

# RF FUNDAMENTALS MICROPHONICS

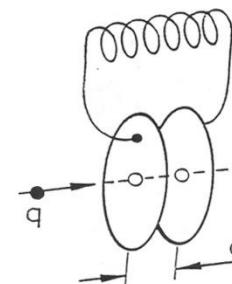
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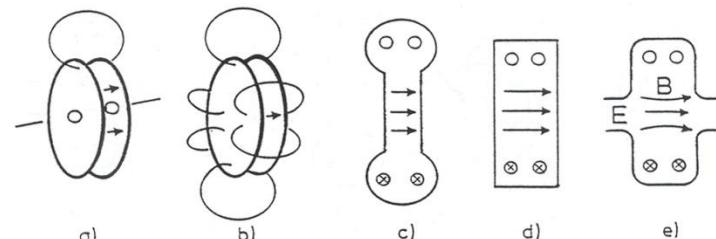
# Equivalent Circuit for an rf Cavity

Simple LC circuit representing an accelerating resonator



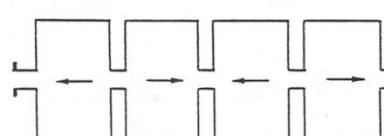
Simple lumped L-C circuit representing an accelerating resonator.  
 $\omega_0^2 = 1/LC$

Metamorphosis of the LC circuit into an accelerating cavity

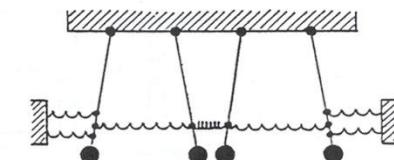


Metamorphosis of the L-C circuit of Fig. 1 into an accelerating cavity (after R.P.Feynman<sup>33</sup>).  
Fig. 5d shows the cylindrical "pillbox cavity" and Fig. 5e a slightly modified pillbox cavity with beam holes (typical  $\beta$  between 0.5 and 1.0). Fig. 5c resembles a low  $\beta$  version of the pillbox variety ( $0.2 < \beta < 0.5$ ).

Chain of weakly coupled pillbox cavities representing an accelerating cavity



Chain of coupled pendula as its mechanical analogue



Chain of weakly-coupled pillbox cavities representing an accelerating module

Chain of coupled pendula as a mechanical analogue to Fig. 6a

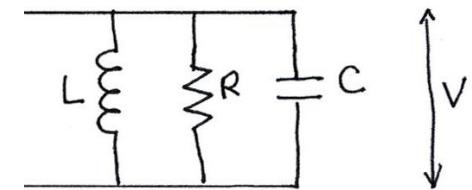
# Parallel Circuit Model of an Electromagnetic Mode

- Power dissipated in resistor R:  $P_{diss} = \frac{1}{2} \frac{V_c^2}{R}$
- Shunt impedance:  $R_{sh} \equiv \frac{V_c^2}{P_{diss}}$   $\Rightarrow R_{sh} = 2R$
- Quality factor of resonator:

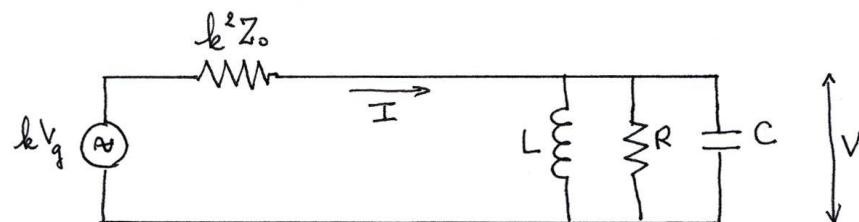
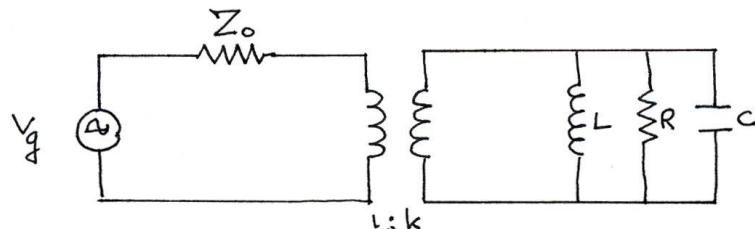
$$Q_0 \equiv \frac{\omega_0 U}{P_{diss}} = \omega_0 CR = \frac{R}{L\omega_c} = R \left( \frac{C}{L} \right)^{1/2}$$

$$\tilde{Z} = R \left[ 1 + i Q_0 \left( \frac{\omega}{\omega_0} - \frac{\omega_0}{\omega} \right) \right]^{-1}$$

$$\omega \approx \omega_0, \quad \tilde{Z} \approx R \left[ 1 + 2i Q_0 \left( \frac{\omega - \omega_0}{\omega_0} \right) \right]^{-1}$$



# 1-Port System



Total impedance:  $k^2 Z_0 + \frac{R}{1 + 2i \frac{Q_0}{\omega_0} \Delta\omega}$

$$I = \frac{kV_g}{k^2 Z_0 + \frac{R}{1 + 2i \frac{Q_0}{\omega_0} \Delta\omega}}$$

$$V = kV_g \frac{R}{R + k^2 Z_0 \left( 1 + 2i \frac{Q_0}{\omega_0} \Delta\omega \right)}$$

# 1-Port System

Energy content  $U = \frac{1}{2} CV^2 = \frac{1}{2} \frac{Q_0}{\omega R} V^2$

$$= \frac{1}{2} \frac{Q_0}{\omega R} k^2 V_g^2 \frac{R^2}{(R + k^2 Z_0)^2 + 4k^4 Z_0^2 Q_0^2 \left(\frac{\Delta\omega}{\omega_0}\right)^2}$$

Incident power:  $P_{inc} = \frac{V_g^2}{8Z_0}$

Define coupling coefficient:  $\beta = \frac{R}{k_0^2 Z_0}$

$$\frac{U}{P_{inc}} = \frac{Q_0}{\omega_0} \frac{4\beta}{(1+\beta)^2} \frac{1}{1 + \left(\frac{2Q_0}{1+\beta}\right)^2 \left(\frac{\Delta\omega}{\omega_0}\right)^2}$$

# 1-Port System

Power dissipated

$$P_{diss} = \frac{\omega U}{Q_0} = P_{inc} \frac{4\beta}{(1+\beta)^2} \frac{1}{1 + \left(\frac{2Q_0}{1+\beta}\right)^2 \left(\frac{\Delta\omega}{\omega_0}\right)^2}$$

Optimal coupling:

$$\frac{U}{P_{inc}} \quad \text{maximum} \quad \text{or} \quad P_{diss} = P_{inc}$$

$$\Rightarrow \Delta\omega = 0, \quad \beta = 1 \quad \text{: critical coupling}$$

Reflected power

$$P_{ref} = P_{inc} - P_{diss} = P_{mc} \left[ 1 - \frac{4\beta}{(1+\beta)^2} \frac{1}{1 + \left(\frac{2Q_0}{1+\beta} \frac{\Delta\omega}{\omega_0}\right)^2} \right]$$

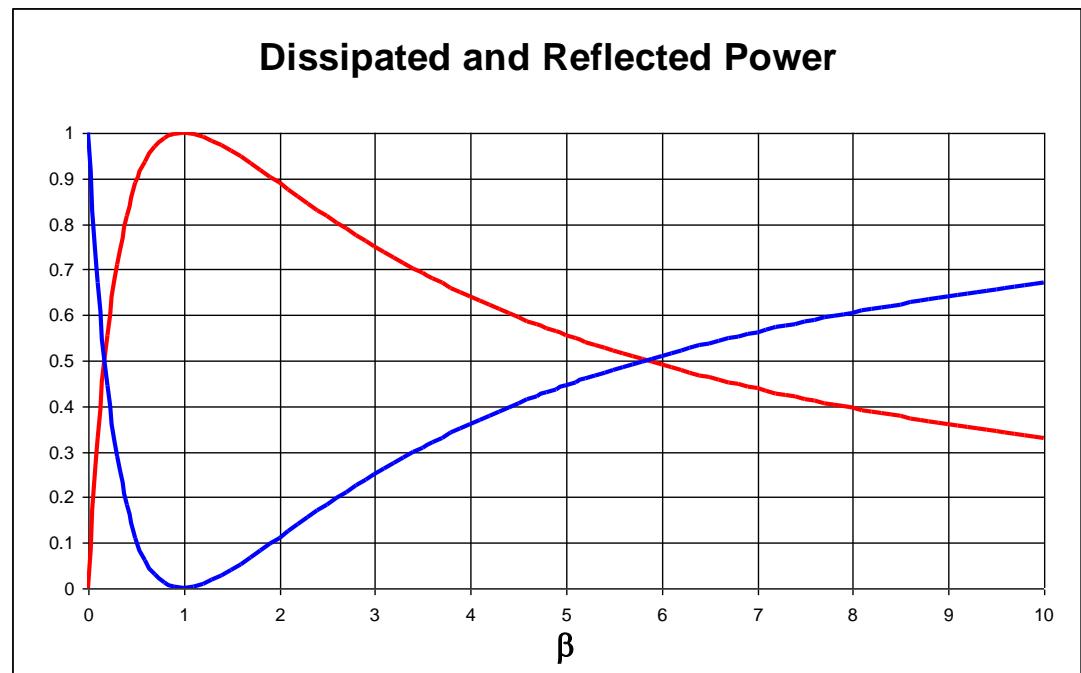
# 1-Port System

At resonance

$$U = \frac{Q_0}{\omega_0} \frac{4\beta}{(1+\beta)^2} P_{inc}$$

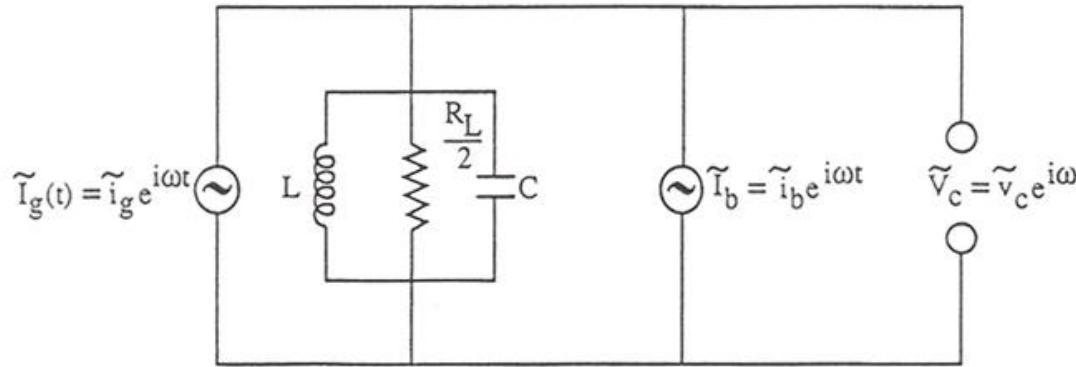
$$P_{diss} = \frac{4\beta}{(1+\beta)^2} P_{inc}$$

$$P_{ref} = \left( \frac{1-\beta}{1+\beta} \right)^2 P_{inc}$$



# Equivalent Circuit for a Cavity with Beam

- Beam in the rf cavity is represented by a current generator.
- Equivalent circuit:



$$R_L = \frac{R_{sh}}{(1 + \beta)}$$

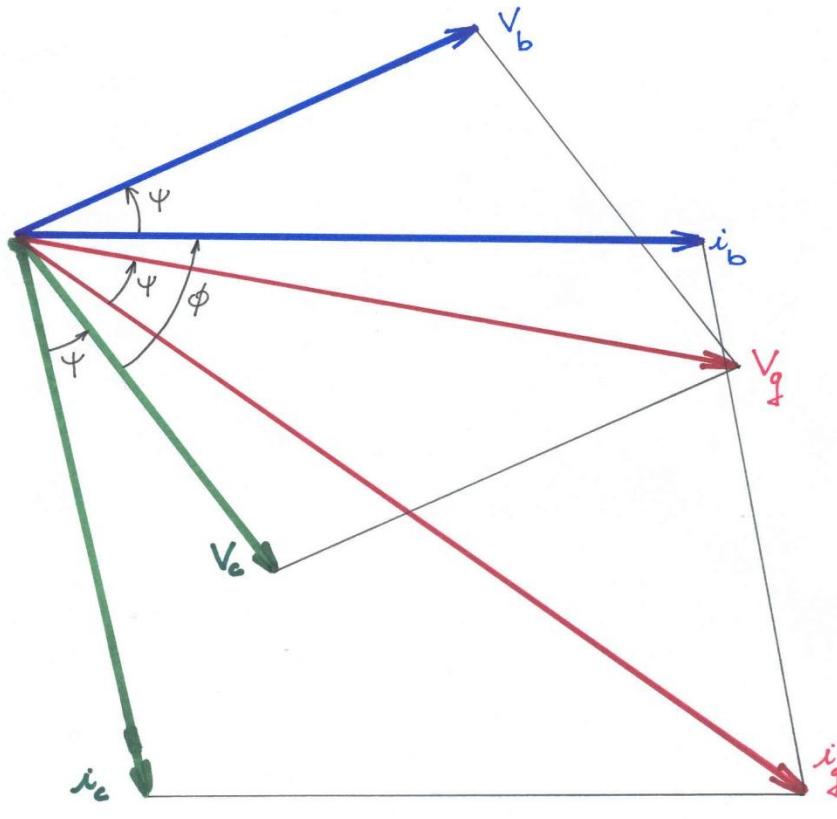
$\tilde{i}_b$  produces  $\tilde{V}_b$  with phase  $\psi$  (detuning angle)

$\tilde{i}_g$  produces  $\tilde{V}_g$  with phase  $\psi$

$$\tilde{V}_c = \tilde{V}_g - \tilde{V}_b$$

$$\tan \psi = -2 \frac{Q_0}{1 + \beta} \frac{\Delta\omega}{\omega_0}$$

# Equivalent Circuit for a Cavity with Beam



$$V_g = (P_g R_{sh})^{1/2} \frac{2\beta^{1/2}}{1+\beta} \cos\psi$$

$$V_b = \frac{i_b R_{sh}}{2(1+\beta)} \cos\psi$$

$$i_b = 2i_0 \frac{\sin \frac{\theta_b}{2}}{\frac{\theta_b}{2}}$$

$i_b$ : beam rf current

$i_0$ : beam dc current

$\theta_b$ : beam bunch length

# Equivalent Circuit for a Cavity with Beam

$$P_g = \frac{V_c^2}{R_{sh}} \frac{1}{4\beta} \left\{ (1 + \beta + b)^2 + [(1 + \beta) \tan \psi - b \tan \phi]^2 \right\}$$

$$b = \frac{\text{Power absorbed by the beam}}{\text{Power dissipated in the cavity}} = \frac{R_{sh} i_0 \cos \phi}{V_c}$$

$$(1 + \beta_{opt}) \tan \psi_{opt} = b \tan \phi$$

**Minimize  $P_g$ :**

$$\beta_{opt} = |1 + b|$$

$$P_g^{opt} = \frac{V_c^2}{R_{sh}} \frac{|1 + b| + (1 + b)}{2}$$

# Frequency Control

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**Energy gain**

$$W = qV \cos \phi$$

**Energy gain error**

$$\frac{\delta W}{W} = \frac{\delta V}{V} - \delta\phi \tan \phi$$

**The fluctuations in cavity field amplitude and phase come mostly from the fluctuations in cavity frequency**

**Need for fast frequency control**

**Minimization of rf power requires matching of average cavity frequency to reference frequency**

**Need for slow frequency tuners**

# Some Definitions

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- **Ponderomotive effects:** changes in frequency caused by the electromagnetic field (radiation pressure)
  - Static Lorentz detuning (cw operation)
  - Dynamic Lorentz detuning (pulsed operation)
- **Microphonics:** changes in frequency caused by connections to the external world
  - Vibrations
  - Pressure fluctuations

