

ILD integration studies

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MDI integration meetings frequency since February : each months, face to face

Huge amount of work ⇒ in this presentation : focus on integration studies.

- Recent progress of some subdetectors groups on integration studies:
 - Yoke
 - Calorimeters
 - TPC
 - Inner detectors

Taking into account specificities of a mountain site : access, available space, seismic considerations..

- Services and patchpanels : highlight on inner part
- 3D CAD model status

- Conclusion(s)

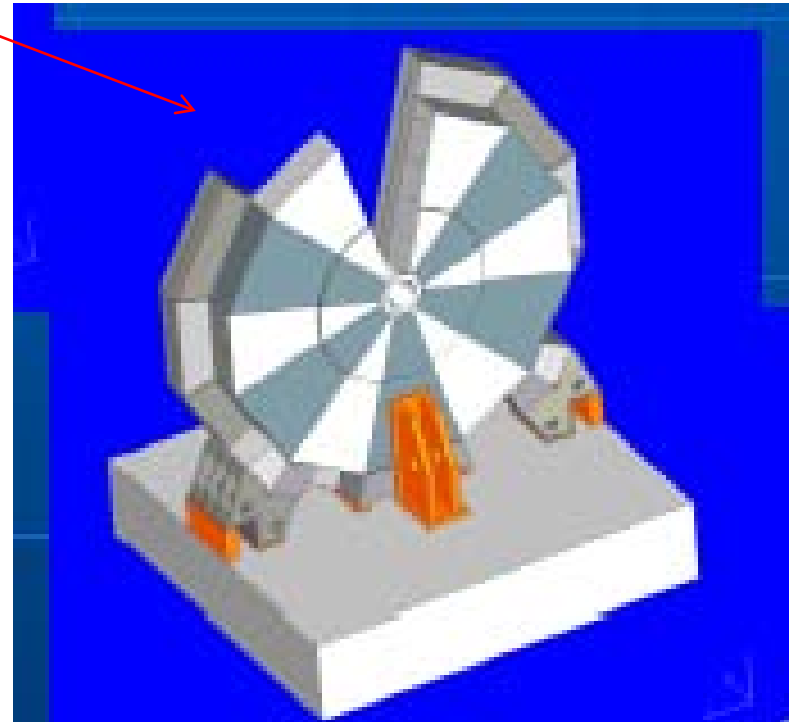
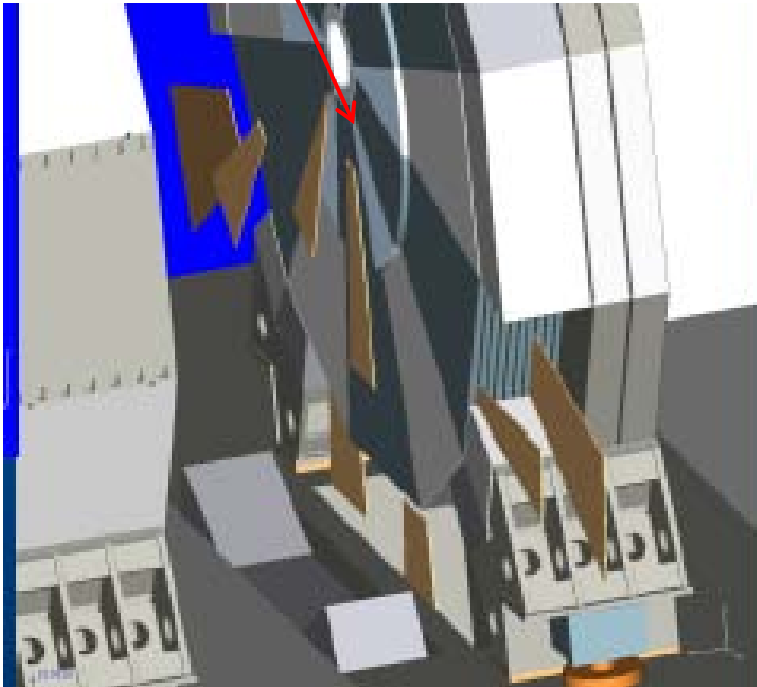
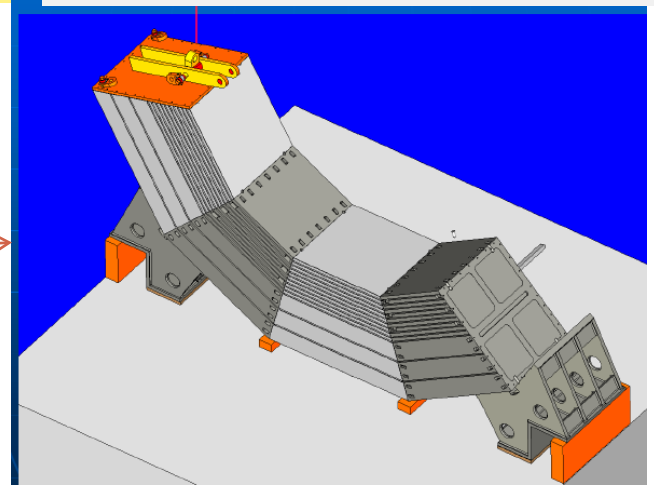
Return Yoke, endcaps Yoke and Coil installation :

From R.Stromhagen (DESY)

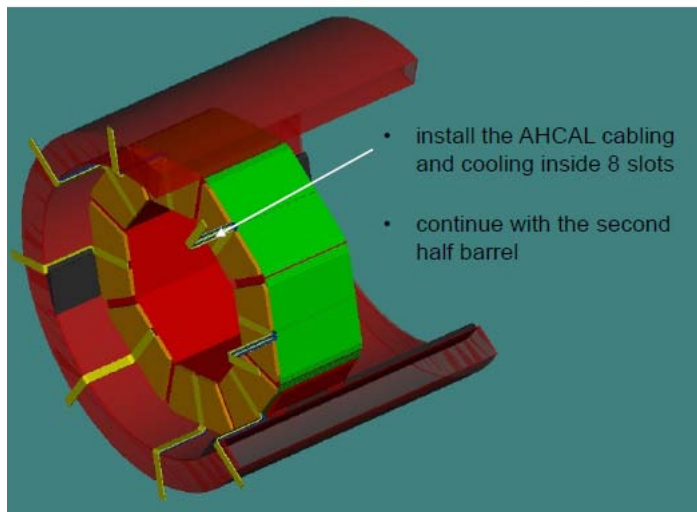
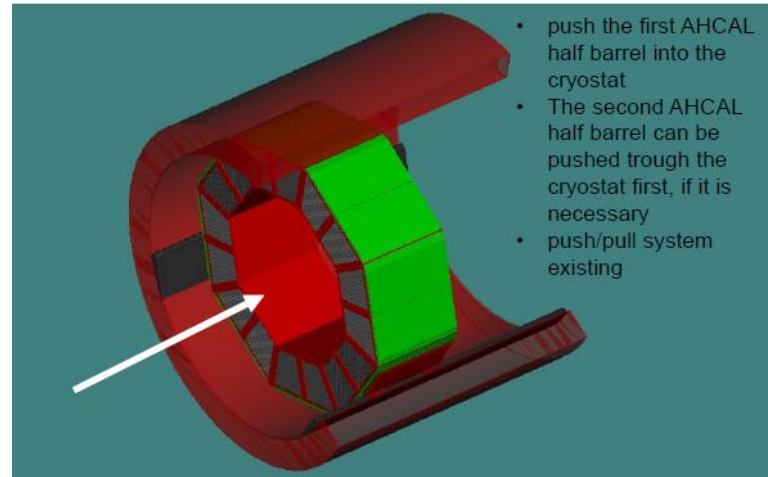
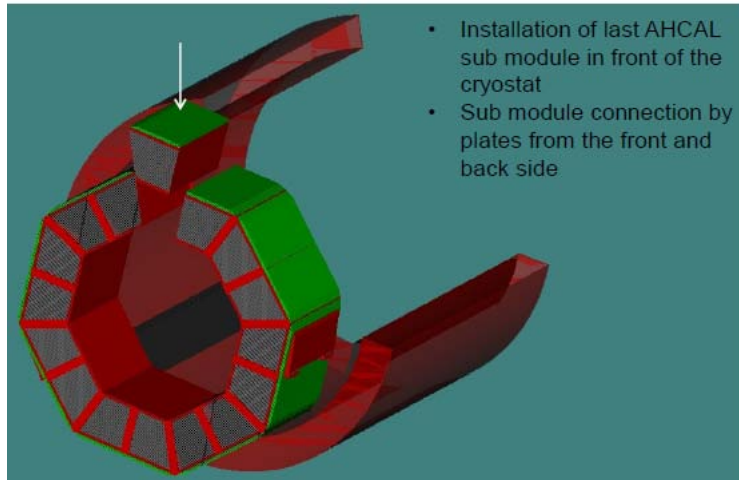
Structure if Yoke is the same whatever the configuration of the site is

Detailed studies of the building of :

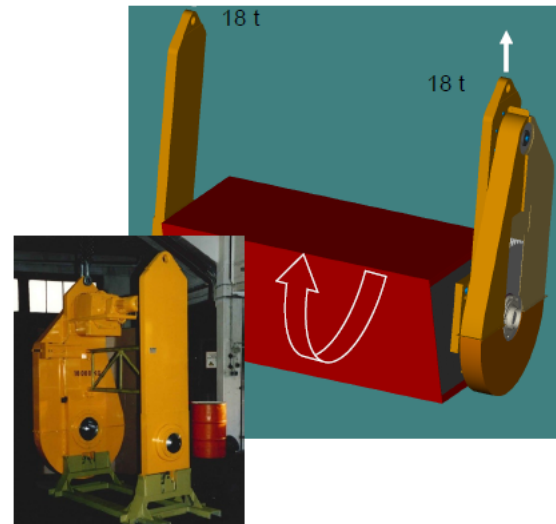
- return Yoke rings
- Endcaps
- Muons chambers insertion...



