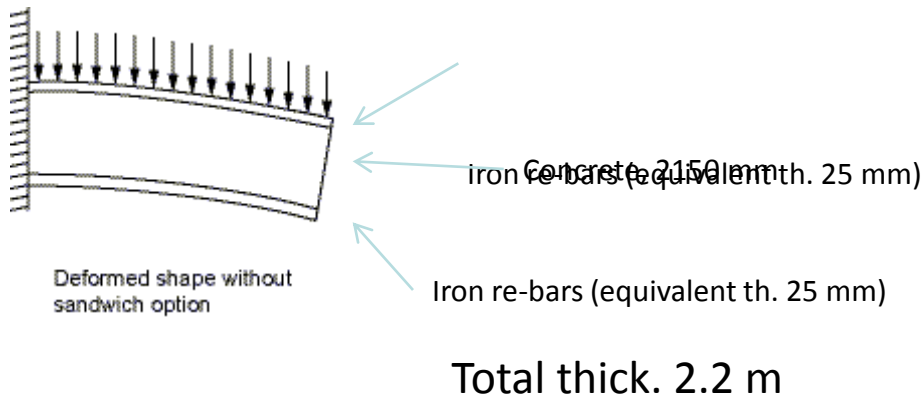
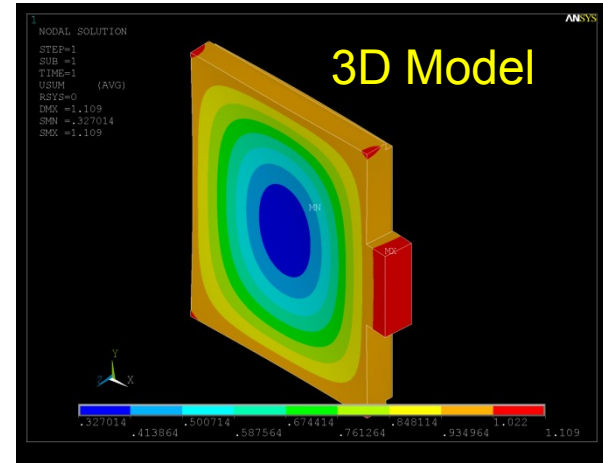
Coherence Vertical direction