**Note:** The two are not completely independent.

When phase and amplitude feedbacks are active, ponderomotive effects can change the response to external disturbances

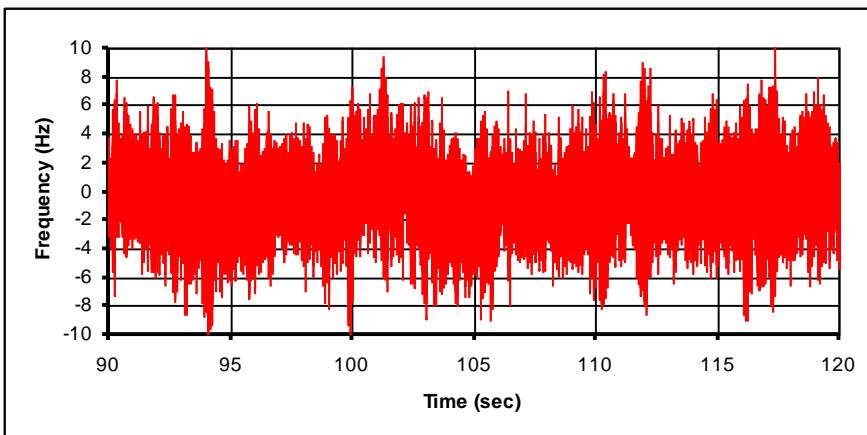
# Cavity with Beam and Microphonics

- The detuning is now

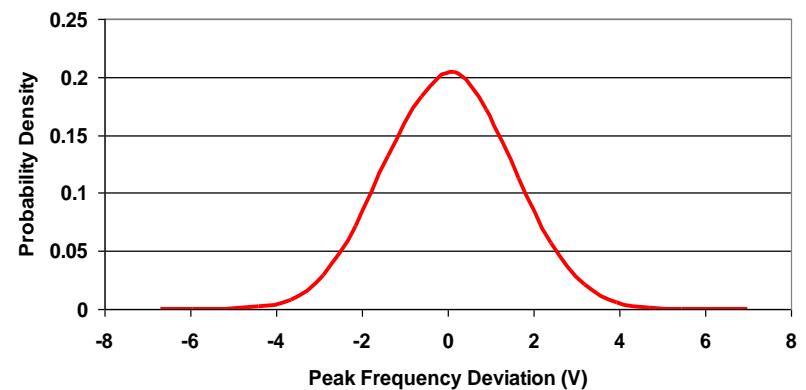
$$\tan \psi = -2Q_L \frac{\delta\omega_0 \pm \delta\omega_m}{\omega_0}$$

$$\tan \psi_0 = -2Q_L \frac{\delta\omega_0}{\omega_0}$$

where  $\delta\omega_0$  is the static detuning (controllable)  
and  $\delta\omega_m$  is the random dynamic detuning (uncontrollable)



Probability Density  
Medium β CM Prototype, Cavity #2, CW @ 6MV/m  
400000 samples



# $Q_{ext}$ Optimization with Microphonics

Condition for optimum coupling:

$$\beta_{opt} = \sqrt{(b+1)^2 + \left(2Q_0 \frac{\delta\omega_m}{\omega_0}\right)^2}$$

and

$$P_g^{opt} = \frac{V_c^2}{2R_{sh}} \left[ (b+1) + \sqrt{(b+1)^2 + \left(2Q_0 \frac{\delta\omega_m}{\omega_0}\right)^2} \right]$$

In the absence of beam ( $b=0$ ):

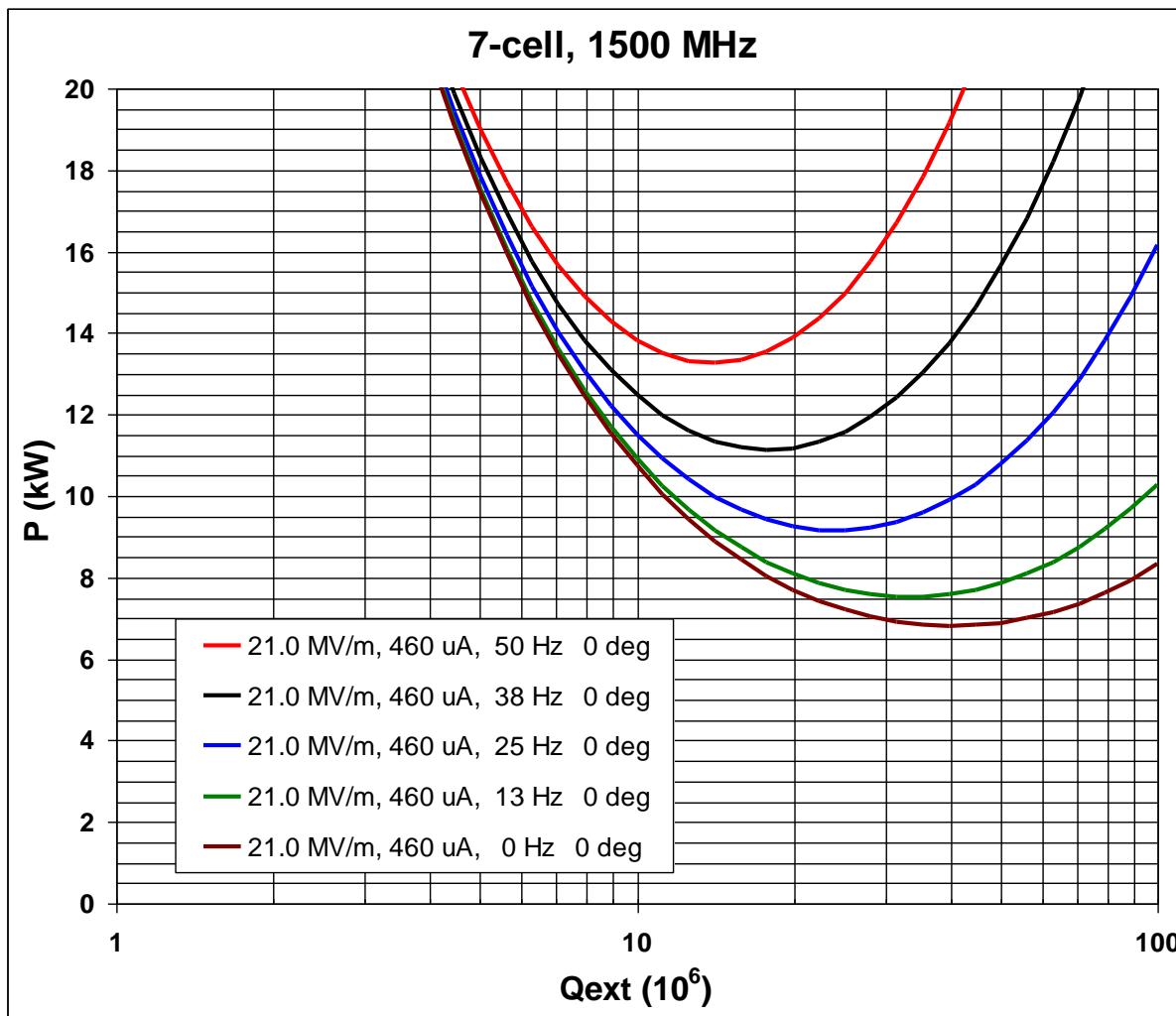
$$\beta_{opt} = \sqrt{1 + \left(2Q_0 \frac{\delta\omega_m}{\omega_0}\right)^2}$$

and

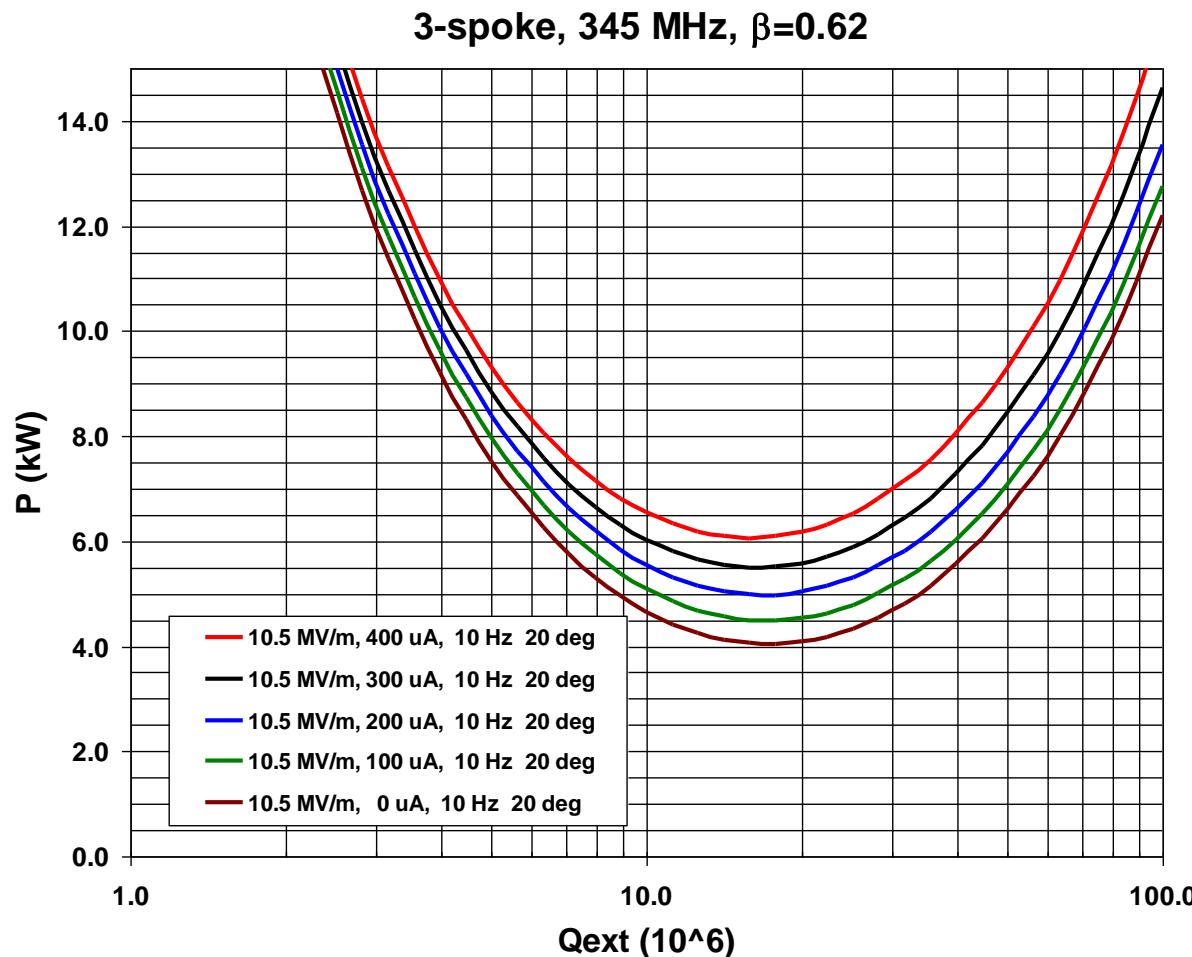
$$P_g^{opt} = \frac{V_c^2}{2R_{sh}} \left[ 1 + \sqrt{1 + \left(2Q_0 \frac{\delta\omega_m}{\omega_0}\right)^2} \right]$$

$= U \delta\omega_m$  If  $\delta\omega_m$  is very large

# Example



# Example

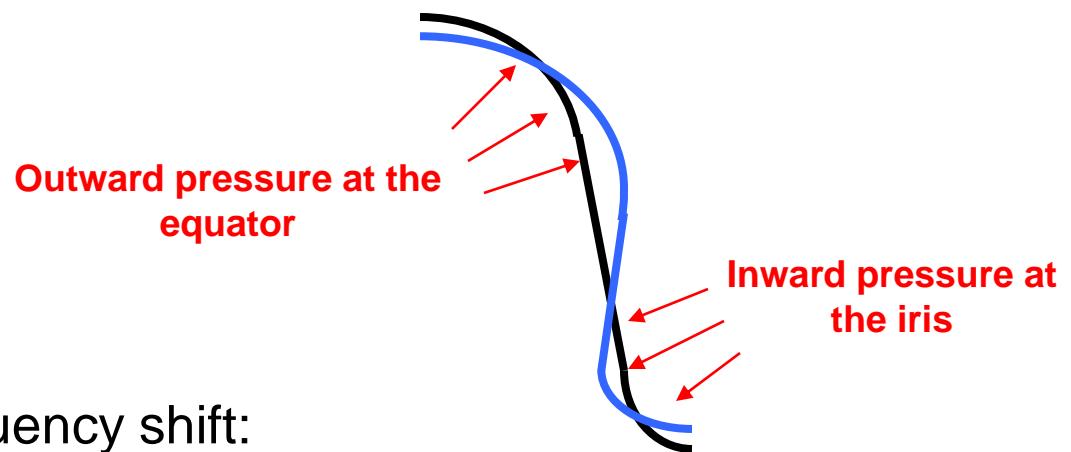


# Lorentz Detuning

**Pressure deforms the cavity wall:**

RF power produces radiation pressure:

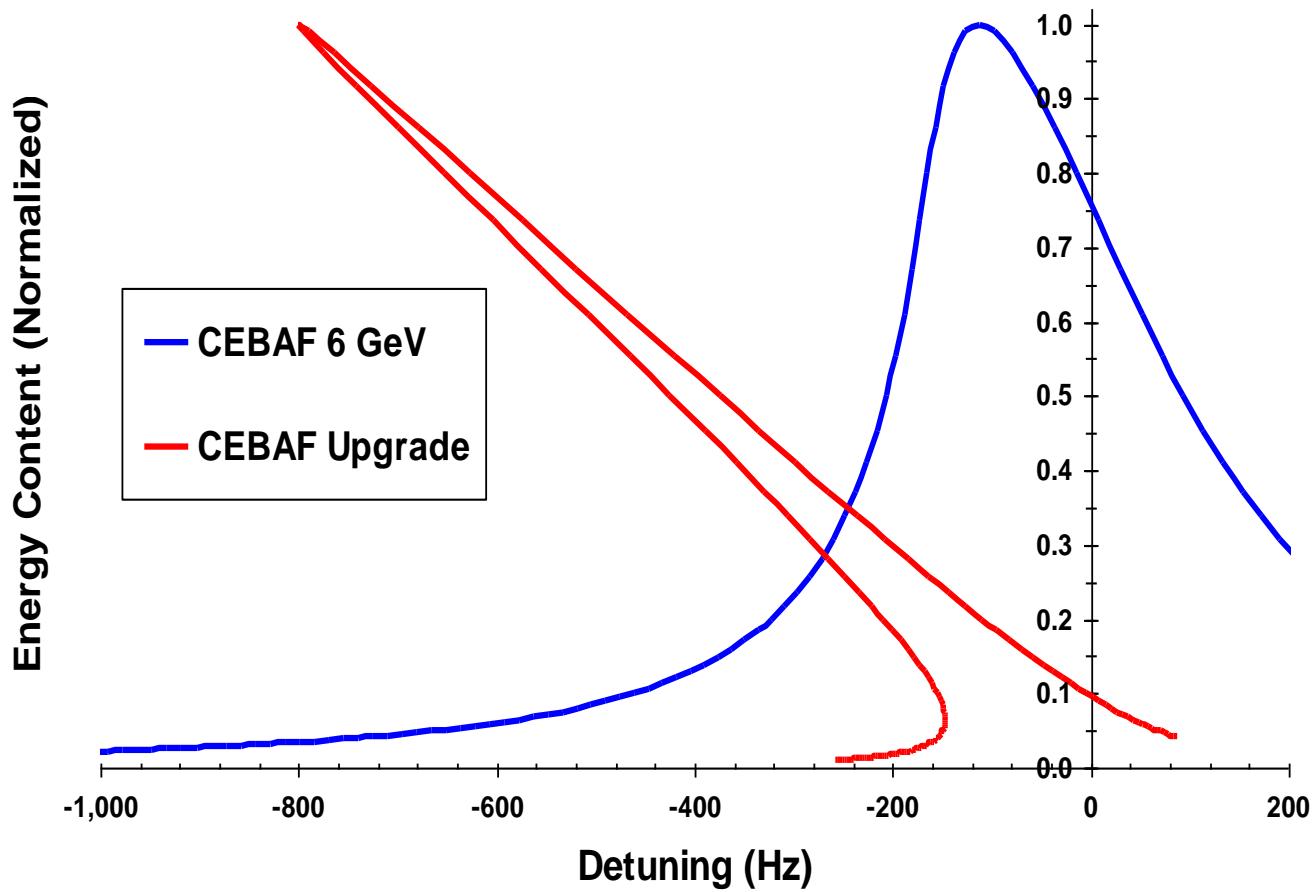
$$P = \frac{\mu_0 H^2 - \varepsilon_0 E^2}{4}$$



Deformation produces a frequency shift:

$$\Delta f = -k_L E_{acc}^2$$

# Lorentz Detuning



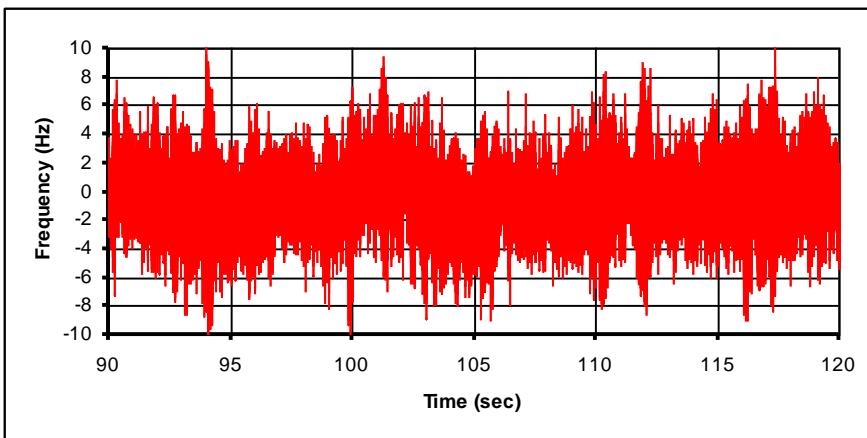
# Micromphonics

- Total detuning

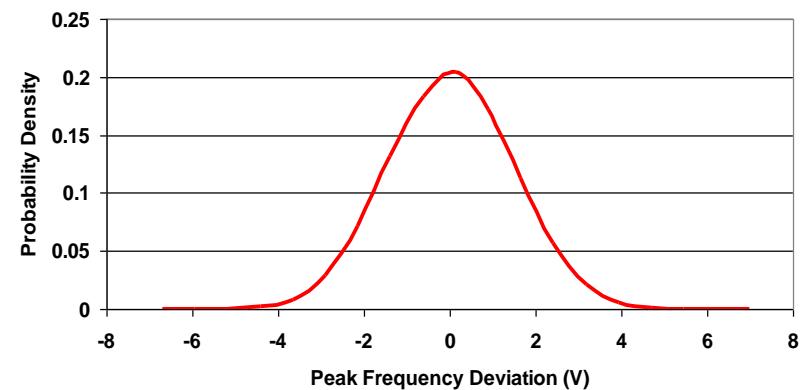
$$\delta\omega_0 + \delta\omega_m$$

where  $\delta\omega_0$  is the static detuning (controllable)

and  $\delta\omega_m$  is the random dynamic detuning (uncontrollable)



Probability Density  
Medium β CM Prototype, Cavity #2, CW @ 6MV/m  
400000 samples



# Ponderomotive Effects

- Adiabatic theorem applied to harmonic oscillators (Boltzmann-Ehrenfest)

If  $\varepsilon = \frac{1}{\omega^2} \frac{d\omega}{dt} \ll 1$ , then  $\frac{U}{\omega}$  is an adiabatic invariant to all orders

$$\Delta \left( \frac{U}{\omega} \right) / \left( \frac{U}{\omega} \right) \sim o(e^{-d/\varepsilon}) \quad \Rightarrow \quad \boxed{\frac{\Delta\omega}{\omega} = \frac{\Delta U}{U}} \quad (\text{Slater})$$

Quantum mechanical picture: the number of photons is constant:  $U = N\hbar\omega$

$$U = \int_V dV \left[ \frac{\mu_0}{4} H^2(\vec{r}) + \frac{\varepsilon_0}{4} E^2(\vec{r}) \right] \text{ (energy content)}$$

$$\Delta U = - \int_S dS \vec{n}(\vec{r}) \cdot \vec{\xi}(\vec{r}) \left[ \frac{\mu_0}{4} H^2(\vec{r}) - \frac{\varepsilon_0}{4} E^2(\vec{r}) \right] \text{ (work done by radiation pressure)}$$

# Ponderomotive Effects

$$\frac{\Delta\omega}{\omega} = - \frac{\int_S dS \vec{n}(\vec{r}) \cdot \vec{\xi}(\vec{r}) \left[ \frac{\mu_0}{4} H^2(\vec{r}) - \frac{\epsilon_0}{4} E^2(\vec{r}) \right]}{\int_V dv \left[ \frac{\mu_0}{4} H^2(\vec{r}) + \frac{\epsilon_0}{4} E^2(\vec{r}) \right]}$$

Expand wall displacements and forces in normal modes of vibration  $\phi_\mu(\vec{r})$  of the resonator

$$\int_S dS \phi_\mu(\vec{r}) \phi_\nu(\vec{r}) = \delta_{\mu\nu}$$

$$\xi(\vec{r}) = \sum_\mu q_\mu \phi_\mu(\vec{r})$$

$$q_\mu = \int_S \xi(\vec{r}) \phi_\mu(\vec{r}) dS$$

$$F(\vec{r}) = \sum_\mu F_\mu \phi_\mu(\vec{r})$$

$$F_\mu = \int_S F(\vec{r}) \phi_\mu(\vec{r}) dS$$

# Ponderomotive Effects

Equation of motion of mechanical mode  $\mu$

$$\frac{d}{dt} \frac{\partial L}{\partial \dot{q}_\mu} - \frac{\partial L}{\partial q_\mu} + \frac{\partial \Phi}{\partial \dot{q}_\mu} = F_\mu \quad L = T - U \quad (\text{Euler-Lagrange})$$

$$U = \frac{1}{2} \sum_\mu c_\mu q_\mu^2 \quad (\text{elastic potential energy}) \quad c_\mu: \text{elastic constant}$$

$$T = \frac{1}{2} \sum_\mu c_\mu \frac{\dot{q}_\mu^2}{\Omega_\mu^2} \quad (\text{kinetic energy}) \quad \Omega_\mu: \text{frequency}$$

$$\Phi = \sum_\mu \frac{c_\mu}{\tau_\mu} \frac{\dot{q}_\mu^2}{\Omega_\mu^2} \quad (\text{power loss}) \quad \tau_\mu: \text{decay time}$$

$$\boxed{\ddot{q}_\mu + \frac{2}{\tau_\mu} \dot{q}_\mu + \Omega_\mu^2 q_\mu = \frac{\Omega_\mu^2}{c_\mu} F_\mu}$$

# Ponderomotive Effects

The frequency shift  $\Delta\omega_\mu$  caused by the mechanical mode  $\mu$  is proportional to  $q_\mu$

$$\Delta\ddot{\omega}_\mu + \frac{2}{\tau_\mu} \Delta\dot{\omega}_\mu + \Omega_\mu^2 \Delta\omega_\mu = -\frac{\omega_0}{c_\mu} \left( \frac{F_\mu}{U} \right)^2 \Omega_\mu^2 U = -k_\mu \Omega_\mu^2 V^2$$

Total frequency shift:  $\Delta\omega(t) = \sum_\mu \Delta\omega_\mu(t)$

Static frequency shift:  $\Delta\omega_0 = \sum_\mu \Delta\omega_{\mu 0} = -V^2 \sum_\mu k_\mu$

Static Lorentz coefficient:  $k = \sum_\mu k_\mu$

# Ponderomotive Effects – Mechanical Modes

$$\Delta \ddot{\omega}_\mu + \frac{2}{\tau_\mu} \Delta \dot{\omega}_\mu + \Omega_\mu^2 \Delta \omega_\mu = -\Omega_\mu^2 k_\mu V_0^2 + \cancel{n(t)}$$

**Fluctuations around steady state:**

$$\begin{aligned}\Delta \omega_\mu &= \Delta \omega_{\mu o} + \delta \omega_\mu \\ V &= V_0(1 + \delta v)\end{aligned}$$

**Linearized equation of motion for mechanical mode:**

$$\delta \ddot{\omega}_\mu + \frac{2}{\tau_\mu} \delta \dot{\omega}_\mu + \Omega_\mu^2 \delta \omega_\mu = -2\Omega_\mu^2 k_\mu V_0^2 \delta v$$

**The mechanical mode is driven by fluctuations in the electromagnetic mode amplitude.**

**Variations in the mechanical mode amplitude causes a variation of the electromagnetic mode frequency, which can cause a variation of its amplitude.**

→**Closed feedback system between electromagnetic and mechanical modes, that can lead to instabilities.**

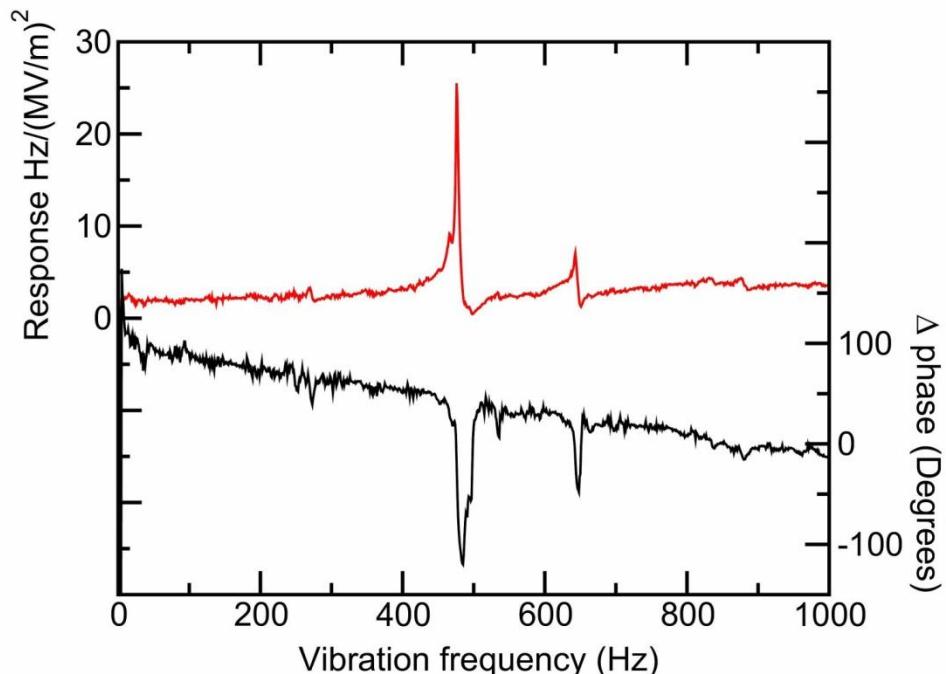
# Lorentz Transfer Function

$$\delta\ddot{\omega}_\mu + \frac{2}{\tau_\mu} \delta\dot{\omega}_\mu + \Omega_\mu^2 \delta\omega_\mu = -2\Omega_\mu^2 k_\mu V_0^2 \delta v$$

$$\delta\omega_\mu(\omega) = \frac{-2\Omega_\mu^2 k_\mu V_0^2}{(\Omega_\mu^2 - \omega^2) + \frac{2}{\tau_\mu} i\omega} \delta v(\omega)$$

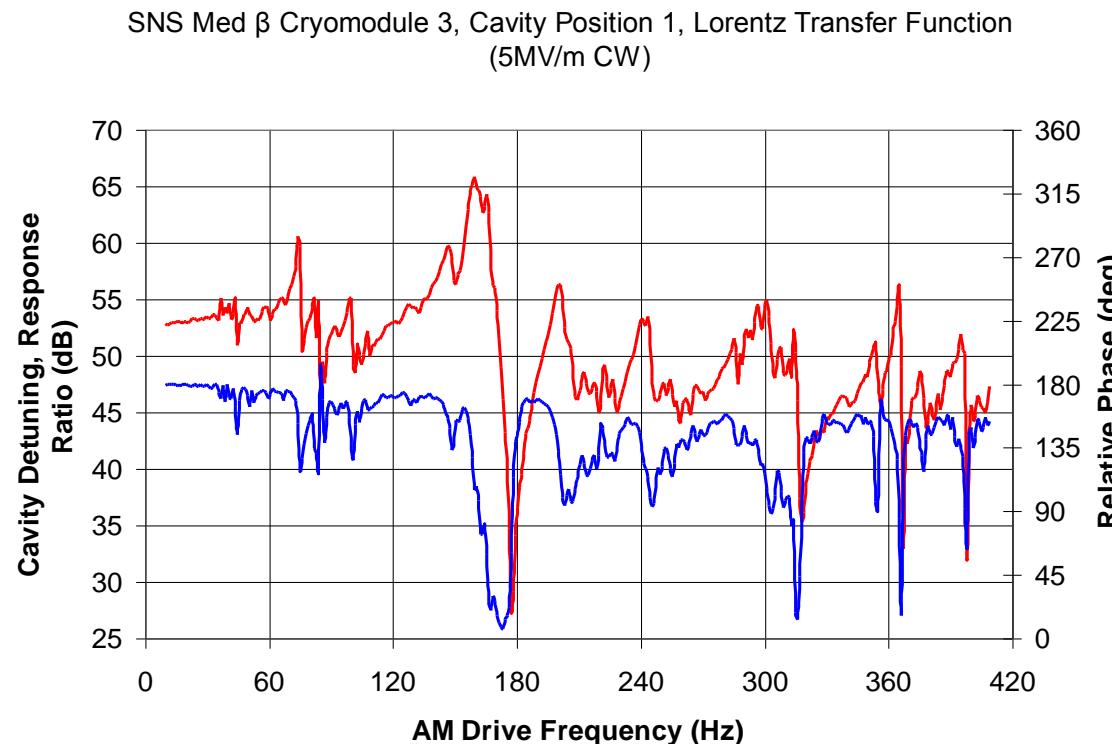
TEM-class cavities  
ANL, single-spoke, 354 MHz,  $\beta=0.4$