AHcal integration sequences



Integration tooling

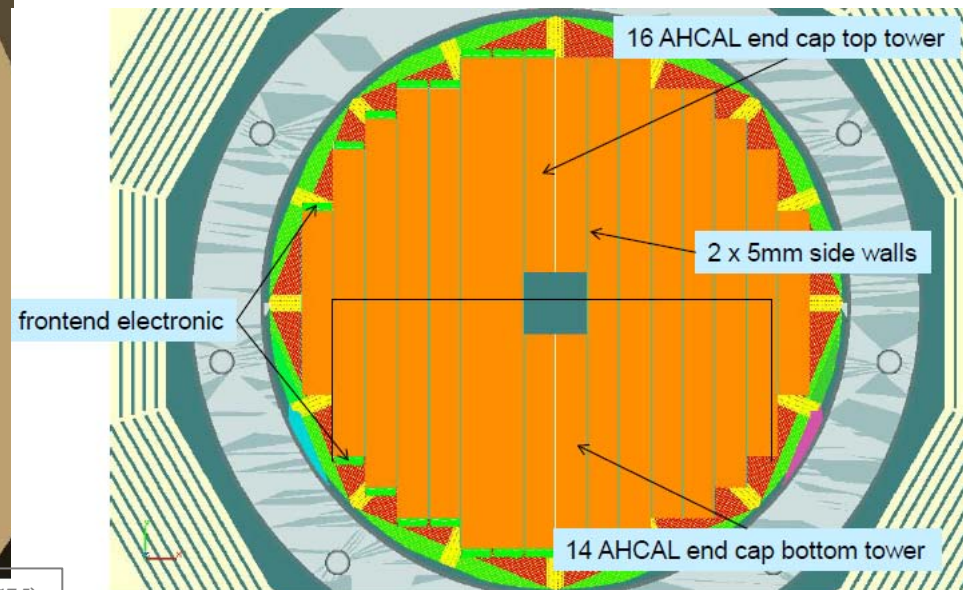
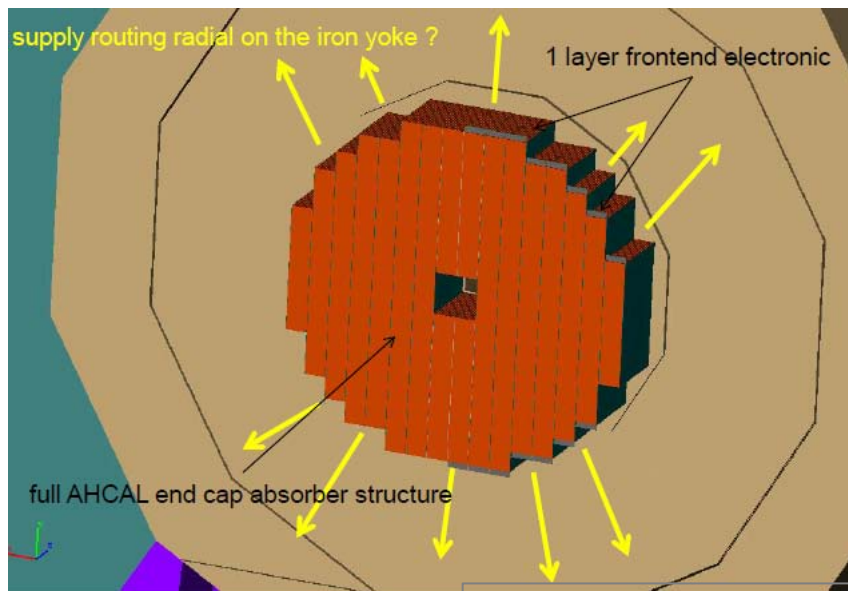
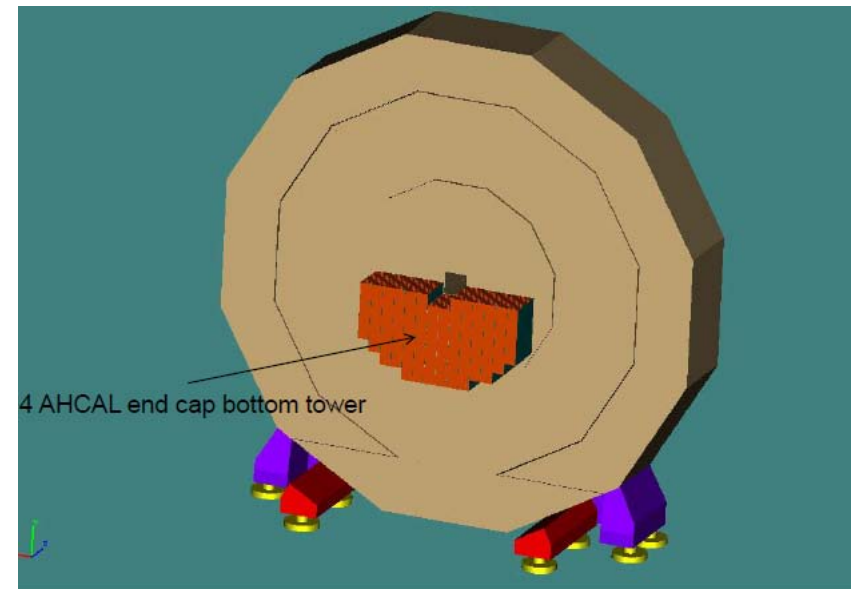
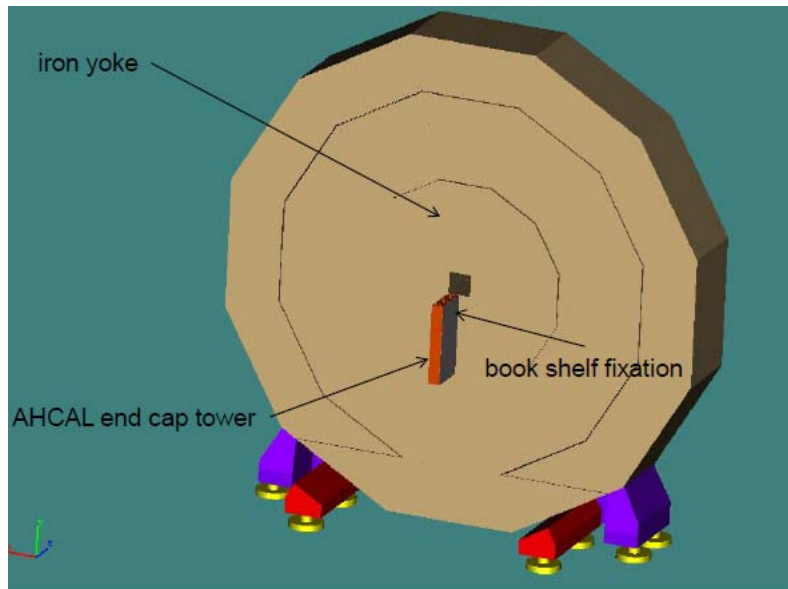


- > lifting and turning tool for AHCAL barrel absorber sub-modules available
 - 2 x 18 t capacity
 - operation with 2 hooks (z angle adjustment)
 - precise motor controlled turning
 - design for adaptation for sub-modules with and without sensitive layers started
- > mounting, support and insertion frame
 - one frame for everything
 - design depends on installation procedure
- > push and pull tool available
 - must be modified to the rail distance and rail shape/size

From K. Gadow (Desy)



AHCAL integration Endcaps studies

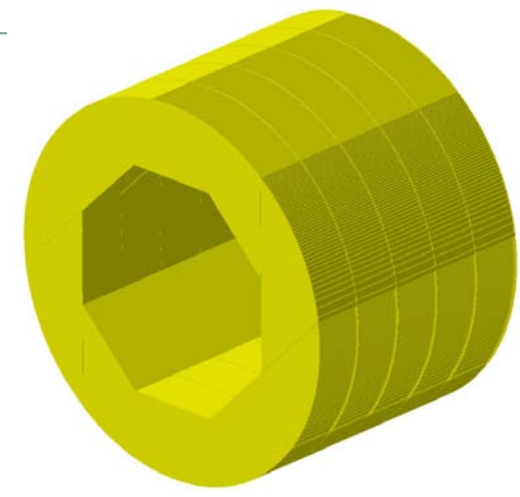
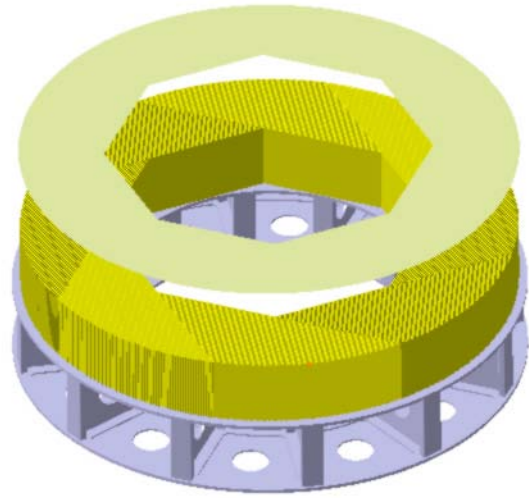


From K. Gadow (DESY)

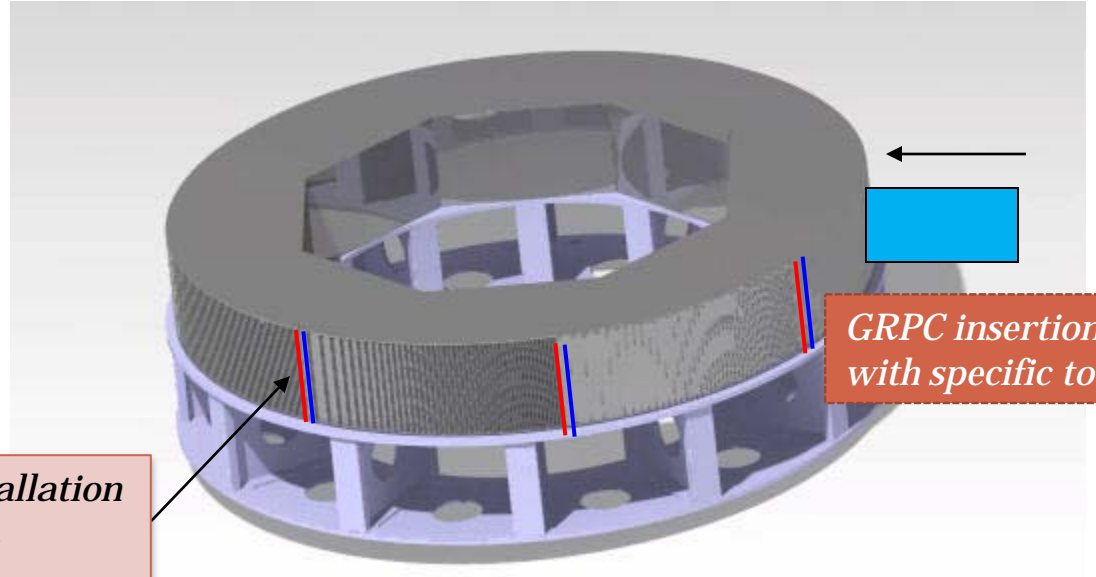
SDHcal integration studies : the 5 wheels concept .