# Finite Element Model, 3D vs. 2D

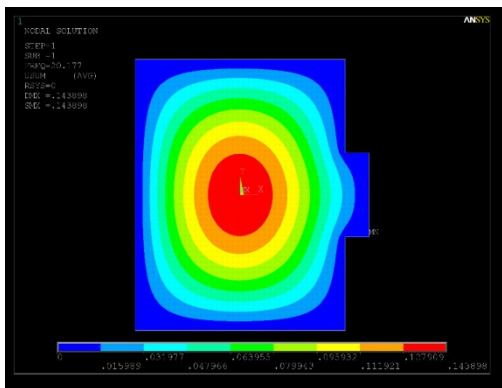
## CMS Platform



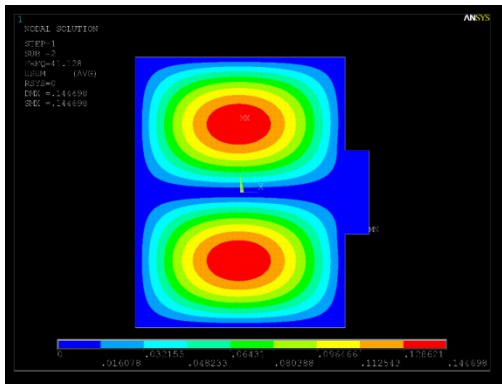
# Free modes

Mode	FREQ
1	20.17
2	41.12
3	53.24
4	72.76
5	73.28
6	95.85

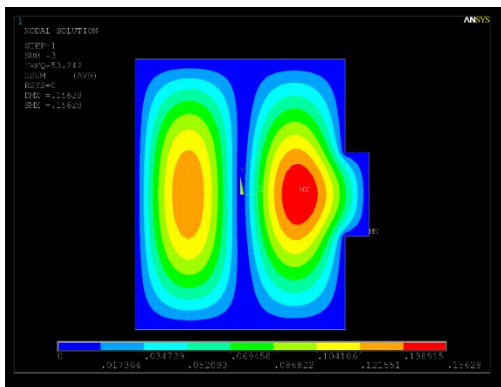
1



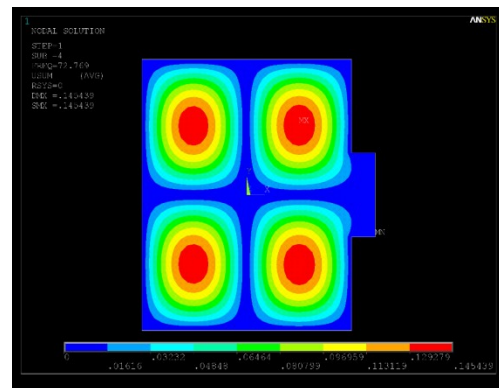
2



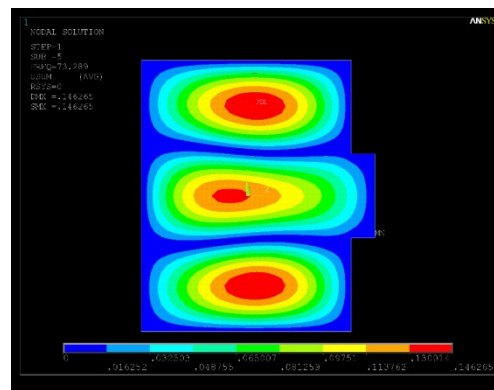
3



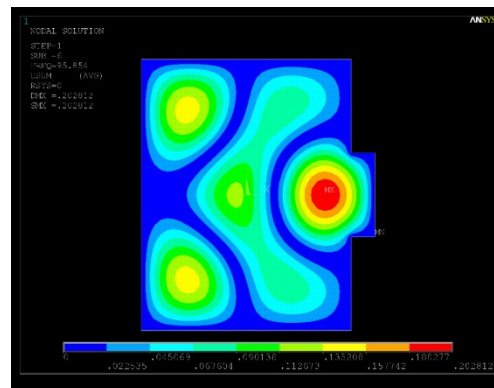
4



5

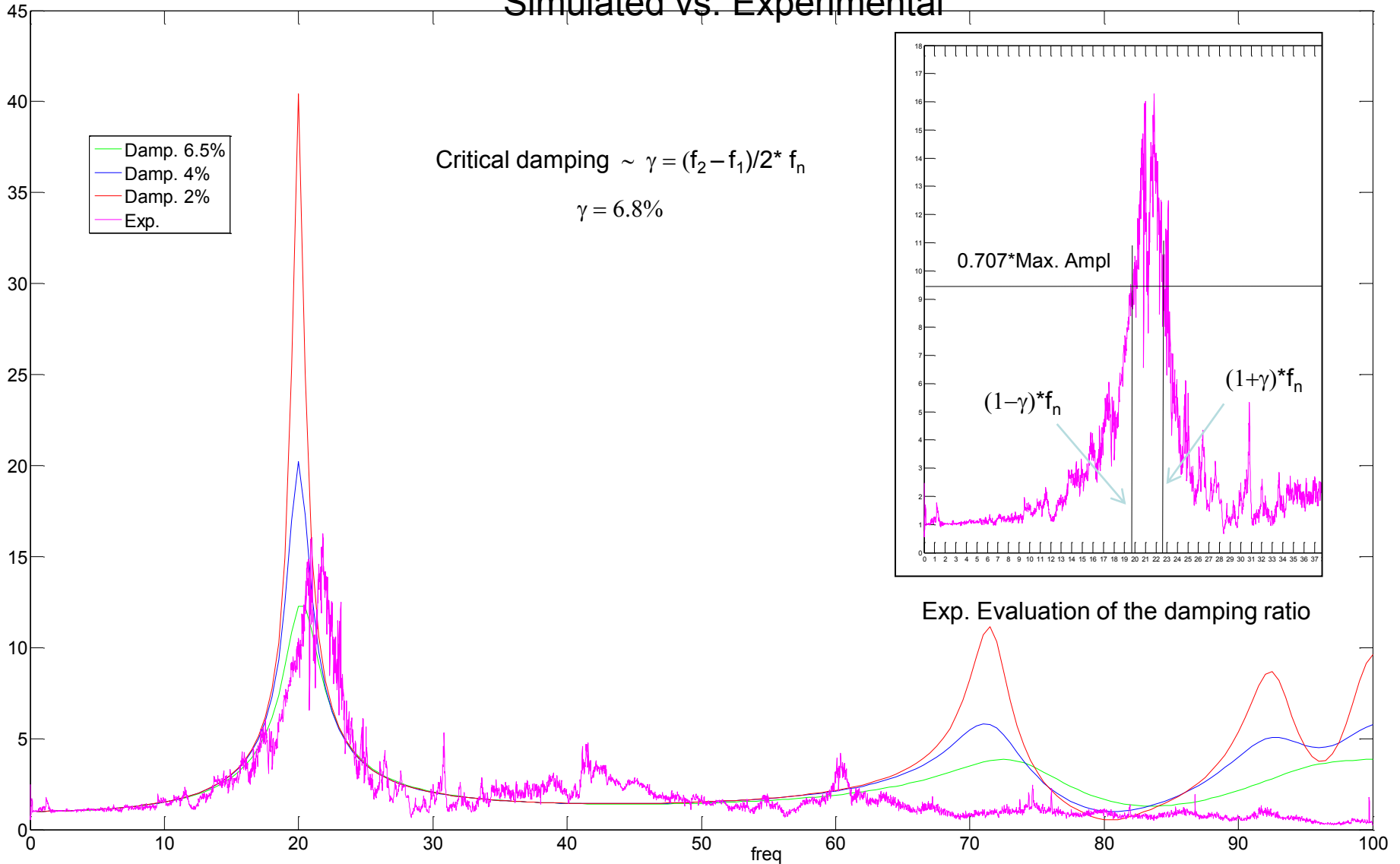


6

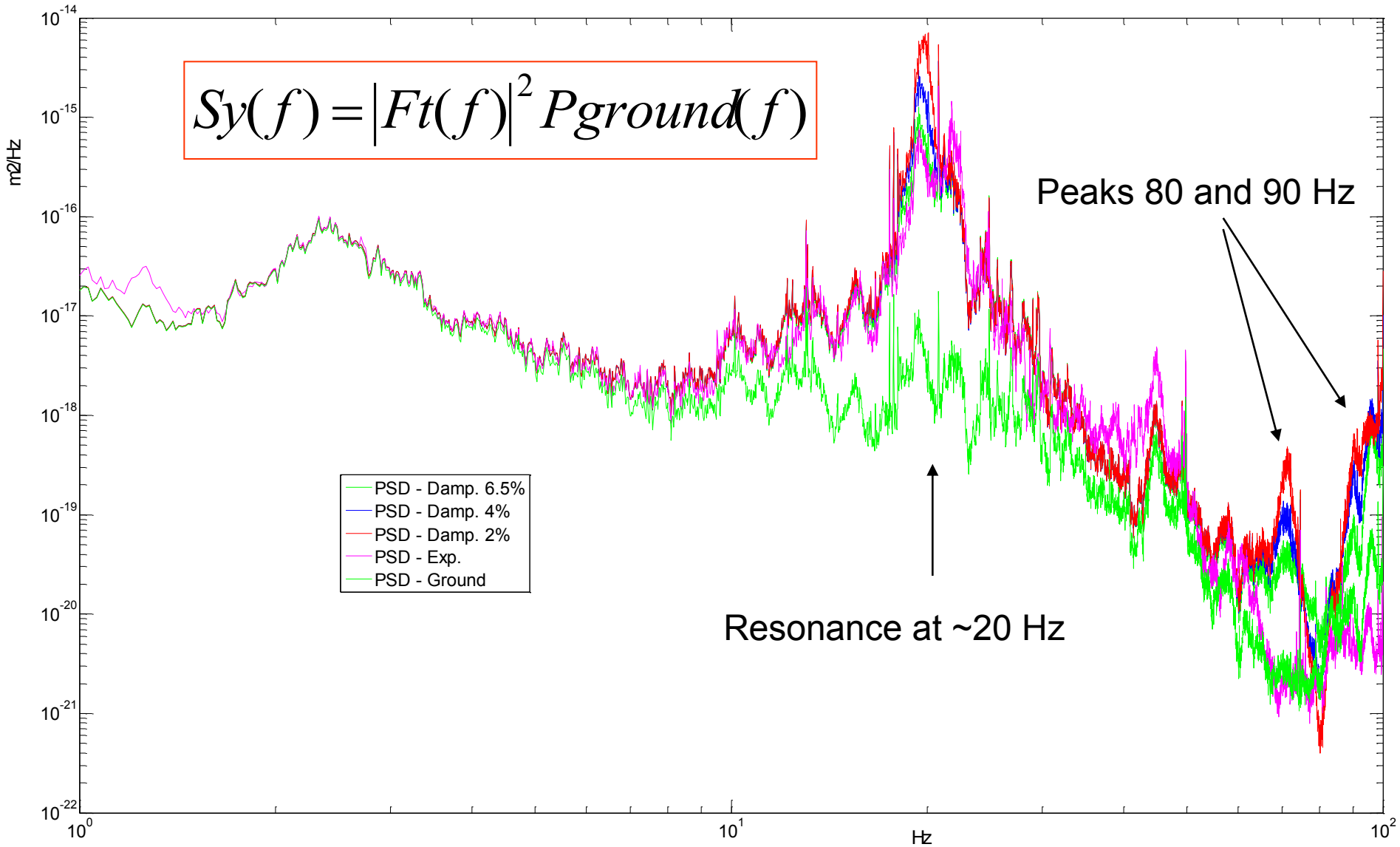