**simple spectrum with  
few modes**

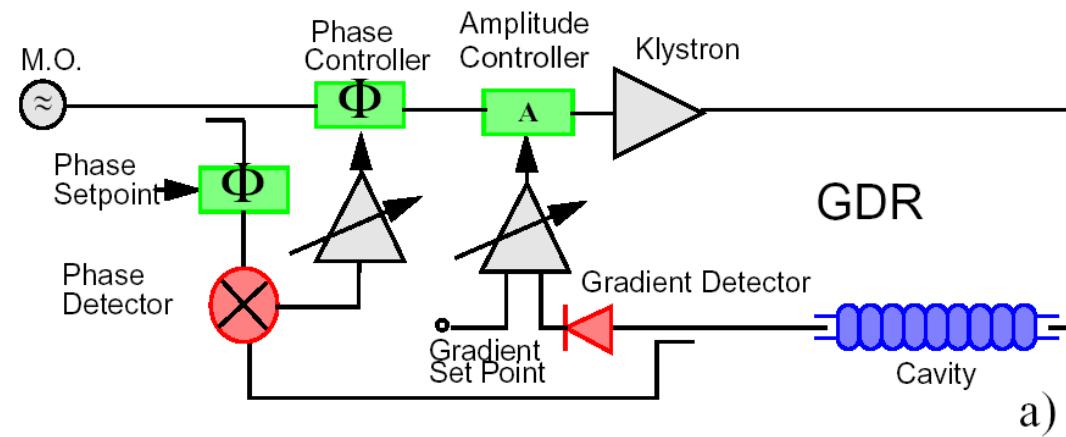


# Lorentz Transfer Function

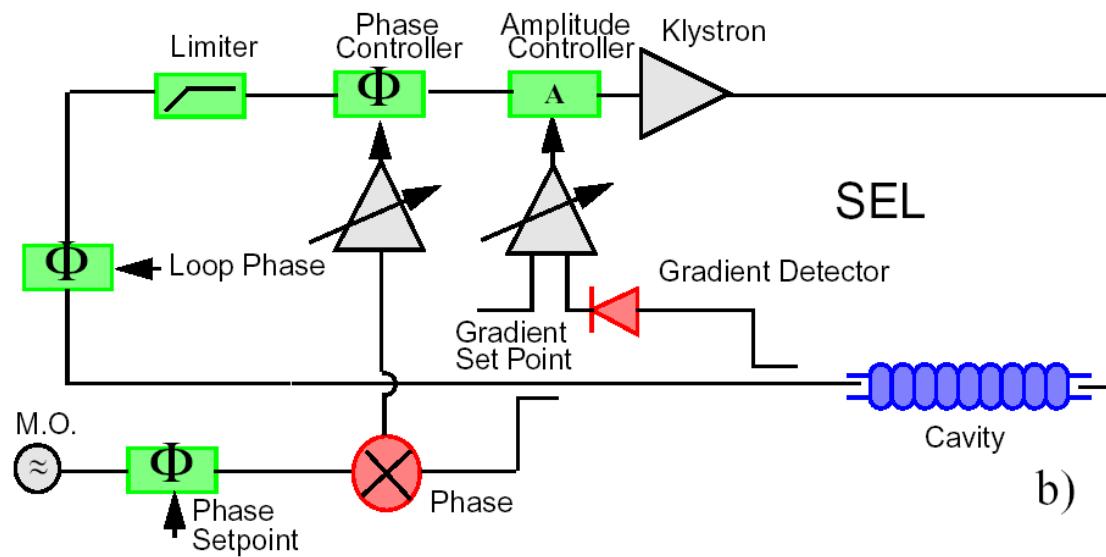
TM-class cavities (Jlab, 6-cell elliptical, 805 MHz,  $\beta=0.61$ )  
Rich frequency spectrum from low to high frequencies  
Large variations between cavities



# GDR and SEL



a)



b)

# Generator-Driven Resonator

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- In a generator-driven resonator the coupling between the electromagnetic and mechanical modes can lead to two ponderomotive instabilities
- **Monotonic instability** : Jump phenomenon where the amplitudes of the electromagnetic and mechanical modes increase or decrease exponentially until limited by non-linear effects
- **Oscillatory instability** : The amplitudes of both modes oscillate and increase at an exponential rate until limited by non-linear effects

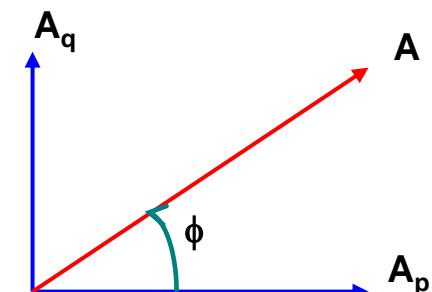
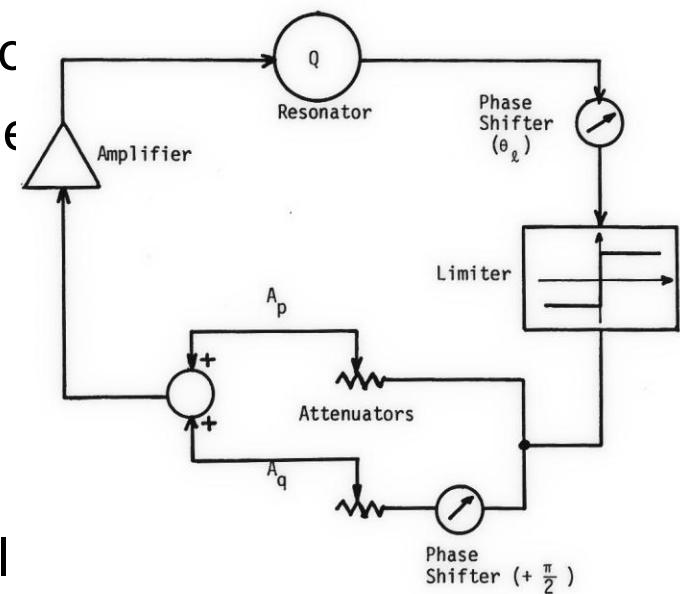
# Self-Excited Loop-Principle of Stabilization

Controlling the external phase shift  $\theta_l$  can compensate fluctuations in the cavity frequency  $\omega_c$  so the external frequency reference  $\omega_r$ .

$$\omega = \omega_c + \frac{\omega_c}{2Q} \tan \theta_l$$

Instead of introducing an additional external reference, this is usually done by adding a signal in quadrature.

→ The cavity field amplitude is unaffected by the phase stabilization even in the absence of amplitude feedback.



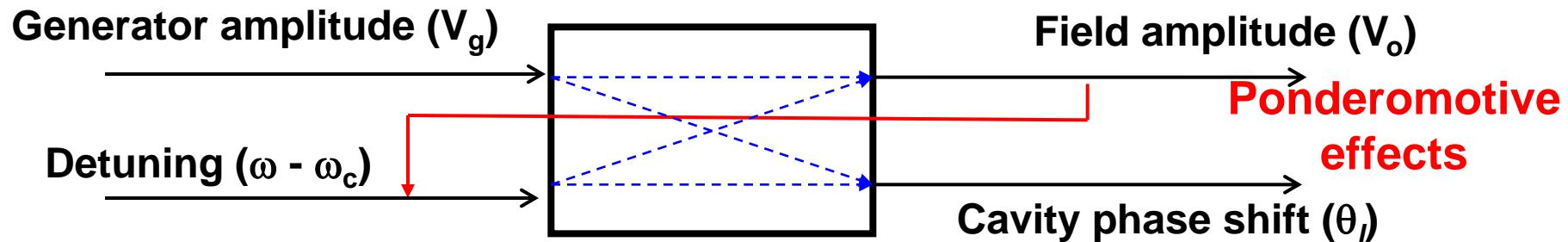
# Self-Excited Loop

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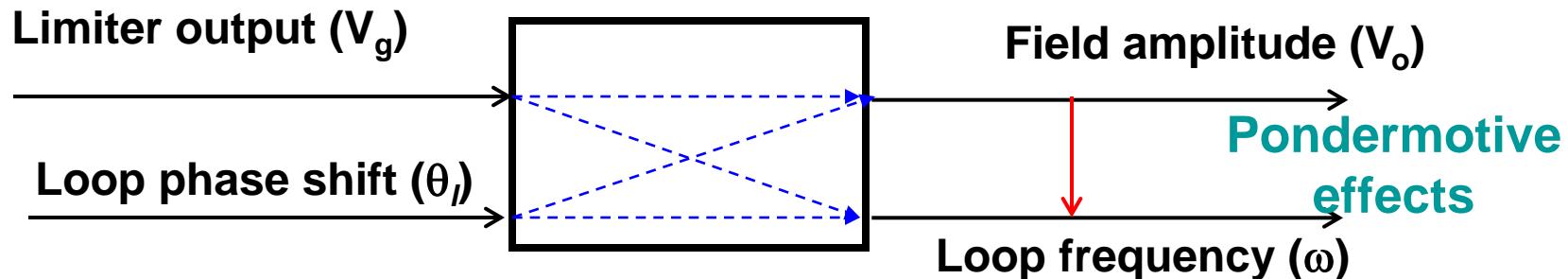
- Resonators operated in self-excited loops in the absence of feedback are free of ponderomotive instabilities. An SEL is equivalent to the ideal VCO.
  - Amplitude is stable
  - Frequency of the loop tracks the frequency of the cavity
- Phase stabilization can reintroduce instabilities, but they are easily controlled with small amount of amplitude feedback

# Input-Output Variables

- Generator - driven cavity

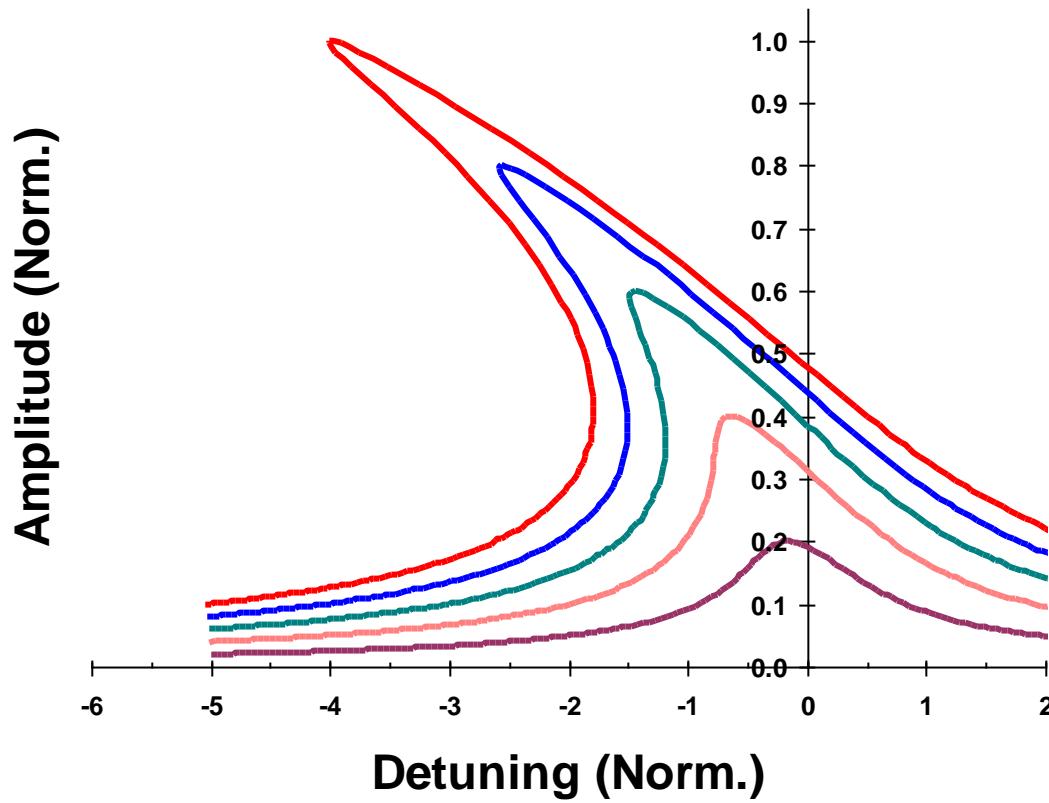


- Cavity in a self-excited loop



# Lorentz Detuning

During transient operation (rise time and decay time) the loop frequency automatically tracks the resonator frequency. Lorentz detuning has no effect and is automatically compensated



# Micromechanics

- Micromechanics: changes in frequency caused by connections to the external world
  - Vibrations
  - Pressure fluctuations

When phase and amplitude feedbacks are active, ponderomotive effects can change the response to external disturbances

$$\delta\ddot{\omega}_\mu + \frac{2}{\tau_\mu} \delta\dot{\omega}_\mu + \Omega_\mu^2 \delta\omega_\mu = -2\Omega_\mu^2 k_\mu V_0^2 \delta v + n(t)$$

# Micromechanics

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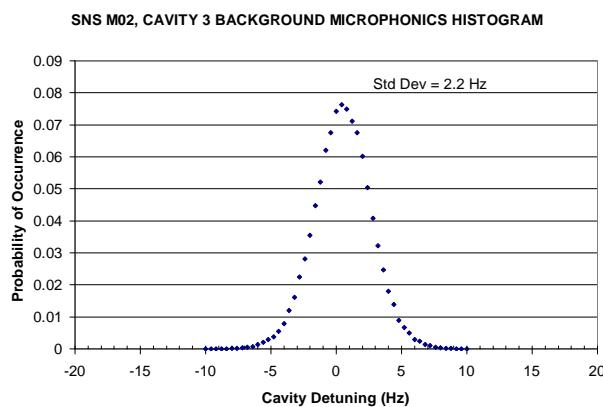
Two extreme classes of driving terms:

- Deterministic, monochromatic
  - Constant, well defined frequency
  - Constant amplitude
- Stochastic
  - Broadband (compared to bandwidth of mechanical mode)
  - Will be modeled by gaussian stationary white noise process

# Microphonics (probability density)

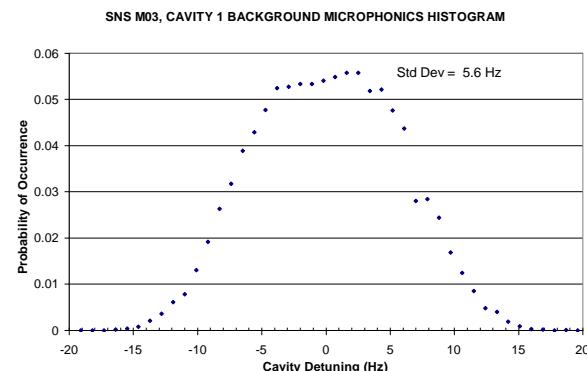
Single gaussian

Noise driven



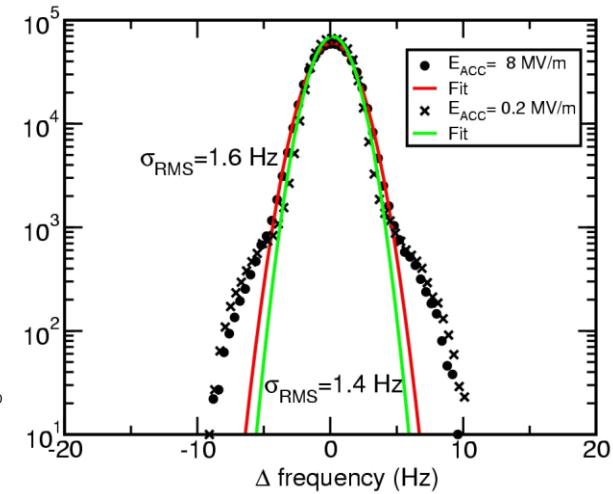
Bimodal

Single-frequency driven



Multi-gaussian

Non-stationary noise



805 MHz TM

805 MHz TM

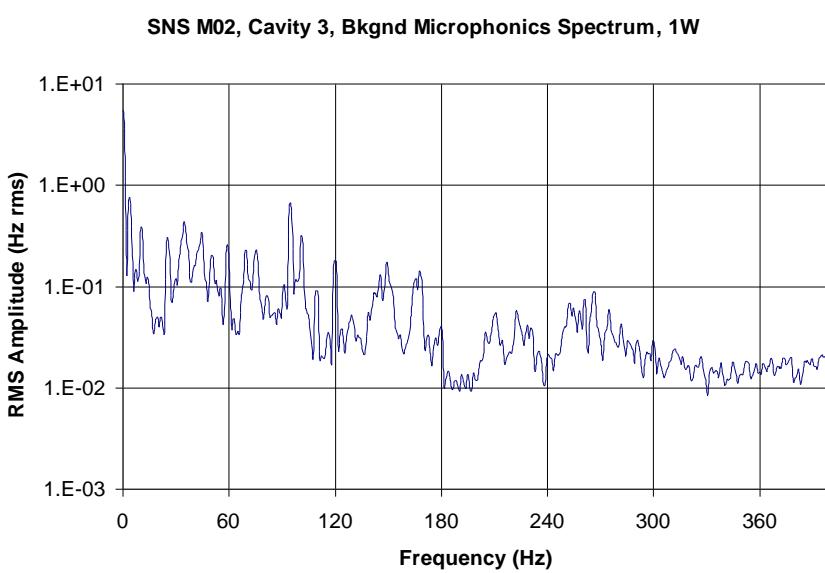
172 MHz TEM

# Microphonics (frequency spectrum)

**TM-class cavities (JLab, 6-cell elliptical, 805 MHz,  $\beta=0.61$ )**

**Rich frequency spectrum from low to high frequencies**

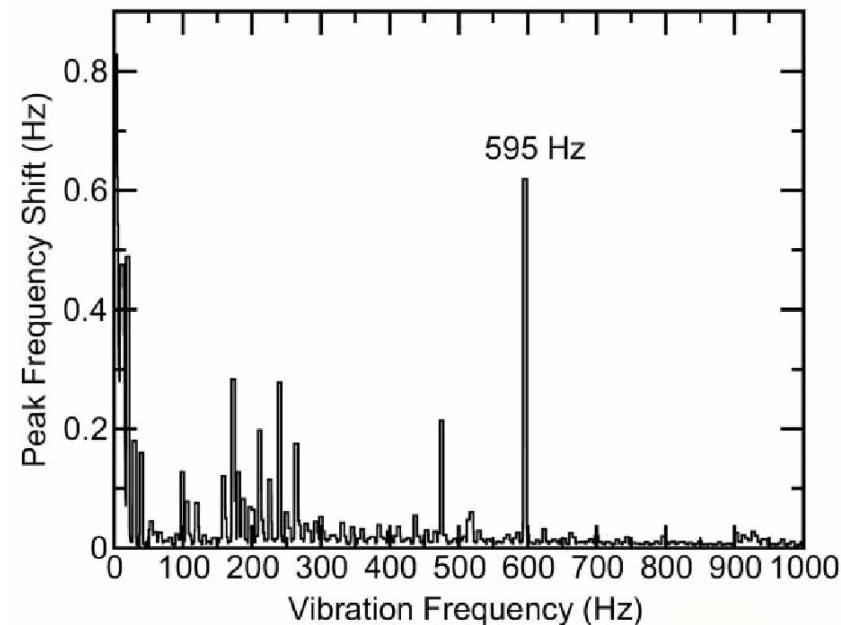
**Large variations between cavities**



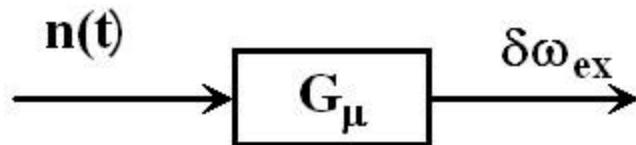
**TEM-class cavities (ANL, single-spoke, 354 MHz,  $\beta=0.4$ )**

**Dominated by low frequency (<10 Hz) from pressure fluctuations**

**Few high frequency mechanical modes that contribute little to microphonics level.**

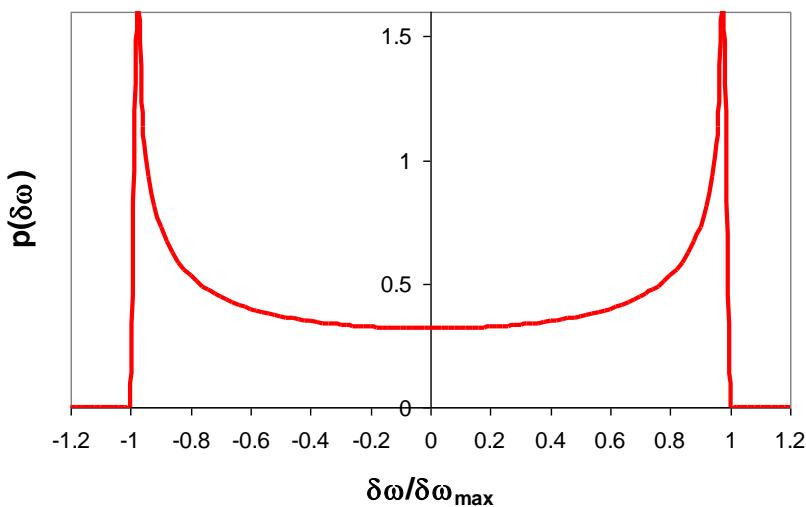


# Probability Density (histogram)



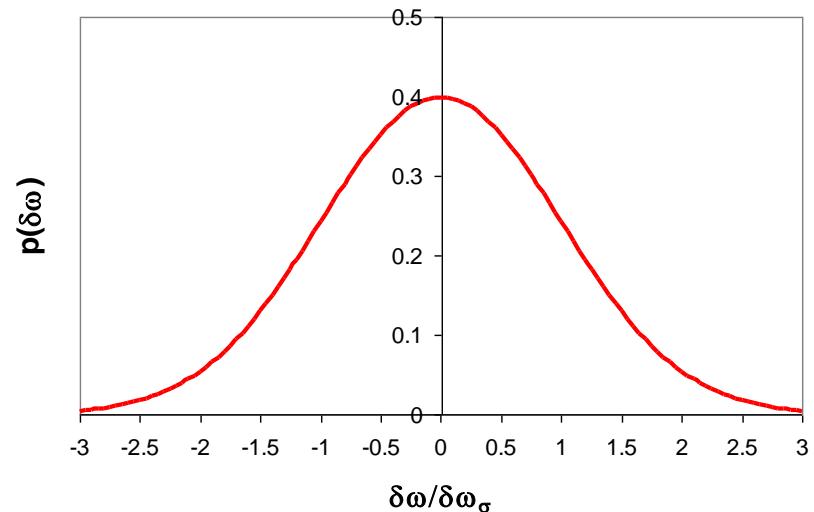
Harmonic oscillator ( $\Omega_\mu, \tau_\mu$ ) driven by:

Single frequency, constant amplitude



$$p(\delta\omega) = \frac{1}{\pi \sqrt{\delta\omega_{max}^2 - \delta\omega^2}}$$

White noise, gaussian



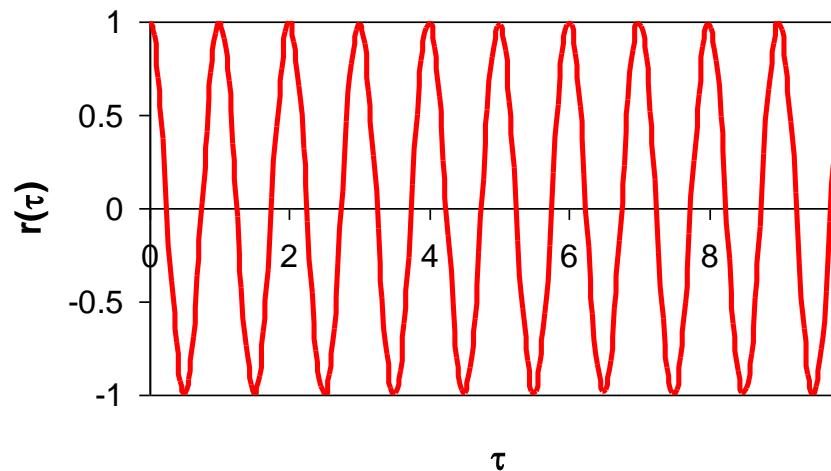
$$p(\delta\omega) = \frac{1}{\sigma_\omega \sqrt{2\pi}} \exp\left[-\frac{1}{2}\left(\frac{\delta\omega}{\sigma_\omega}\right)^2\right]$$

# Autocorrelation Function

$$R_x(\tau) = \langle x(t) x(t + \tau) \rangle = \lim_{T \rightarrow \infty} \frac{1}{T} \int_0^T x(t) x(t + \tau) dt$$

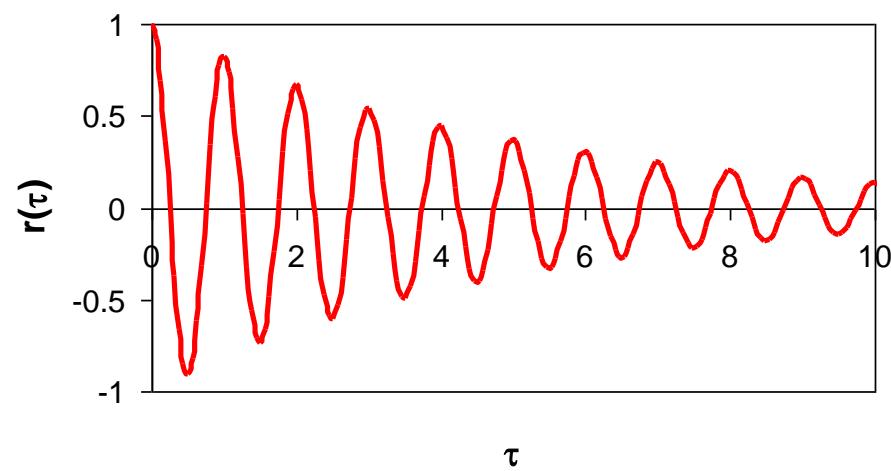
Harmonic oscillator ( $\Omega_\mu, \tau_\mu$ ) driven by:

Single frequency, constant amplitude



$$r_{\delta\omega}(\tau) = \frac{R_{\delta\omega}(\tau)}{R_{\delta\omega}(0)} = \cos(\omega_d \tau)$$

White noise, gaussian



$$r_{\delta\omega}(\tau) = \frac{R_{\delta\omega}(\tau)}{R_{\delta\omega}(0)} = \cos(\Omega_\mu \tau) e^{-|\tau/\tau_\mu|}$$

# Stationary Stochastic Processes

$x(t)$ : stationary random variable

Autocorrelation function:

$$R_x(\tau) = \langle x(t) x(t + \tau) \rangle = \lim_{T \rightarrow \infty} \frac{1}{T} \int_0^T x(t) x(t + \tau) dt$$

Spectral Density  $S_x(\omega)$ : Amount of power between  $\omega$  and  $d\omega$

$S_x(\omega)$  and  $R_x(\tau)$  are related through the Fourier Transform (Wiener-Khintchine)

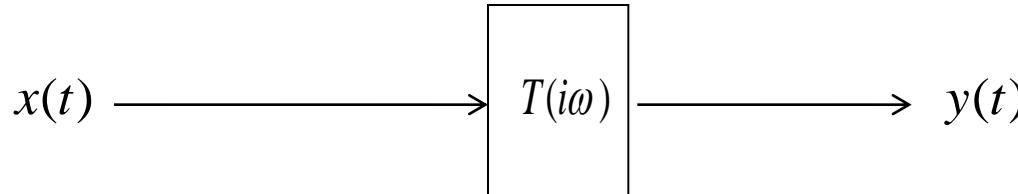
$$S_x(\omega) = \frac{1}{2\pi} \int_{-\infty}^{\infty} R_x(\tau) e^{-i\omega\tau} d\tau \quad R_x(\tau) = \int_{-\infty}^{\infty} S_x(\omega) e^{i\omega\tau} d\omega$$

Mean square value:

$$\langle x^2 \rangle = R_x(0) = \int_{-\infty}^{\infty} S_x(\omega) d\omega$$

# Stationary Stochastic Processes

For a stationary random process driving a linear system



$$\langle y^2 \rangle = R_y(0) = \int_{-\infty}^{+\infty} S_y(\omega) d\omega \quad \langle x^2 \rangle = R_x(0) = \int_{-\infty}^{+\infty} S_x(\omega) d\omega$$

$R_y(\tau)$  [ $R_x(\tau)$ ]: auto correlation function of  $y(t)$  [ $x(t)$ ]