Building SCENARIO :

- 8x48 in position on specific tool
- 1 face put down
- 8*48 plates welded on one face
- One other tool in place
- 180° rotation
- 8*48 plates welded on this other face



GRPC
insertion



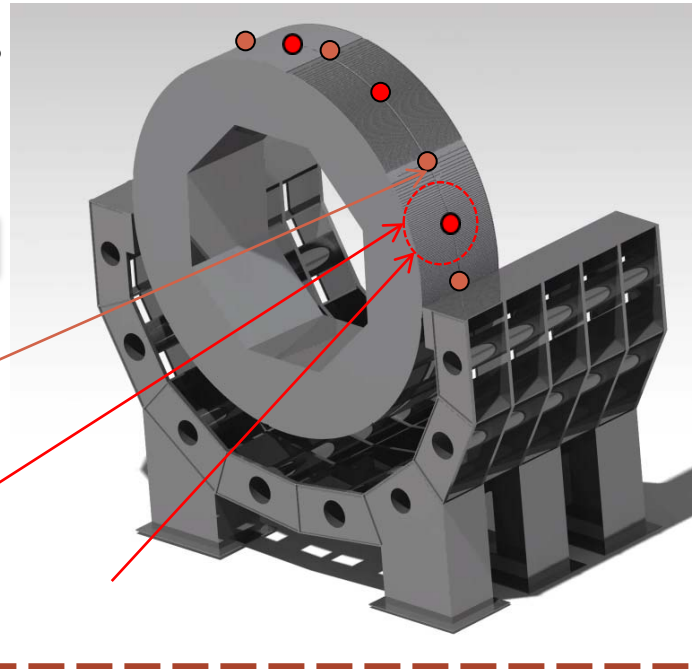
*Common services Installation
Ready to be connected*

*GRPC insertion
with specific tool*

From J.C.Ianigro (IPNL)

SDHcal integration, Barrel installation .

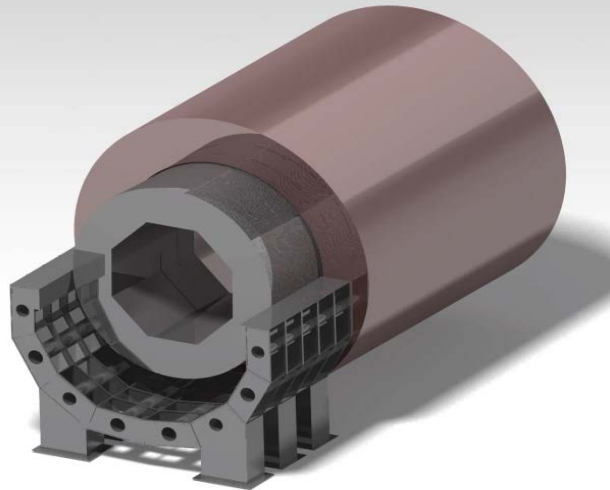
Bolting of the 5 wheels



Wheels carried one by one - P=200 t

services connection in 8 zones

Wheels linked together by bolting on 8 points

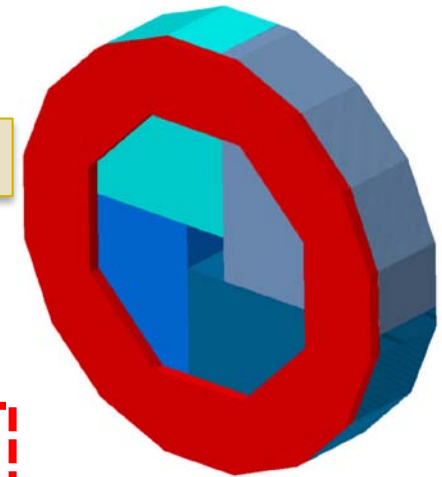


Insertion in the coil with a insertion tool « à la CMS »

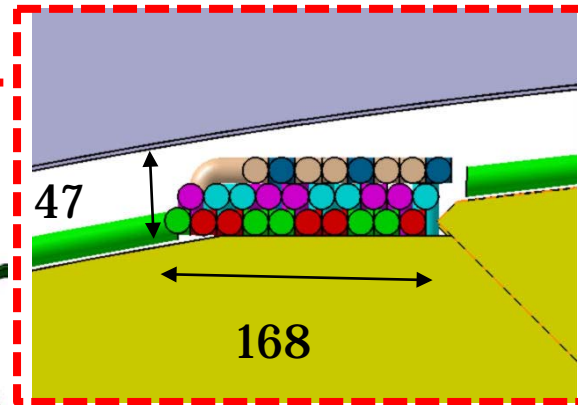
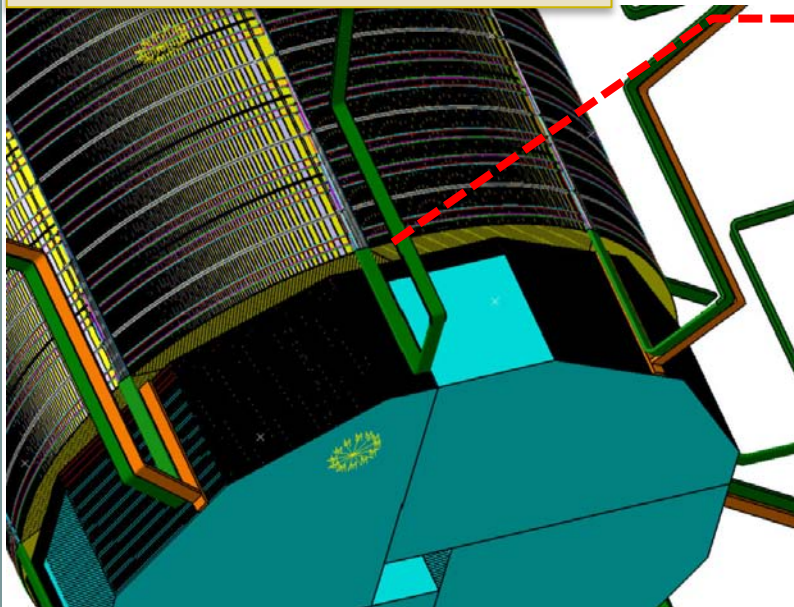


From J.C.Ianigro(IPNL)

Each endcap of 4 modules + Endcap rings (red)



Services distribution



Cooling considerations

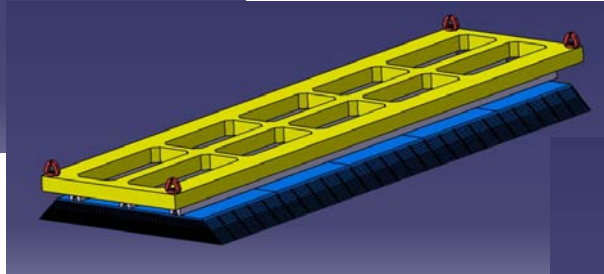
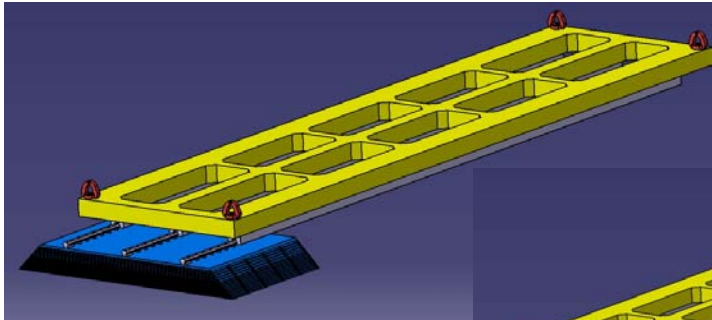
Cooling Options after local simulations

Low heat to extract: 0.3 W/m^2 for GRPC

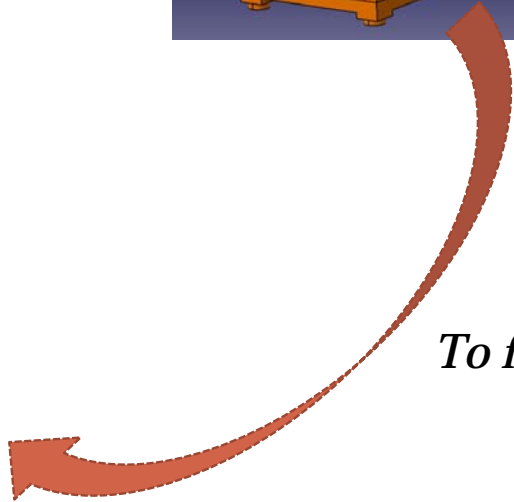
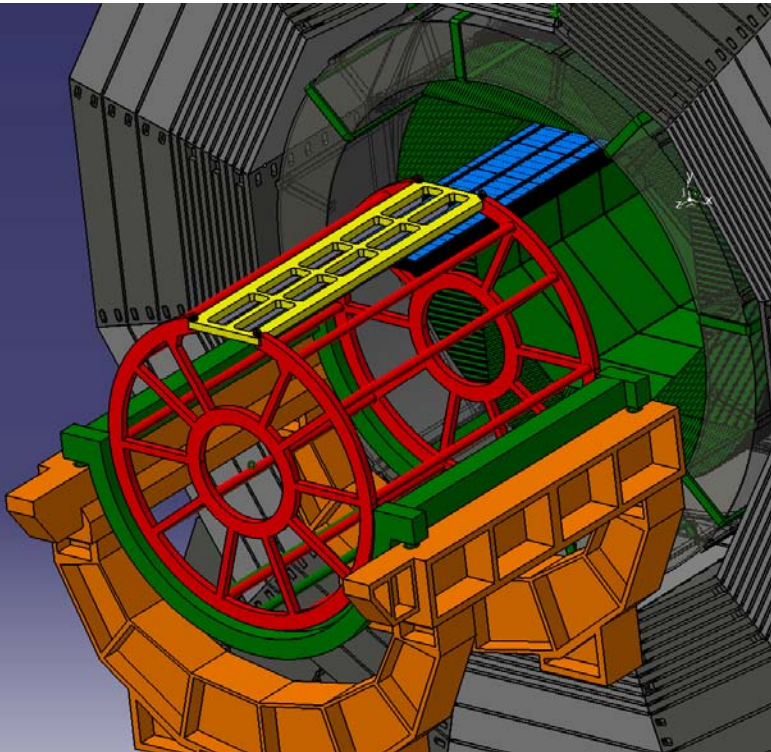
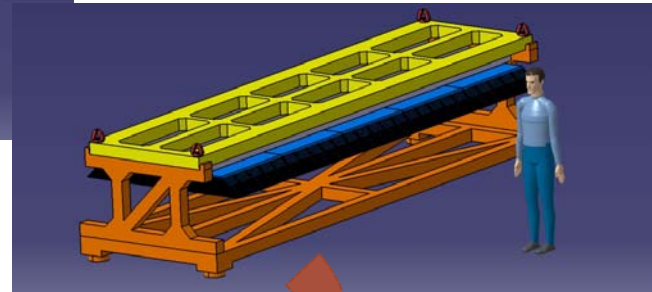
Leak-less water cooling or
Bi-phasic gas like CO_2 : High Pressure (100 bars),

