Passive isolation: Pre-isolation for FF quads



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Retrospective view.

In the first CLIC MDI layout (legacy of ILC MDI), the QD0s were supported by the detector, or by a pillar on the detector moving platform. However it became clear, after the measuring campaign at CMS, that the vibrations generated by the detector itself would make impossible to achieve the stability requirements given for the FF magnets at CLIC.



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Scope of work.

Try to attenuate, at its source, ground motion vertical excitations, in the range 1 – 50 Hz, to make life easier to the following stabilization systems.

Objective.

Stabilize FF magnets to better than 0.2 nm (rms) at 4 Hz, using an integrated approach of three systems, each one with its dynamic range and frequency response:

♦ Passive pre-isolator

♦ Active mechanical stabilization

♦ Beam-based stabilization





Stabilization block diagram.



- X ground motion
- K_i isolator transfer function
- D external disturbances else than ground-motion
- W white noise on beam position
- ΔY beam position signal

from B. Carron/CLIC Stabilization Working Group





Pre-isolator – How does it work ?

Low dynamic stiffness (k) mount

natural frequency around 1 Hz

Acts as a low-pass filter for the ground motion (w)



between 50 and 100 tons

Provides the inertia necessary to withstand the external disturbances (F_a) , such as air flow, acoustic pressure, etc.)



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Where does it fit ?

Ideally located at the end of the machine tunnel, just in front of the detector, on both sides.

drawing by N. Siegrist









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Things missing in the model:

- Pre-alignment mechanics
- Final doublet's geometries (using, for now, 3-D point masses with estimated inertias)
- Final doublet's supporting structures (girders, etc.)
- Pre-isolator's supports (using , for now, 1-D springs with appropriate stiffness)





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Response to excitation in the vertical direction.







Response to excitation in the horizontal directions.



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Random vibration response.







Random vibration response.



Reduction in r.m.s. displacements by a factor 30 above 4 Hz



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Example of pre-isolator application in industry.

Vibration isolation system at the Centre for Metrology and Accreditation – Helsinki, Finland

4 independent seismic masses (3x70 ton + 1x140 ton) 0.8 Hz pneumatic vibration isolators ("air springs")







Experimental set-up – Why ?

The idea is promising, but...

Is the system's performance amplitude dependent ? How will it react to different noise sources, else than micro-seism ? What about energy loss mechanisms (friction...) ? What performance can we expect ?

We have built a prototype to try to answer to these questions. Although rather different from the reference design, it will serve to validate the idea of having a low-frequency mechanical filter in front of the FF stabilization chain.





Experimental set-up – How ? The prototype needs to be: Frictionless pivotal joints Simple to design/build/assemble Easy to "debug" & tune Cheap Center of mass **Proposal:** 40 ton mass supported by 4 structural beams acting as flexural springs







Photo of the prototype.







Experimental set-up – expected performance.







Experimental set-up – position of the vibration sensors.





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feet resonances





Future plans.

- A passive low-frequency pre-isolator has been proposed as a support for the FF magnets. It can be integrated at the interface between the machine tunnel and the detector cavern.
- This pre-isolator will constitute the first "layer" of the stabilization chain.
- This concept has already been used in other industrial and research facilities.

• Experimental tests will be performed to understand how the system behaves in "real life" conditions (no simulation can accurately take into account all these effects). A simple and relatively inexpensive experimental set-up has been conceived and built. Measurements are under way.

• Further studies are necessary to integrate the pre-isolator with the civilengineering and the other MDI systems (pre-alignement, FF magnets, forward detector region, beam-pipe, etc...)