$S_y(\omega)$  [ $S_x(\omega)$ ]: spectral density of  $y(t)$  [ $x(t)$ ]

$$S_y(\omega) = S_x(\omega) |T(i\omega)|^2$$

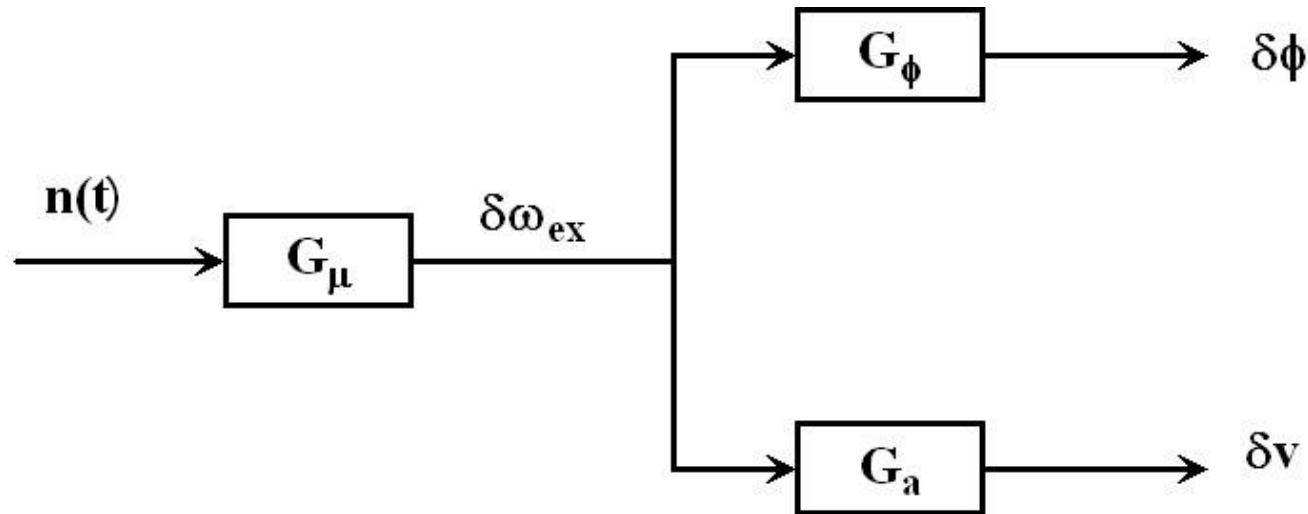
$$\langle y^2 \rangle = \int_{-\infty}^{+\infty} S_x(\omega) |T(i\omega)|^2 d\omega$$

# Performance of Control System

Residual phase and amplitude errors caused by microphonics

Can also be done for beam current amplitude and phase fluctuations

Assume a single mechanical oscillator of frequency  $\Omega_\mu$  and decay time  $\tau_\mu$  excited by white noise of spectral density  $A^2$



# Performance of Control System

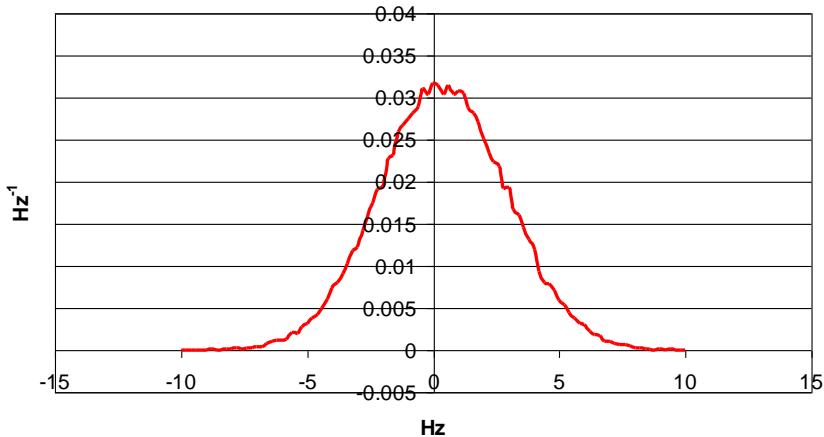
$$\langle \delta\omega_{ex}^2 \rangle = A^2 \int_{-\infty}^{+\infty} |G_\mu(i\omega)|^2 d\omega = A^2 \int_{-\infty}^{+\infty} \frac{d\omega}{\left| -\omega^2 + \frac{2}{\tau_\mu} i\omega + \Omega_\mu^2 \right|^2} = A^2 \frac{\pi\tau_\mu}{2\Omega_\mu^2}$$

$$\langle \delta v^2 \rangle = A^2 \int_{-\infty}^{+\infty} |G_\mu(i\omega) G_a(i\omega)|^2 d\omega = \langle \delta\omega_{ex}^2 \rangle \frac{2\Omega_\mu^2}{\pi\tau_\mu} \int_{-\infty}^{+\infty} \left| \frac{G_a(i\omega)}{-\omega^2 + \frac{2}{\tau_\mu} i\omega + \Omega_\mu^2} \right|^2 d\omega$$

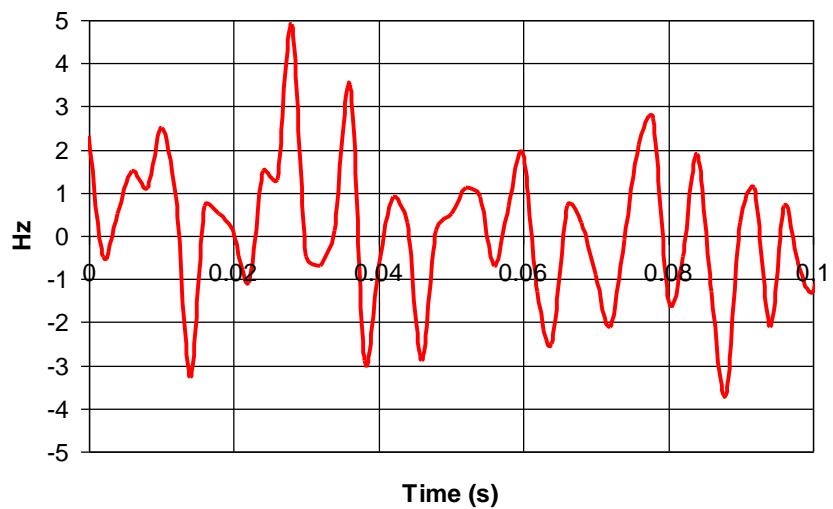
$$\langle \delta\phi^2 \rangle = A^2 \int_{-\infty}^{+\infty} |G_\mu(i\omega) G_\phi(i\omega)|^2 d\omega = \langle \delta\omega_{ex}^2 \rangle \frac{2\Omega_\mu^2}{\pi\tau_\mu} \int_{-\infty}^{+\infty} \left| \frac{G_\phi(i\omega)}{-\omega^2 + \frac{2}{\tau_\mu} i\omega + \Omega_\mu^2} \right|^2 d\omega$$