From J.C.Ianigro (IPNL)

Ecal integration scenario (Barrel)



From One stave



To full barrel ...

From M.Anduze(LLR)

24 May 2012

Task	Description / constraint	tooling	FTE	Time
1	Handling of 1 (over 40) module. Weigth :1 t	Scraft, table	2 T	9 days
2	Alignment			
3	Module 1 in position on frame		2T	
4	Stave 1 and its frame on the support structure			
5	Insertion of 375 slabs per stave. 1 Slab =10 to 15 kg alignment within alveola = 500 μm over 1.8 m		2 T	2 weeks
6	Electrical connections up to LDA boards		2 T	
7	Cooling blocks (5) up to Module edge, over LDA up to main distribution line position		2 T	
8	Electrical and cooling distribution lines on top of the stave			
9	Tests (electronic and signal)		2 T part- time	2 month

to be repeated 8 times.

Some parts can be done in parallel

(depends on the available manpower, because not the same qualification) :

Tasks 1-4 of stave (n+1) and Tasks 5 to 9 of the stave (n)

Needed space :

For a stave assembly : 7*4 m²

Per Beam , for storage : 6*3 m²

Stave (fully equipped)+ frame + support

about 12 t

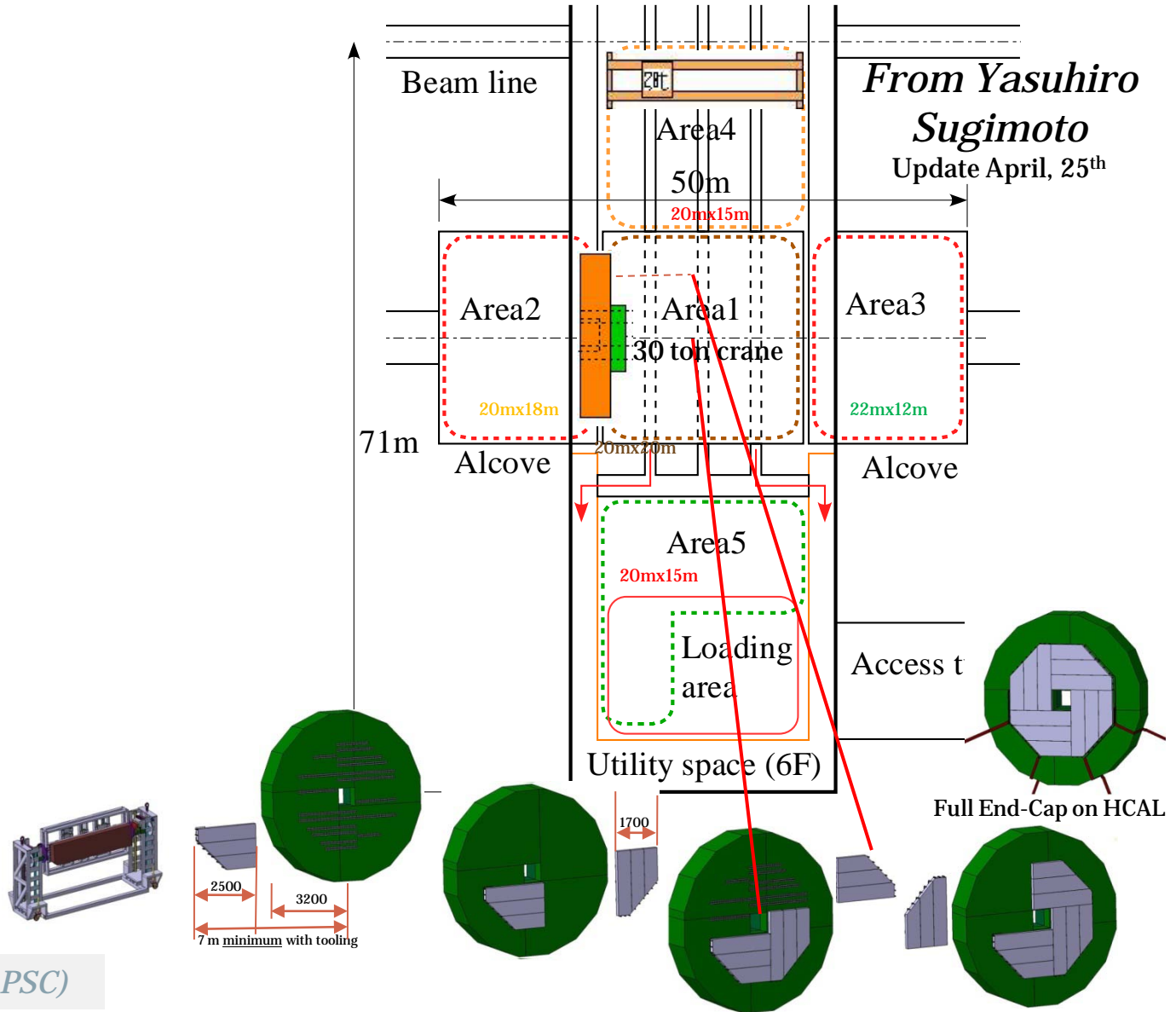
ILD MDI session, Kyushu University

21 May 2012

These studies include estimation of :

- the timescale (assembly hall and experimental area
- Manpower
- Needed space for Assembly, test, storage & integration work
- Tooling

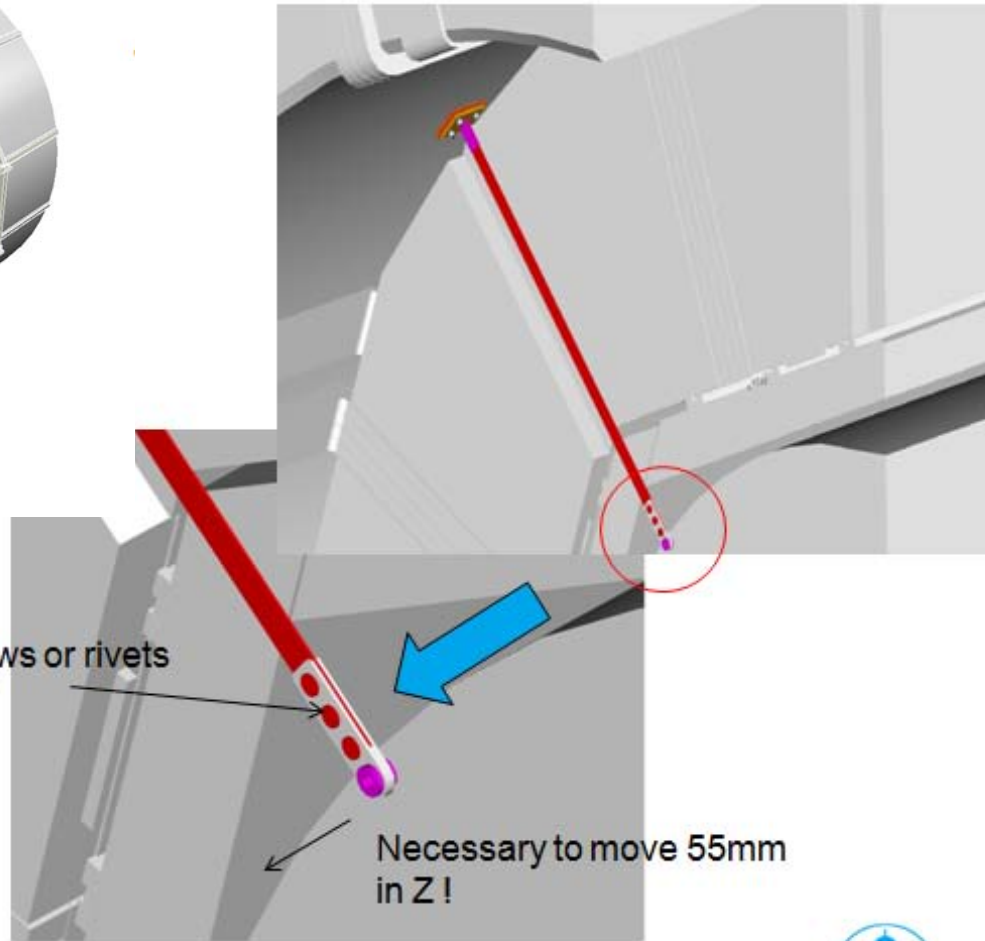
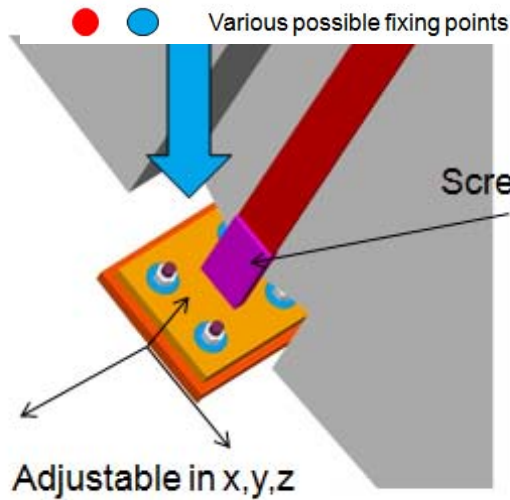
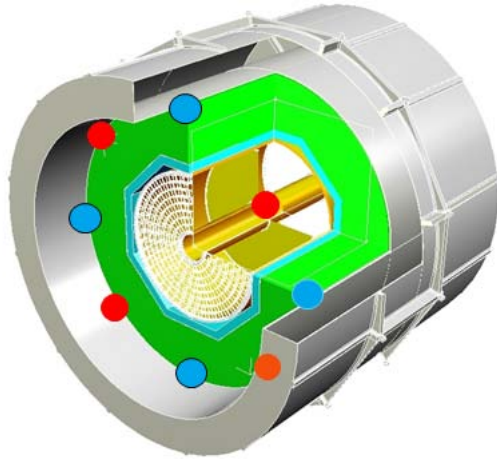
Same studies for the endcaps