# Transfer Functions - Middle Point (Geophone N.3)

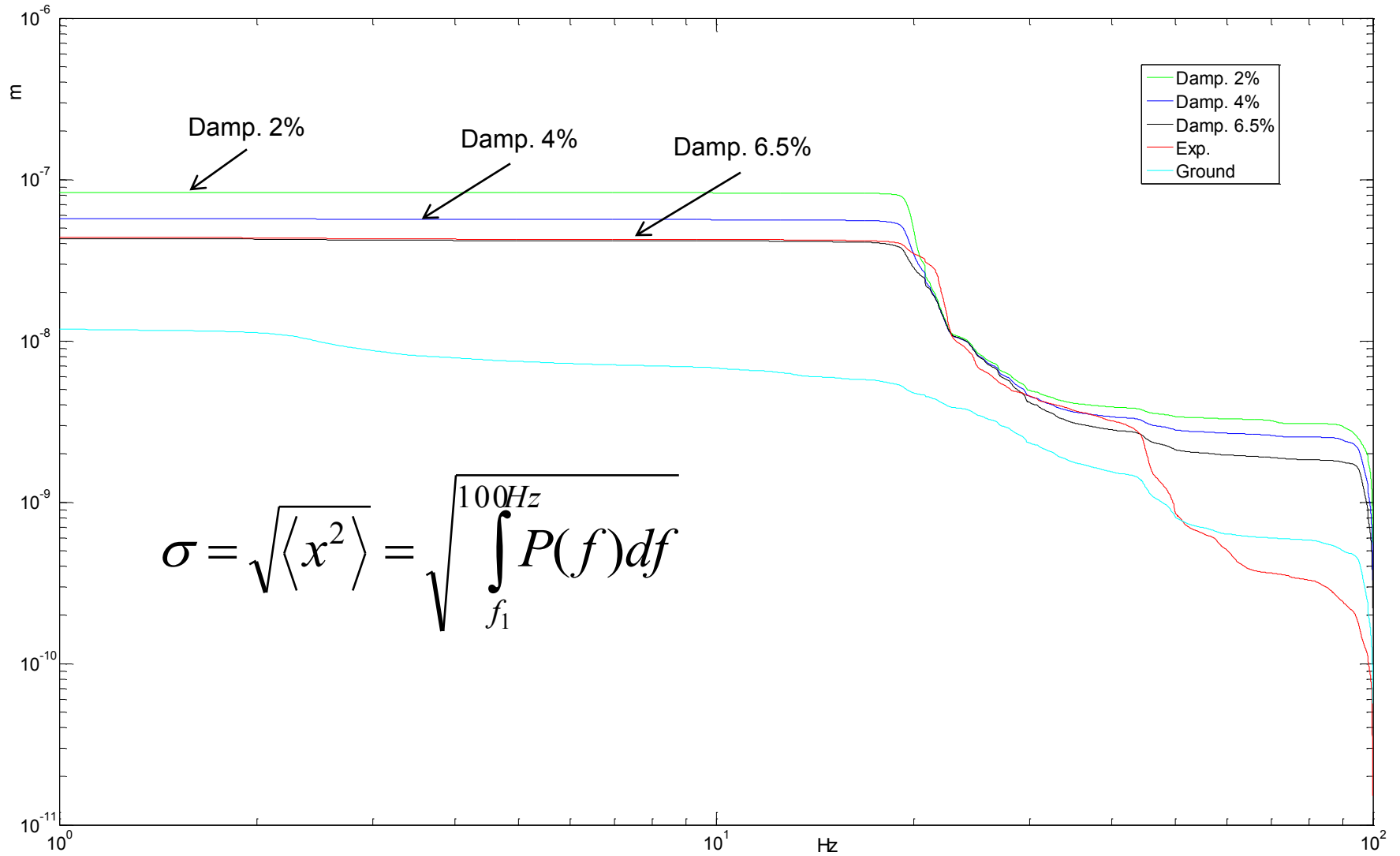
## Simulated vs. Experimental



# Simulations vs. Measured Power Spectra (Platform Center)



# Integrated Displacement (r.m.s.)






# Summary

SiD is designed with the QD0' s supported from the doors

SiD can be moved without a platform, ILD can't. The Platform is the only compatible solution, which does not require modification in the design of both detectors

 SiD will move on a platform upon condition it meets functional requirements : dimensions, static, vibration, floor, etc

 With these requirements, we expect the platform designed by the CFS group. The two platforms do not need to be identical.

The effects of vibrations on beam stability remain a subject which need more studies

New set of experimental vibration data available for the CMS Pug/platform

The data have been used to benchmark FEA with positive results.

Experimental characterization of the dynamic properties of reinforced concrete structure is underway (See K.Bertsche talk today)