# The Real World

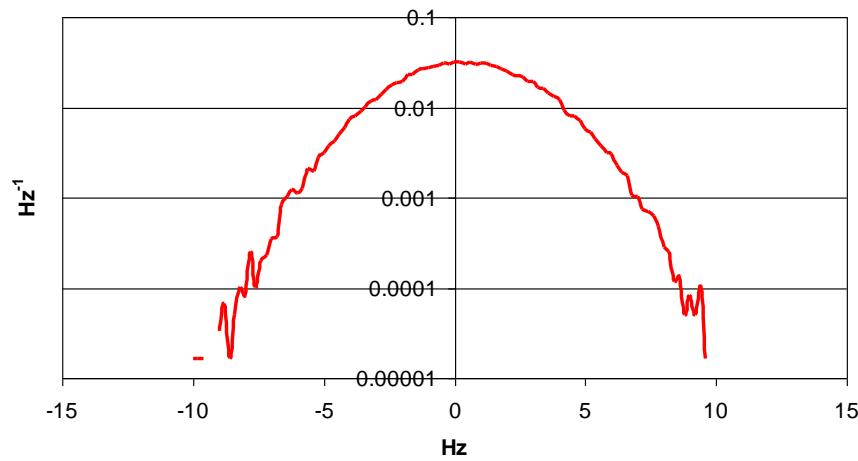
Probability Density



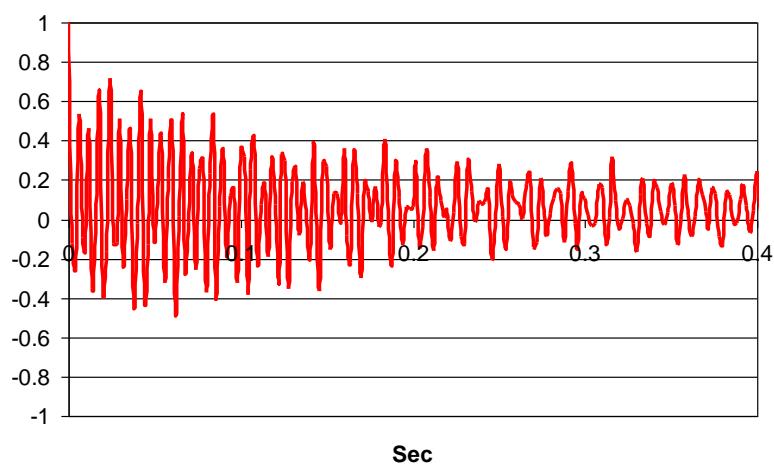
Microphonics



Probability Density



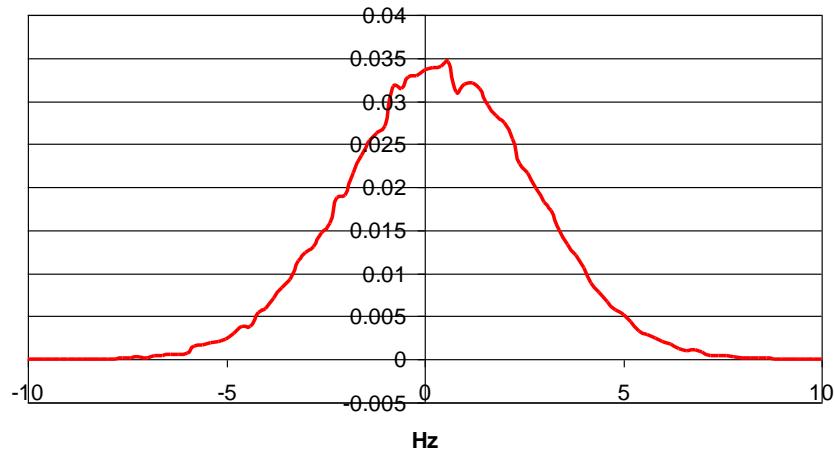
Normalized Autocorrelation Function



# The Real World

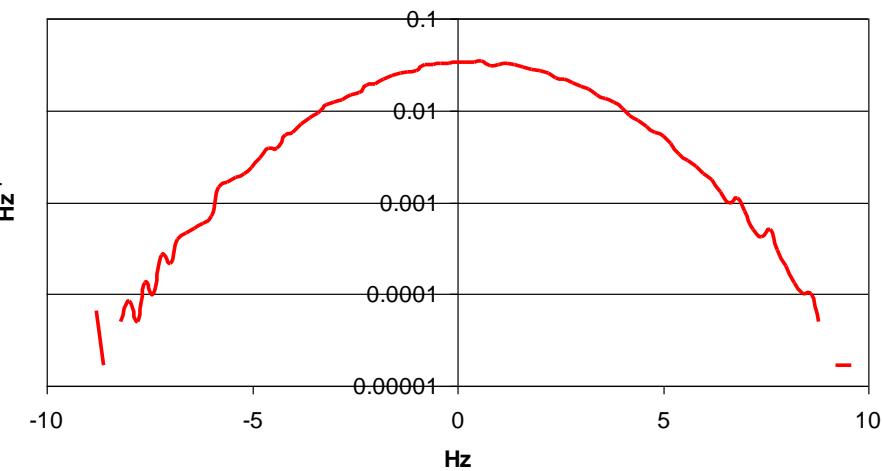
Probability Density

Hz<sup>-1</sup>



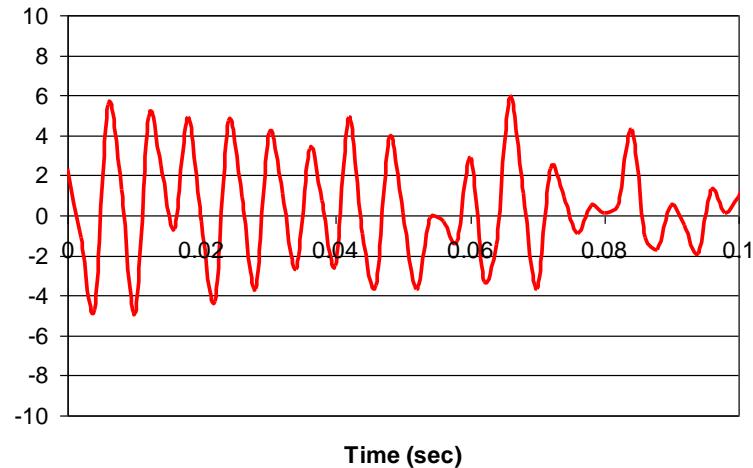
Probability Density

Hz<sup>-1</sup>

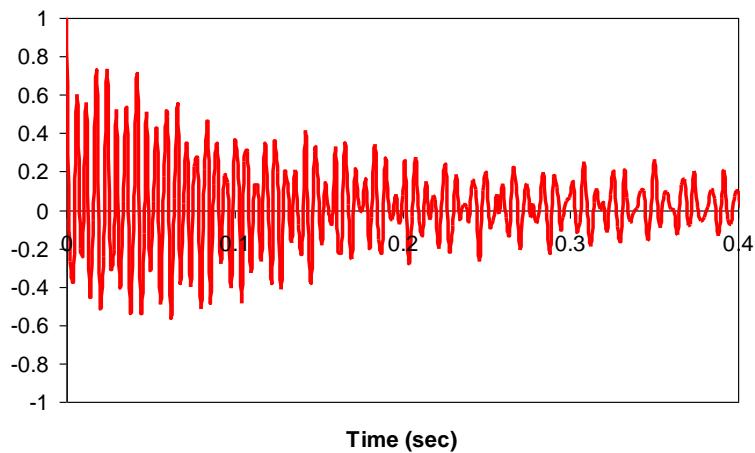


Microphonics

Hz



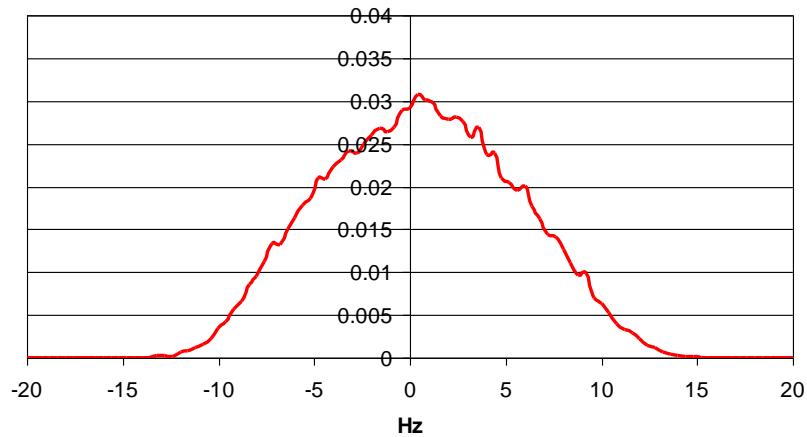
Normalized Autocorrelation Function



# The Real World

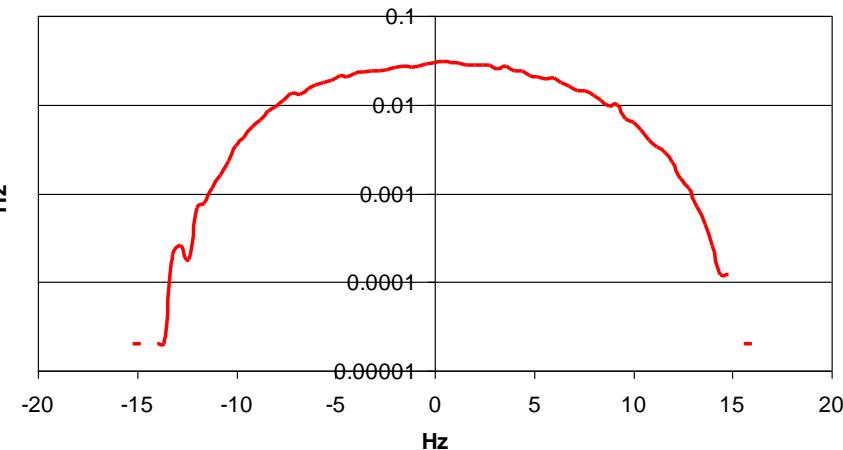
Probability density

Hz<sup>-1</sup>

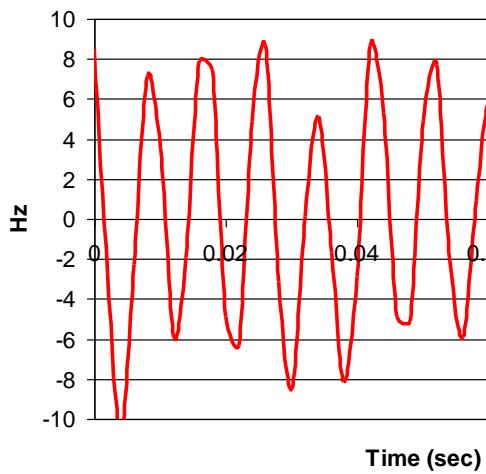


Probability density

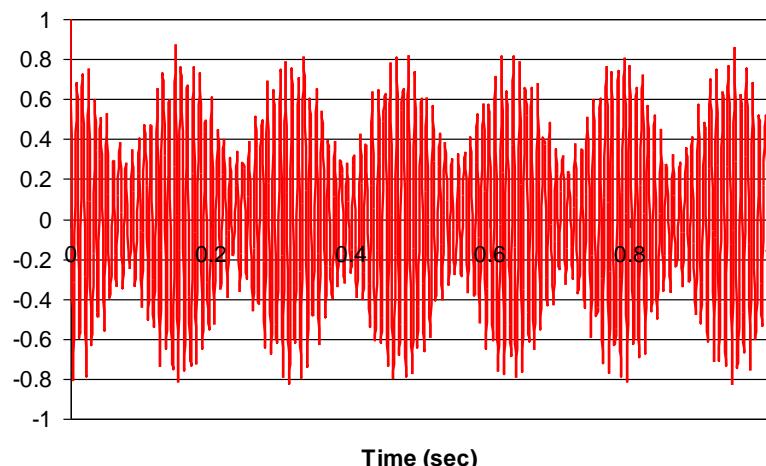
Hz<sup>-1</sup>



Micromphonics

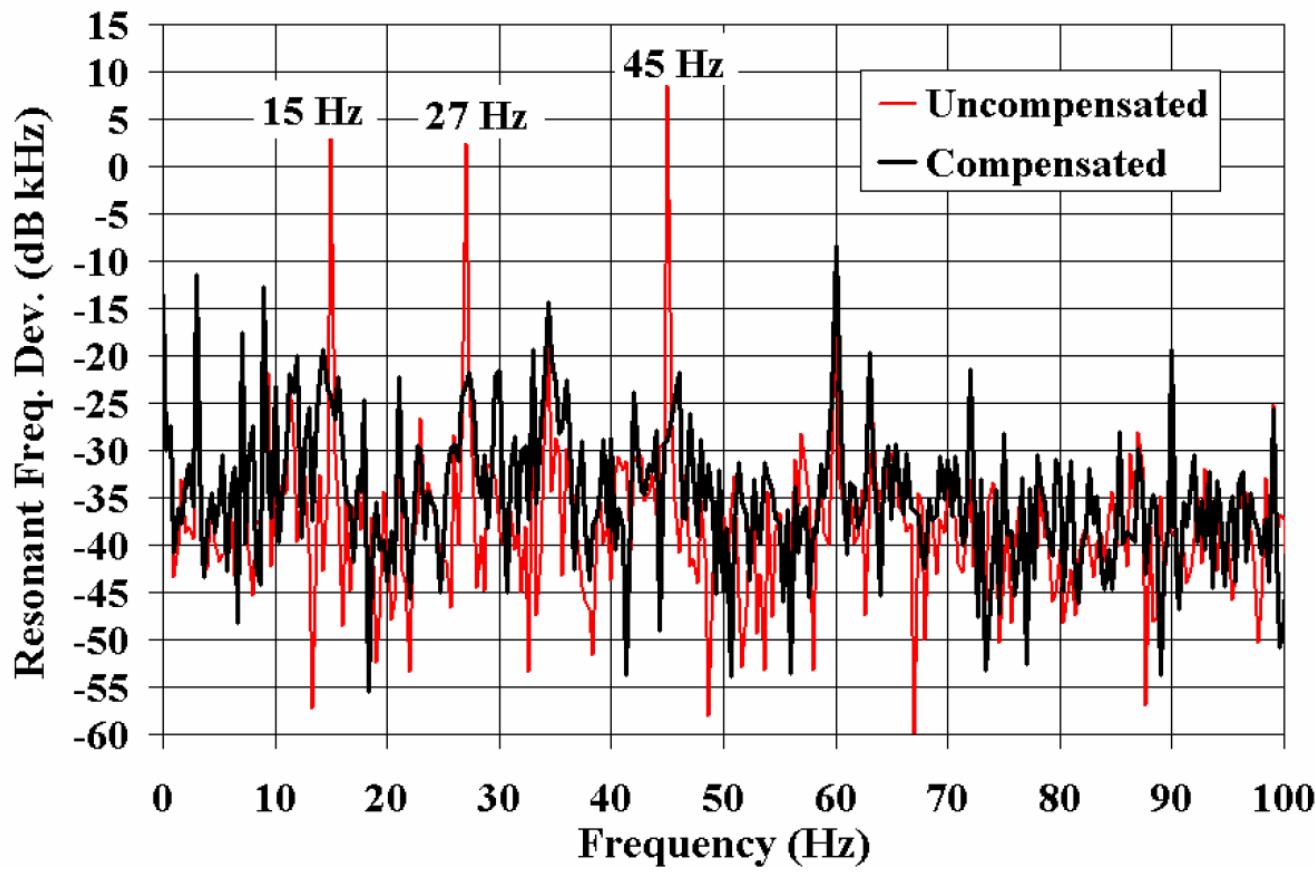


Normalized Autocorrelation Function



# Piezo Control of Microphonics

FNAL, 3-cell 3.9 GHz



# Piezo control of microphonics

MSU, 6-cell elliptical 805 MHz,  $\beta=0.49$

Adaptive feedforward compensation

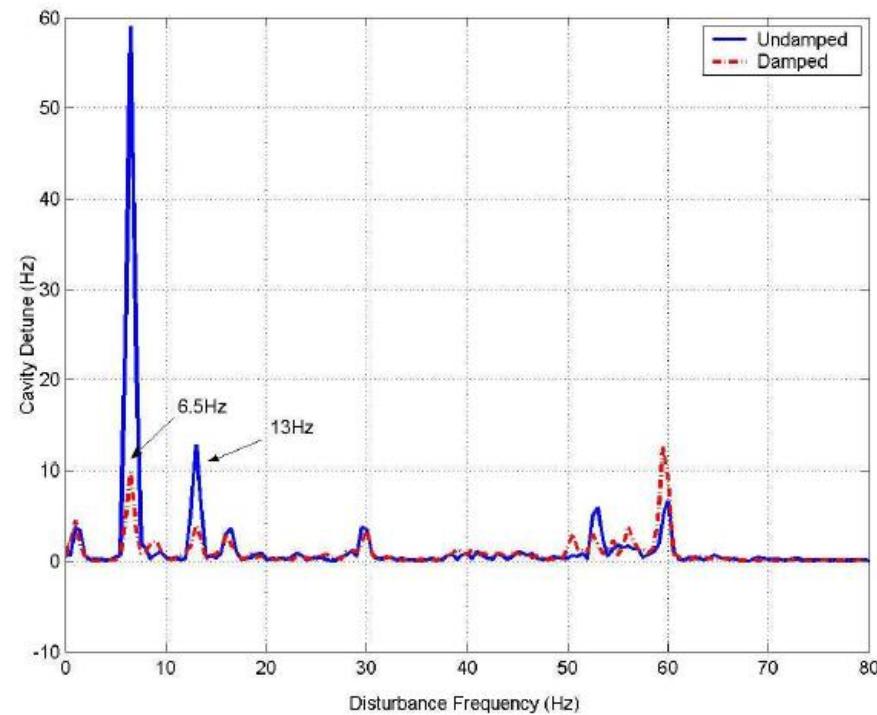


Figure 2. Active damping of helium oscillations at 2K.

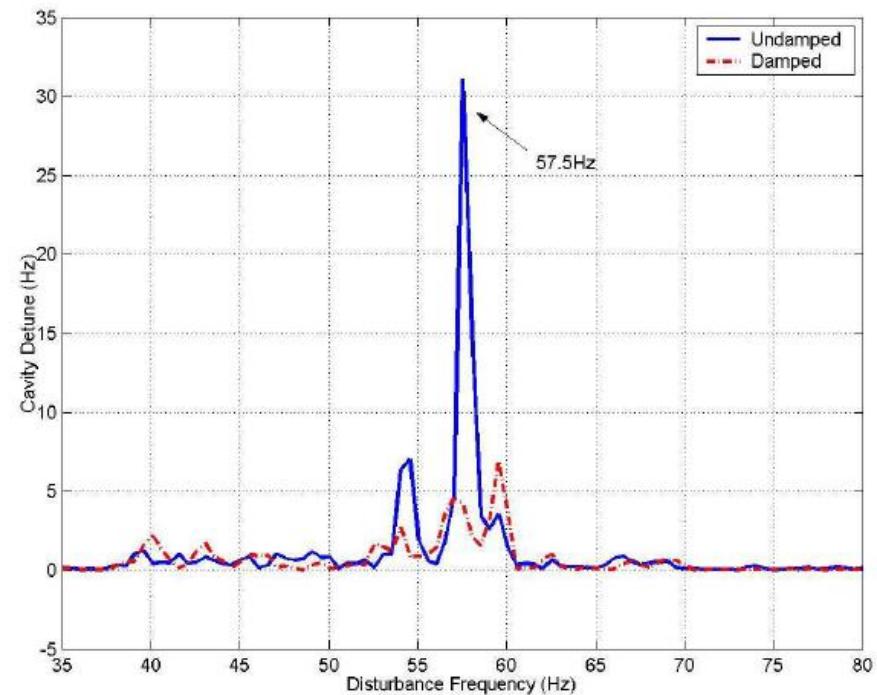


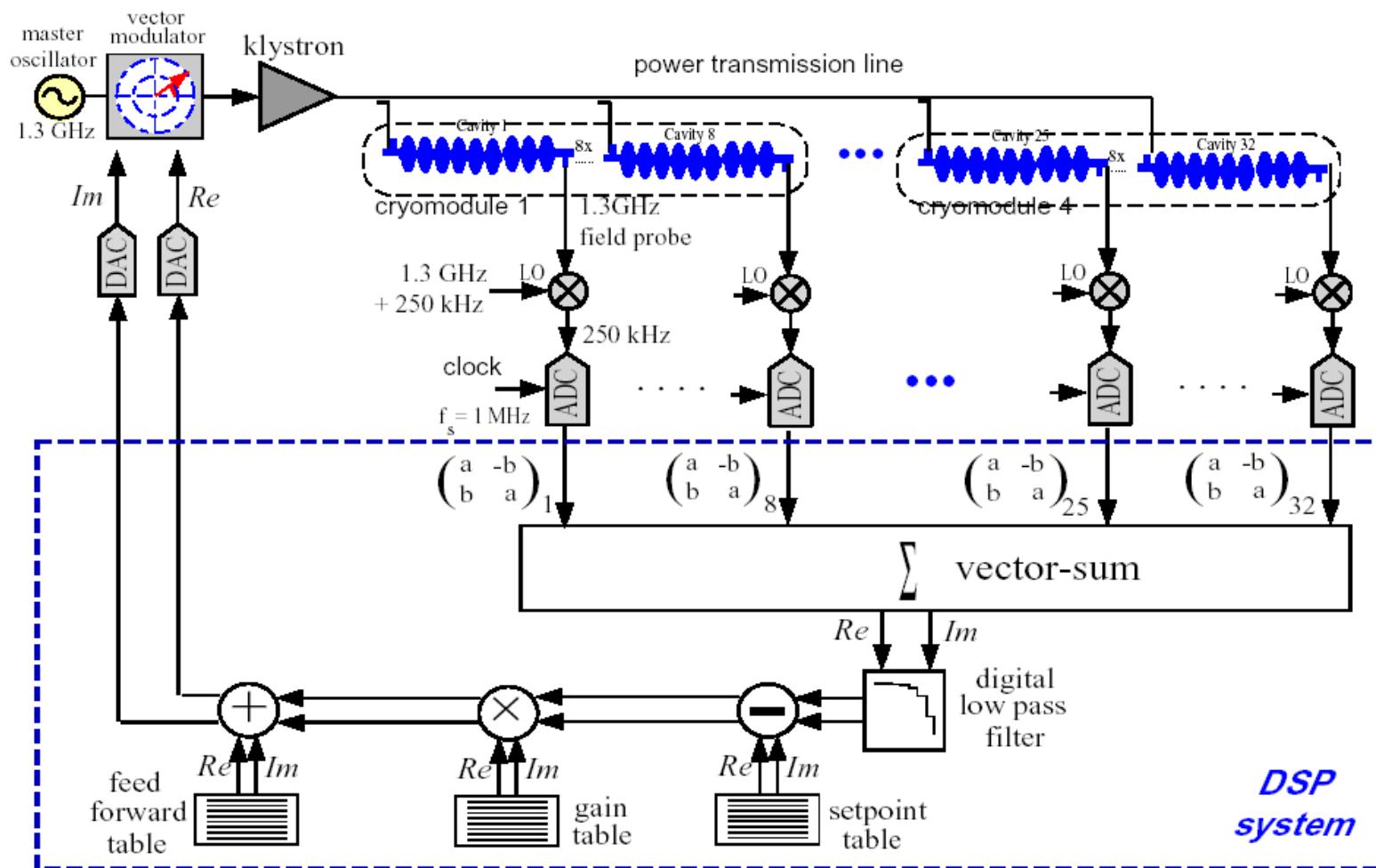
Figure 3. Active damping of external vibration at 2K.