From D.Gronidin (LPSC)

4 cantilevers arms or ropes from Coil Cryostat

Detailed design under study (Bar support or Flat Ribbon):



Overview of an first idea of the HV-cable routing

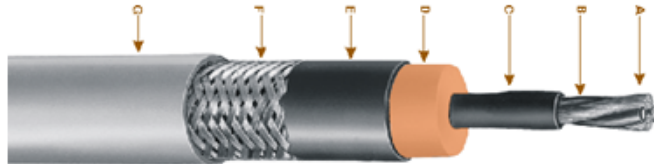
HV Cable and routing

Samples of HV-cables

Okonite Hi-Voltage Cable: www.okonite.com

100kV, od= 16,76mm,

bending radius = $4 * od > 70mm$



- A** Coated Stranded Copper Conductors
- B** Polyester Insulation
- C** Extruded Semiconducting Layer
- D** Primary Insulation – Okoguard
- E** Extruded Insulation Shield
- F** Coated Copper Braid
- G** Jacket – Okoseal

Heinzinger HVC100 Best. No.:00.220.853.9 www.heinzinger.com

100kV, od= 14mm, bending radius min. 280mm!

FUG C 2124, Mat.- No.: 0502032124

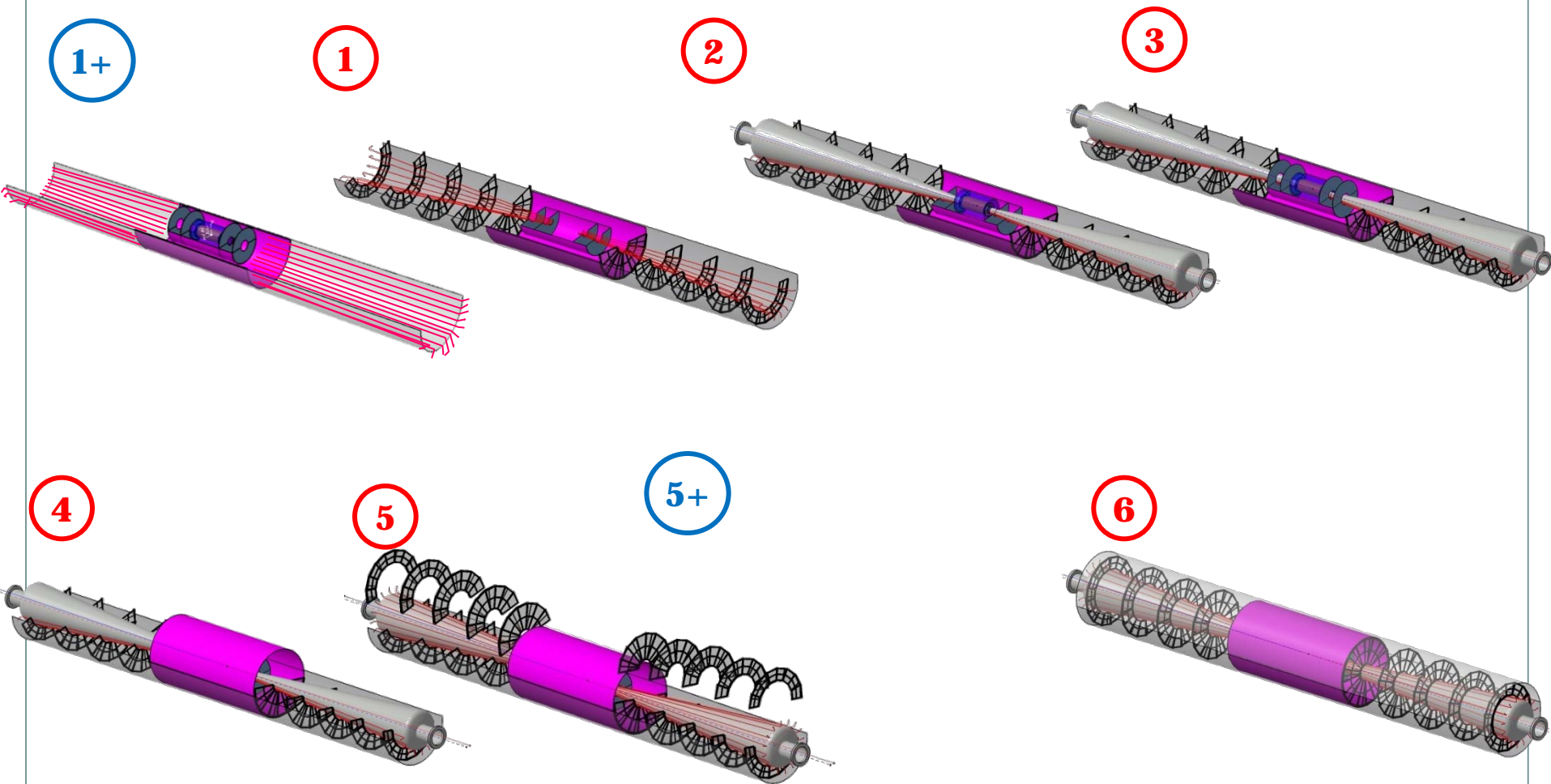
Cross section of the HV-cable:
255-300mm² necessary

http://www.fug-elektronik.de/webdir/PDF/e/Access_data_sheet.pdf

100kV, od= 11,2mm, bending radius min. 152mm



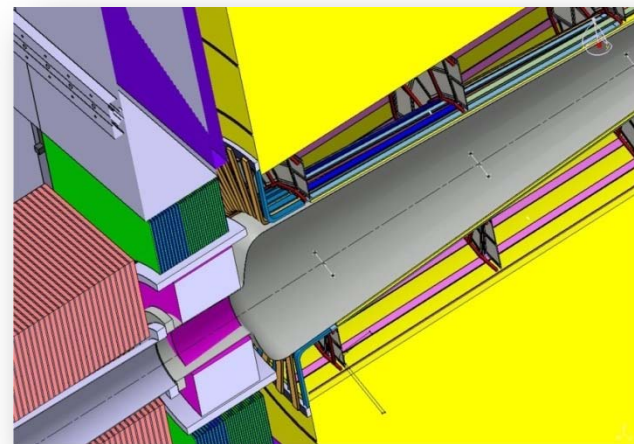
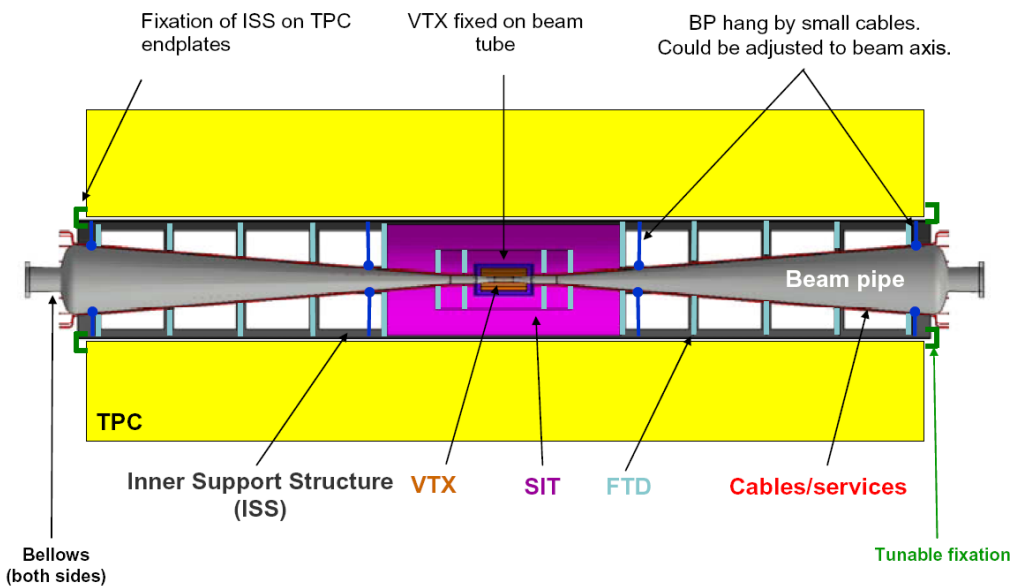
- **Assembly procedure :**



From A. Gonnin & C. Bourgois(LAL)

- Integration procedure :

- Using the same TPC insertion tool, adapt on it an apparatus to support and insert the inner parts.
- Used the TPC center to guide and roll inside,
- During insertion, control the deformation, stress and alignment
- Adjusted with the TPC references
- Fixed on TPC

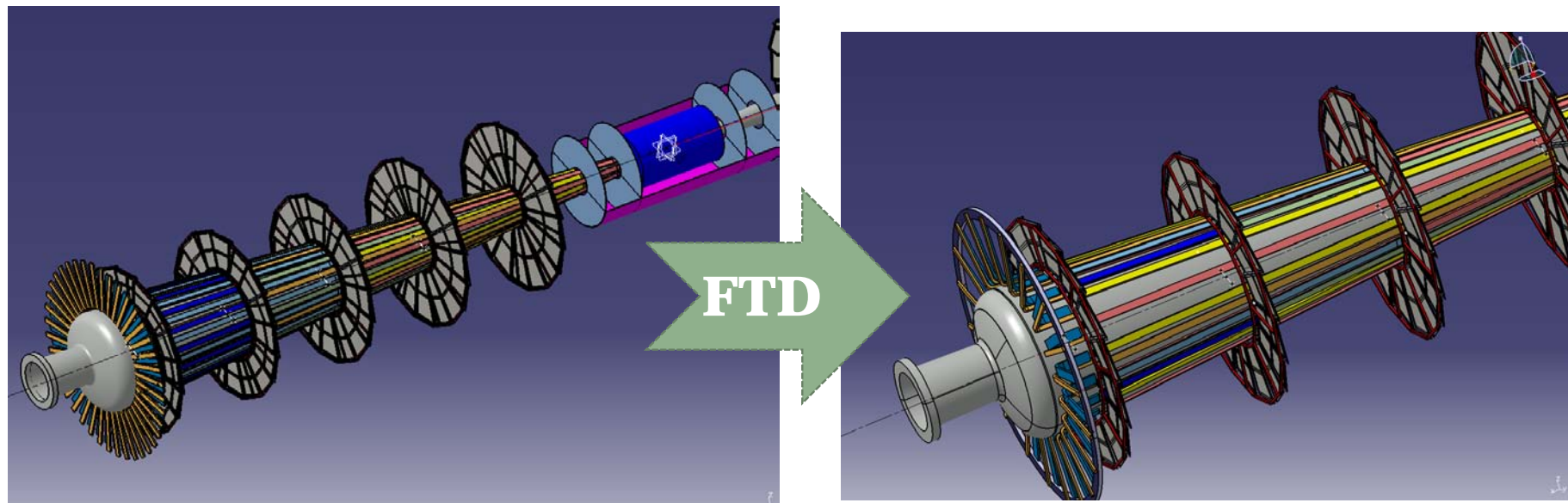


From A. Gonnin & C. Bourgois

Recent optimization work of the power distribution for the FTD3 to 7 :

Thanks to D.Moya, F.Ateche, I.Alvarez, A.Ruiz Jimeno IFCA Cantabria

- *Reduction of the amount of cables per FTD from 98 to 16*
- *Conductor section per FTD from 158 to 14.8 mm² in Cu...*

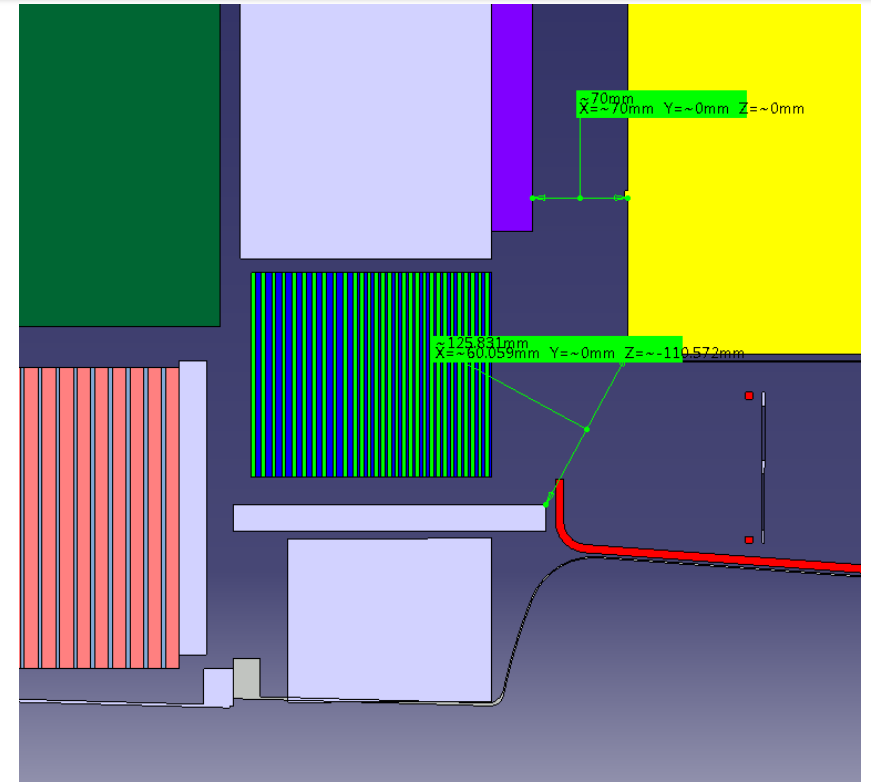
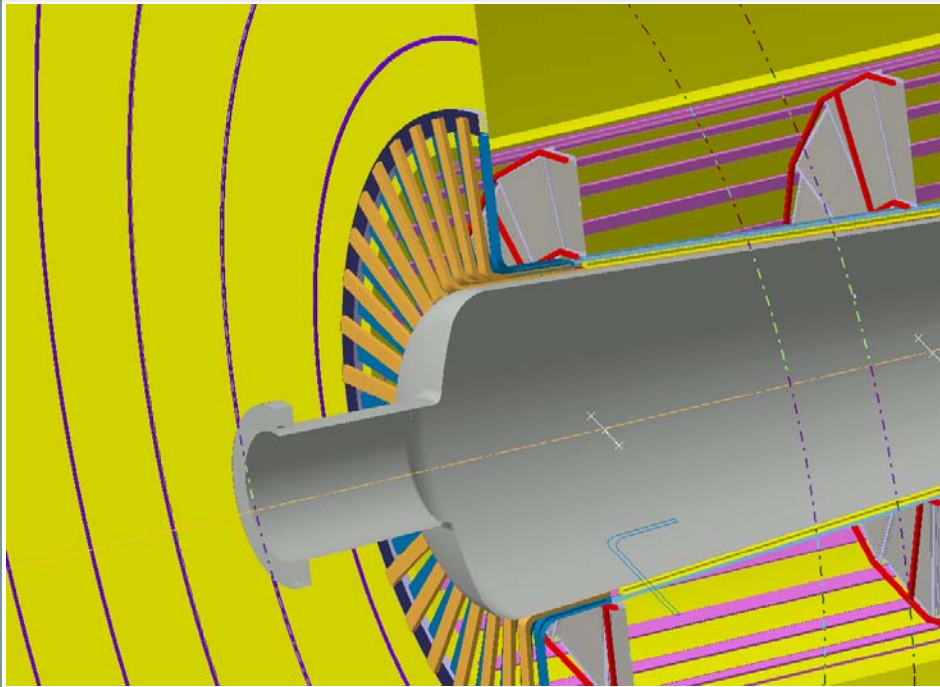


**Amount of Dead material around the beam pipe
(Al as conductor, Security margin 2 because of unknowm subdetectors, material nature...etc...).....**

Aluminum									
	position	FTD7	FTD6	FTD5	FTD4	FTD3	FTD 2	FTD 1	VTX
new values	eq Th of Al	0,85	0,99	1,22	1,54	2,18	2,76	2,00	0,67
	X0% FCPPD	0,95%	1,11%	1,37%	1,73%	2,45%	3,10%	2,25%	0,75%
old values	eq Th of Al	2,16	2,39	2,74	3,20	4,12	3,45	2,00	0,67
	X0% FCPPD	2,43%	2,68%	3,08%	3,60%	4,63%	3,87%	2,25%	0,75%
Nbre of Cables									
	position	FTD7	FTD6	FTD5	FTD4	FTD3	FTD 2	FTD 1	VTX
new values	total	514,00	482,00	450,00	418,00	386,00	266,00	126,00	30,00
	old values	900,00	804,00	708,00	612,00	516,00	288,00	126,00	30,00

This work of optimisation should be done for each detector.....

Patch panels positions : Inner radius of TPC endplates, what for ? :



TPC endplate position , one patch panel needed :

- 540 cables for power + $\times 100$ for signal transmission + Cooling fluid distribution
- Inner part integration steps
- Optical conversion of Signal ???
- Reduction of the power distribution cables ?

Patch panels positions : Inner radius of the coil:

About 2000 cables per side full detector
BARREL

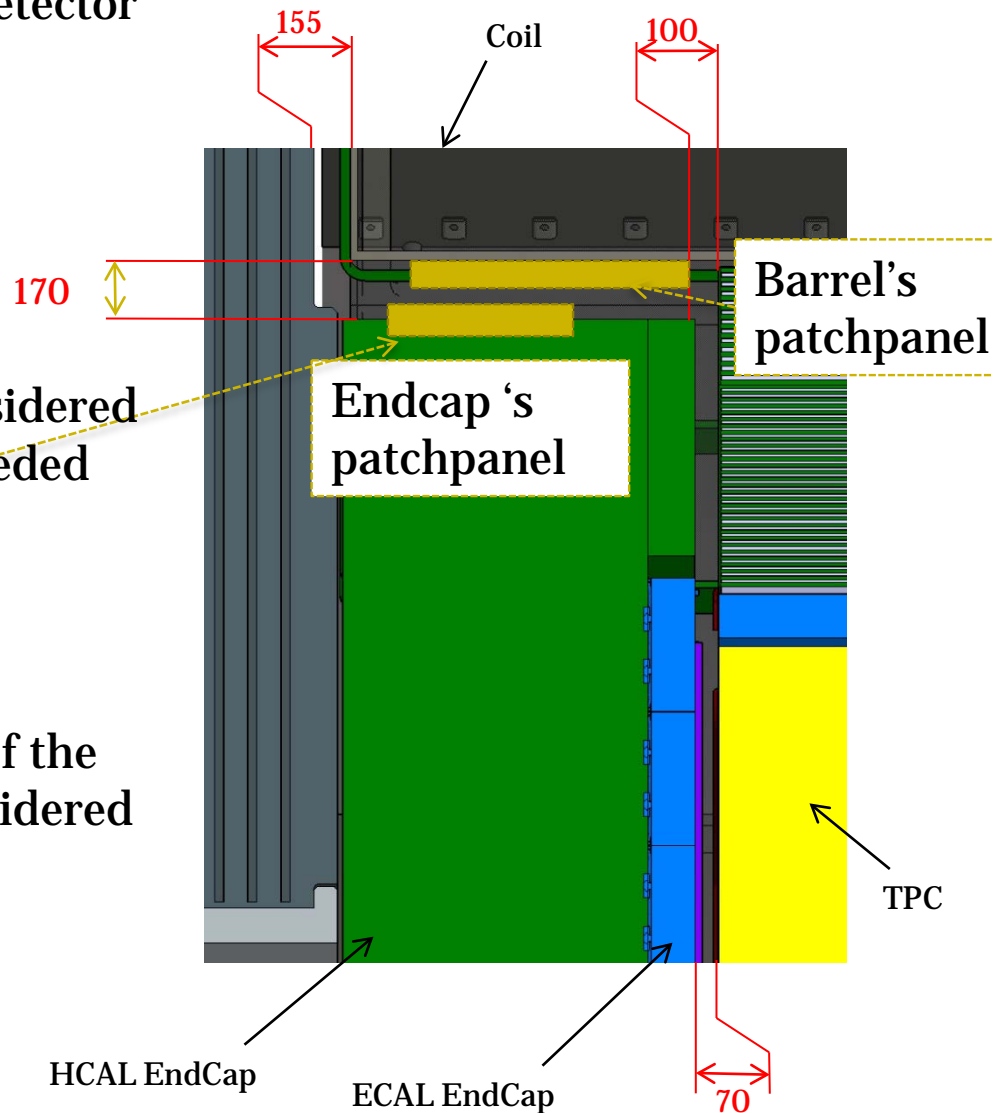
But:

What for the endcaps ?