# SEL and GDR

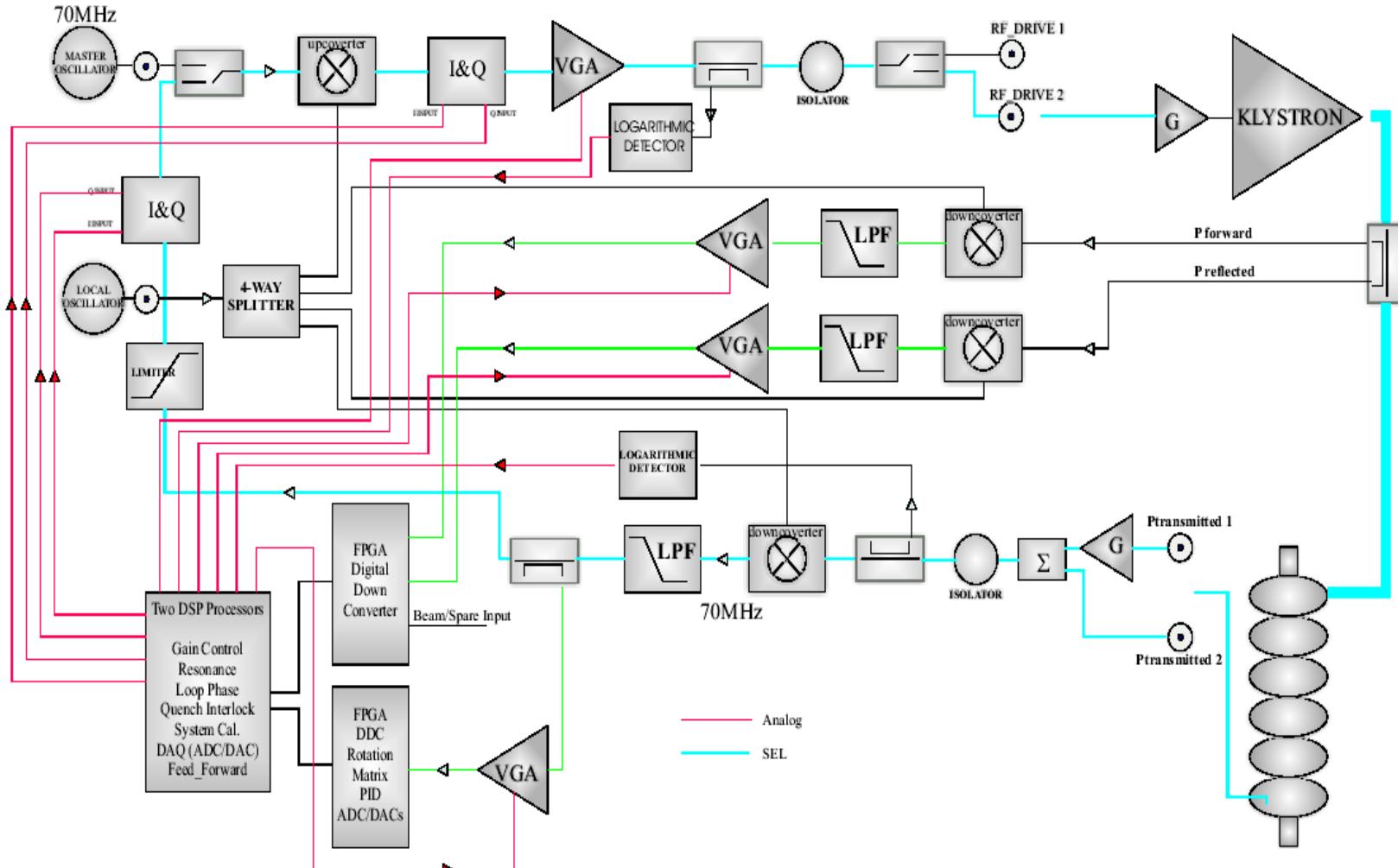
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- SEL are best suited for high gradient, high-loaded Q cavities operated cw.
  - Well behaved with respect to ponderomotive instabilities
  - Unaffected by Lorentz detuning at power up
  - Able to run independently of external rf source
  - Rise time can be random and slow (starts from noise)
- GDR are best suited for low-Q cavities operated for short pulse length.
  - Fast predictable rise time
  - Power up can be hampered by Lorentz detuning

# TESLA Control System

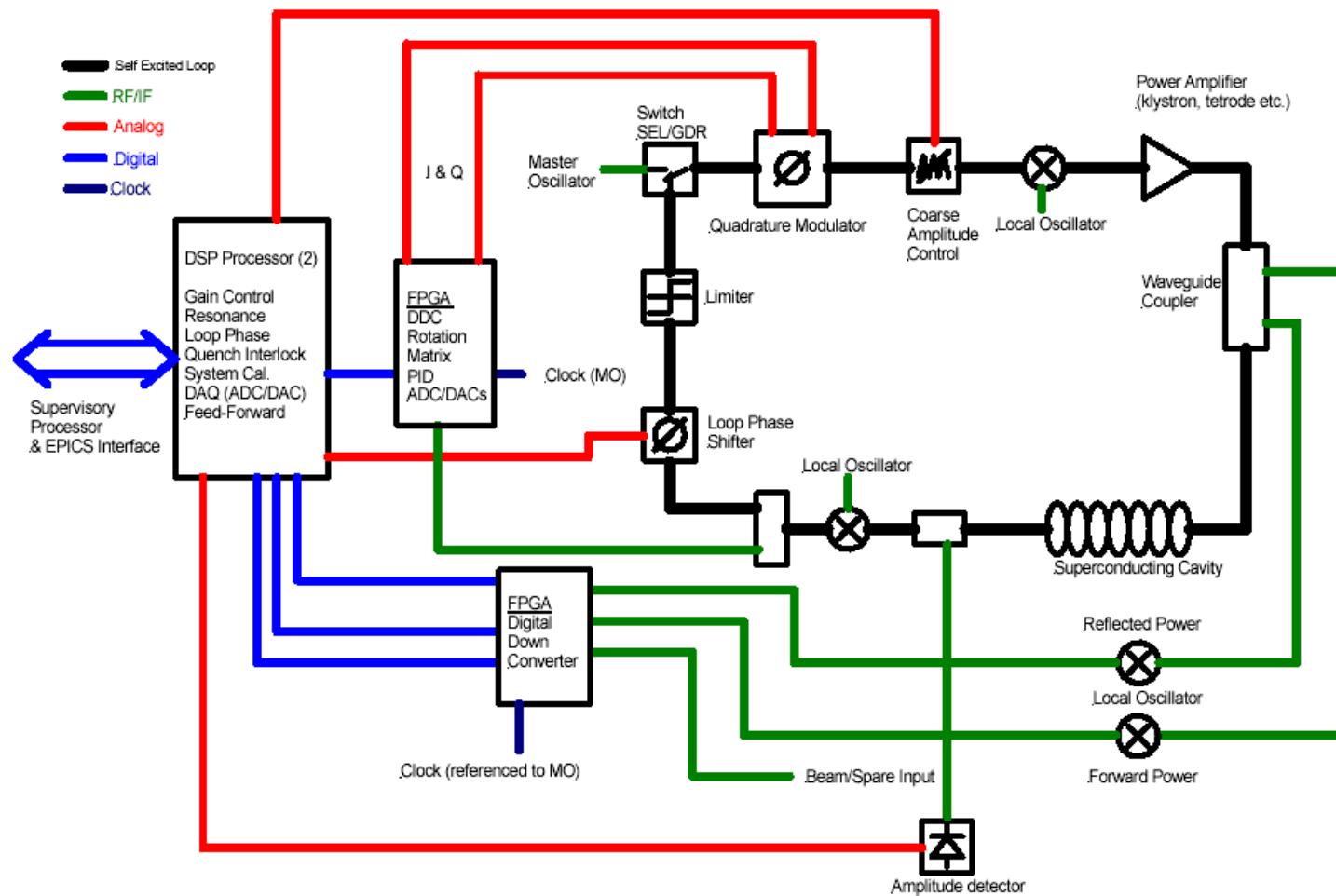


# Low level rf control development



Concept for a LLRF control system

# Basic LLRF Block Diagram



# Pulsed Operation

- Under pulsed operation Lorentz detuning can have a complicated dynamic behavior

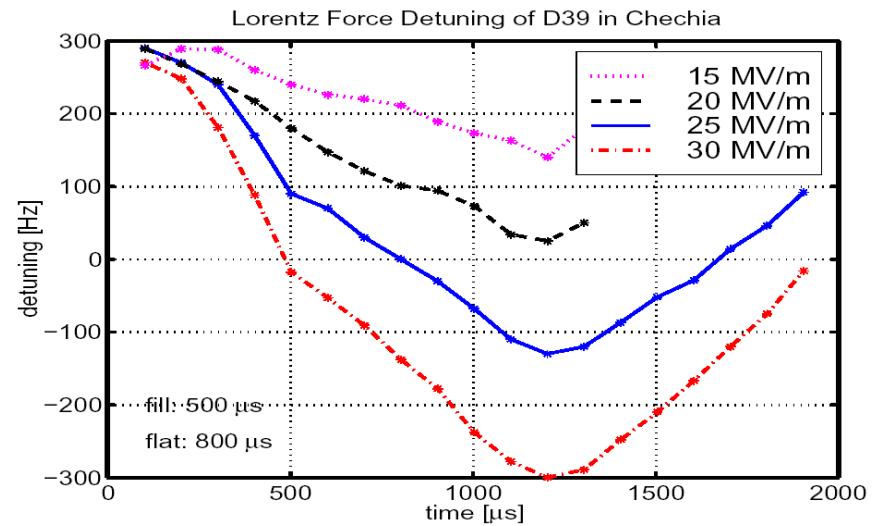
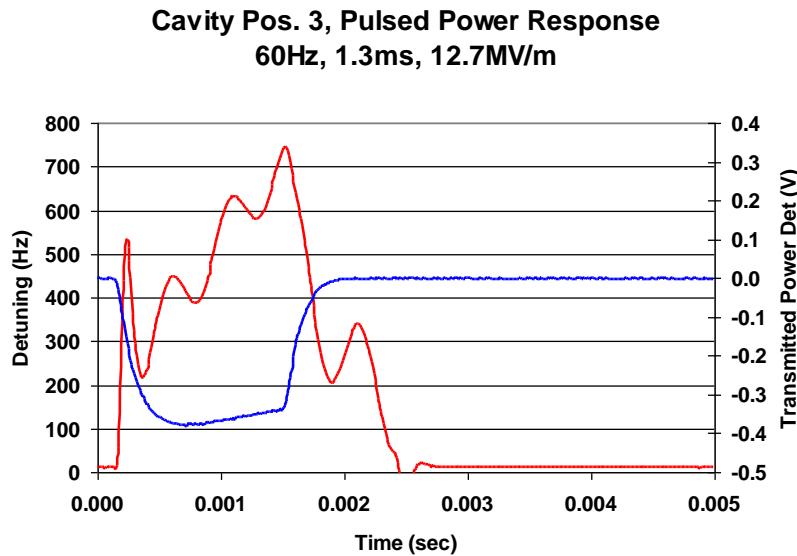


Fig. 2: Lorentz force detuning measured for a TESLA cavity at different gradients.

# Pulsed Operation

- Fast piezoelectric tuners can be used to compensate the dynamic Lorentz detuning

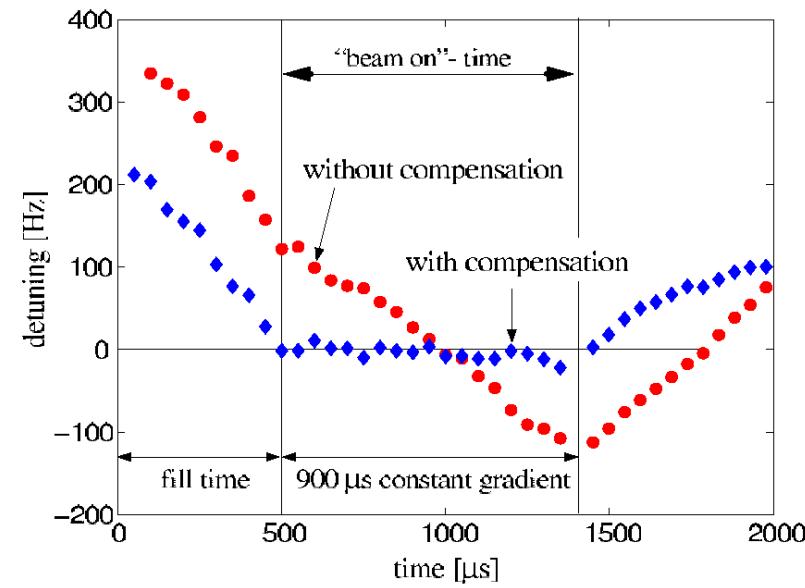
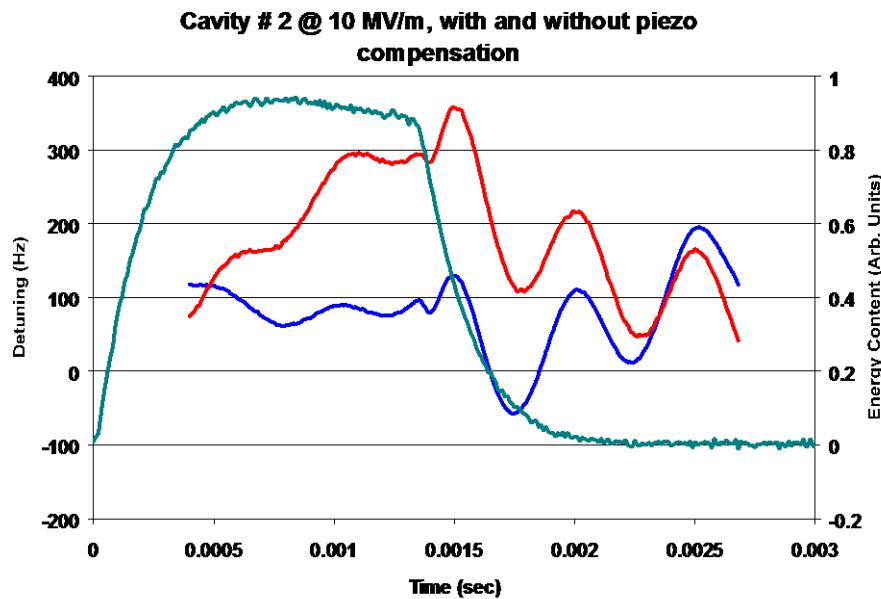


Figure 2. Lorentz force compensation at the TTF