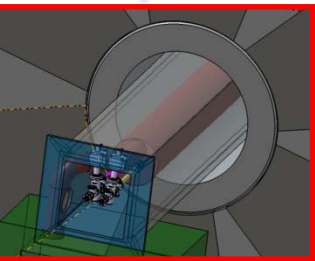
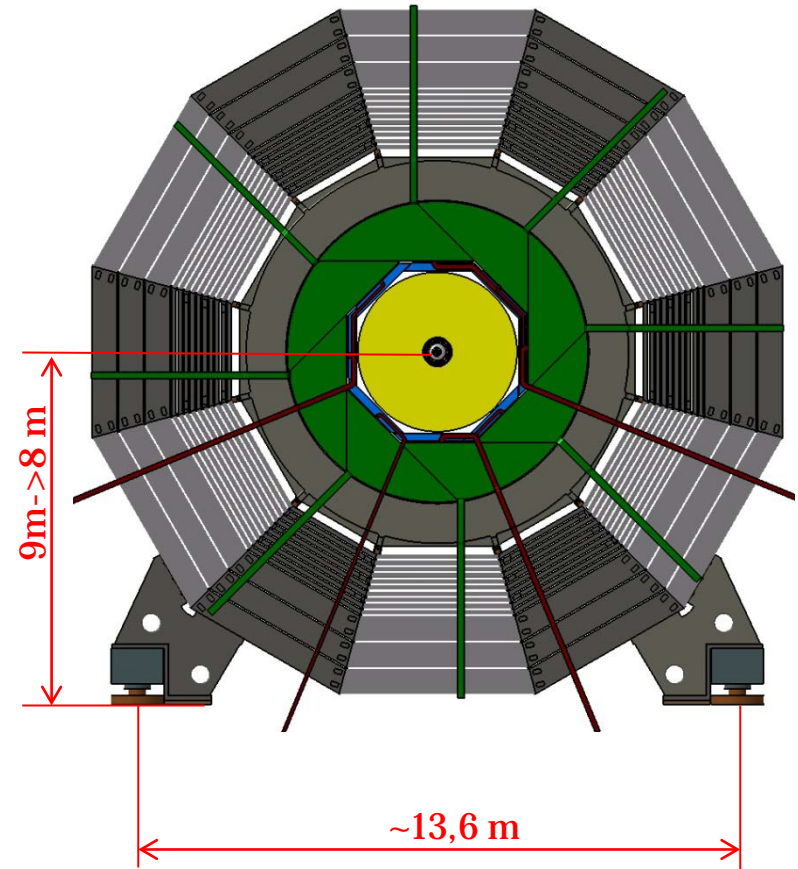
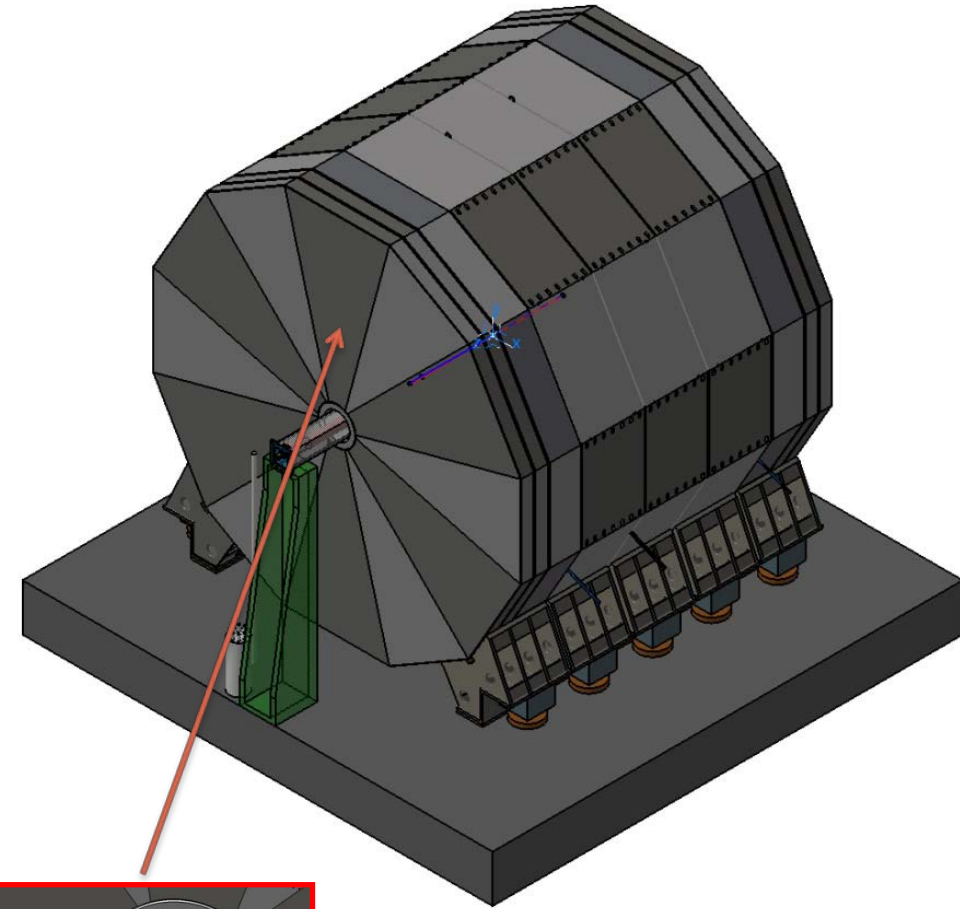
Signal transmission : not yet considered
and may strongly increase the needed
space

Optical conversion

Cooling : the volume occupancy of the
cooling distribution must be considered



Beam height / platform: decrease from 9 to 8 m



Changes in the shape of return Yoke :

- No more central supporting feet
- Shape of the central hole

From A. Gonnin & C. Bourgois (LAL)

Reminder : Roadmap for DBD 2012 :

“Complete basic mechanical integration of the baseline design accounting for insensitive zones such as the beam holes, support structure, cables, gaps or inner detector material”

“Develop a realistic simulation model of the baseline design, including the identified faults and limitations”



Still some discrepancies between overall mechanical model and some subdetectors engineering models :

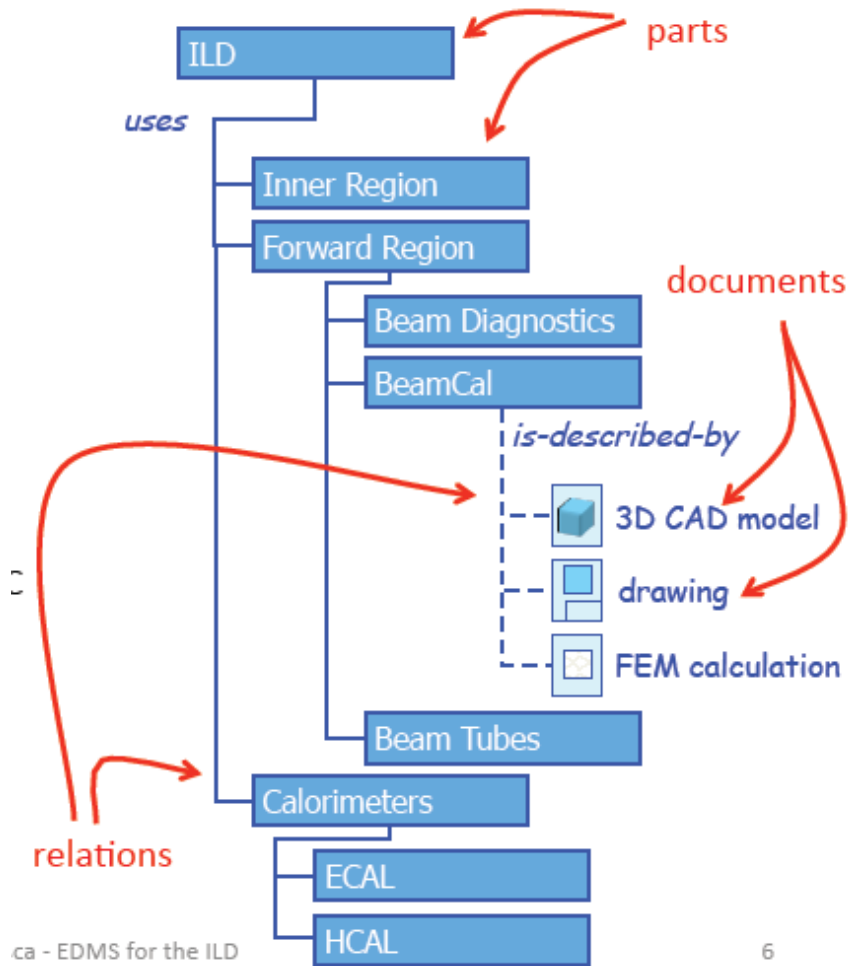
Send updates to A.Gonnin/C.Bourgeois @ LAL

But these models should also be in EDMS (**Contact A.Rosca**):

The idea is to have :

1. an **overall** « not too heavy » **CAD mechanical** model, @LAL
2. **detailed subdetectors models** at each relevant node of the EDMS tree structure

Documents in EDMS



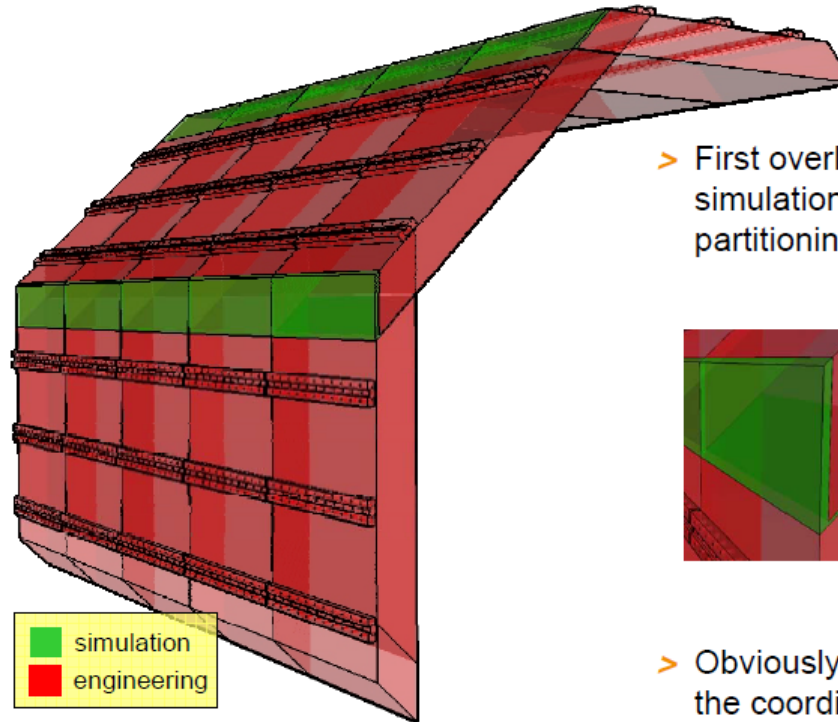
Overall CAD model

Detailed CAD model

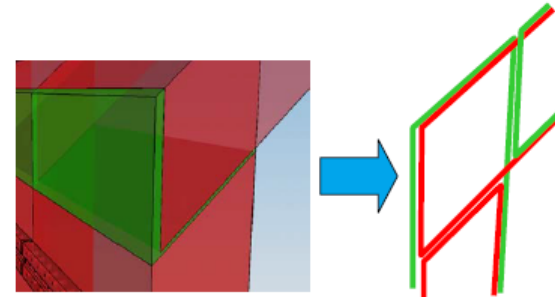
From A.Rosca (DESY)

EDMS might ease the comparison of the Simulation and Mechanical model ...

Comparing Simulation and Detailed Model (1)



- > First overlay of detailed model with simulation model revealed different partitioning of ECal into modules



- > Obviously a problem of orientation of the coordinate system which can be easily remedied (rotate around X)



Conclusion (1)

Huge amount of work on detector integration studies :

First detailed time estimate for ILD assembly : **3,25 Y**

But ...The more work is done... the more questions arise.....

Detectors :

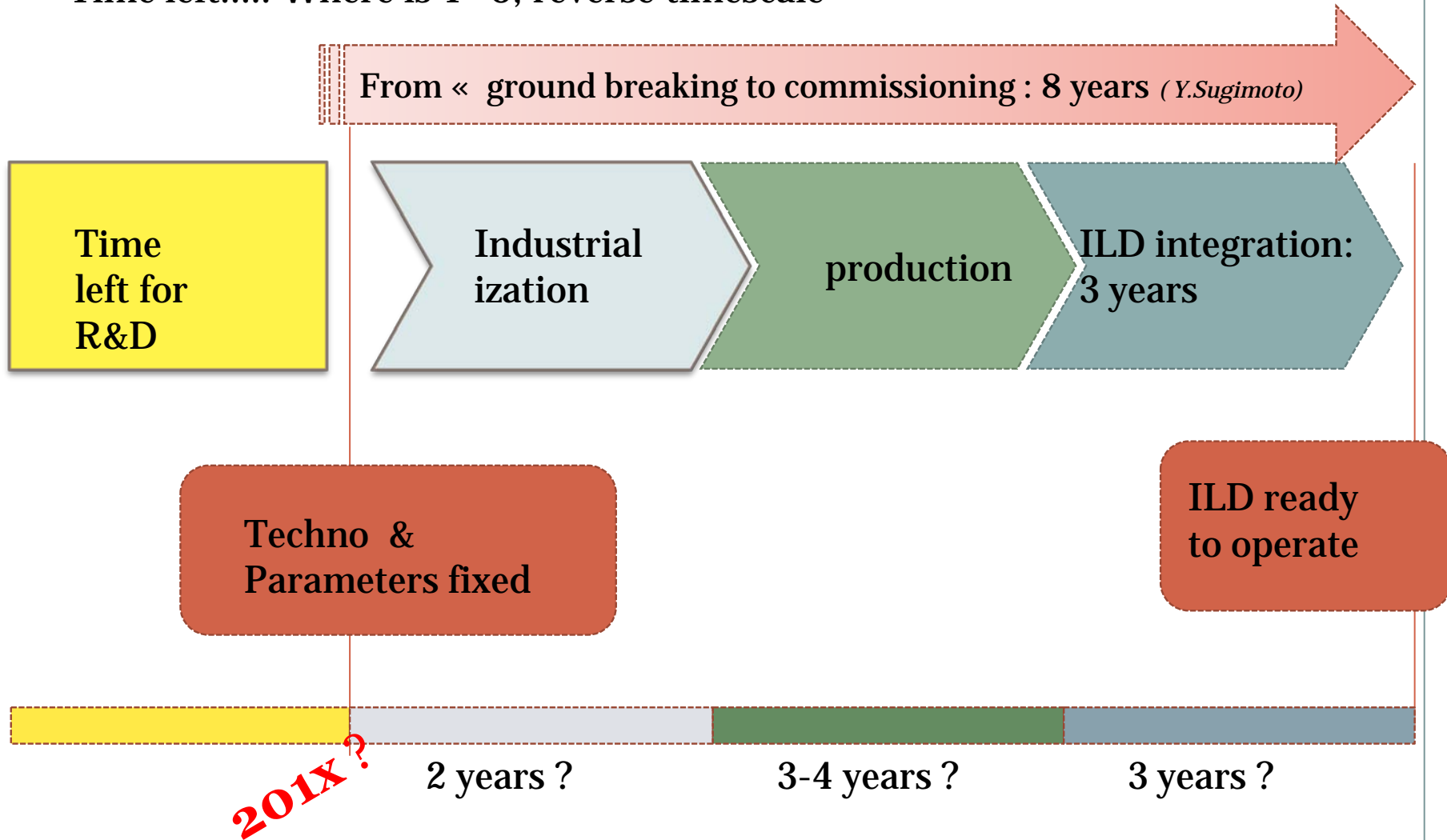
- Services : not only power distribution.... Signal + Cooling
- Update the inputs for both Barrel, Endcaps & forward region, SET, ETD...
- Alignment & calibration issues
- Pooling the distribution ?

Integration :

- Optimization & Parallelization of the integration sequences for both sites constraints (mountain/non mountain)
- Sharing tooling for integration,
- Estimate the place for storage, tooling assembly...etc...

Conclusion (2)

Time left..... Where is T=0, reverse timescale

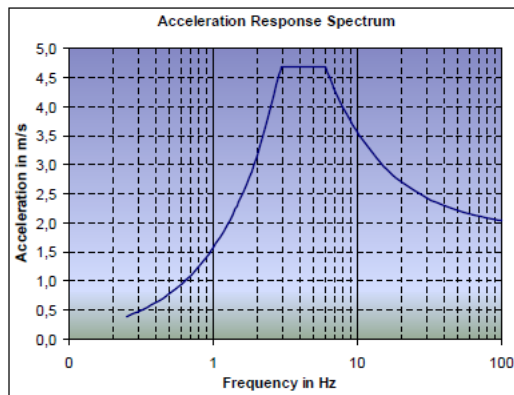


Spare

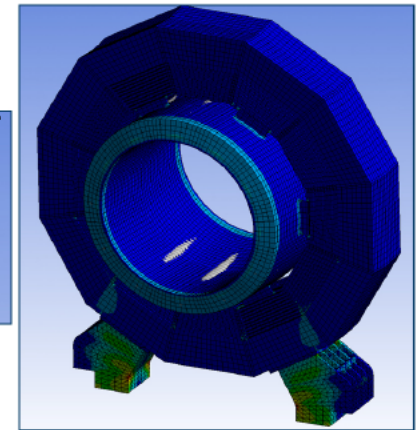
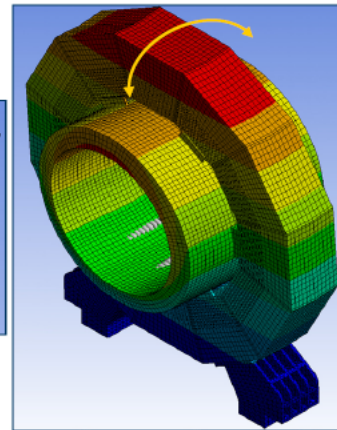
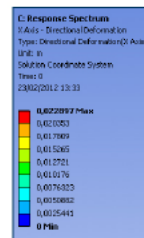
Seismic Hazard for Japanese site consideration, coil & RY behaviour

Parameters for acceleration spectrum

- Peak Ground Acceleration: 1.5 m/s²
- Damping ratio for steel structure: 2%
- Soil type: hard soil



- ❖ With the acceleration response spectrum applied along the detector axis, the fundamental mode of the structure dominates: back and forth motion of the yoke ring
- ❖ The max displacement is around 23mm, which is quite high
- ❖ The peak stress is located in the feet. The level seems acceptable but the results need to be checked with a proper design and model
- ❖ Attaching the 3 rings together is probably the way to go to increase the overall stiffness and reduce the peak displacement



From O.Ferreira (LLR)